

# A new high-order particle method for solving high Reynolds number incompressible flows

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### Abstract

In this study, a new high-order particle method is proposed to solve the incompressible Navier–Stokes equations. The proposed method combines the advantages of particle and mesh methods to approximate the total and the spatial derivative terms under the Lagrangian and the Eulerian frameworks. Our aim is to avoid convective instability and increase solution accuracy at the same time. Data transfer from Lagrangian particles to Eulerian grids is realized by moving least squares interpolation. In contrast to the previously proposed method, there is no need to interpolate diffusion terms from Eulerian grids to Lagrangian particles. Therefore, the accuracy of the present solution will not be deteriorated by interpolation error. Additionally, no extra work is required to manage particles for searching procedure. Because no convection term needs to be discretized by upwinding schemes, false diffusion and dispersion errors will not be introduced, thereby increasing the solution accuracy. To verify the proposed particle method, several benchmark problems are solved to show that the present simulation is more stable, accurate, and efficient. The proposed particle method renders fourth- and second-order accurate solutions in space for velocity and pressure, respectively.

**Keywords** High-order particle method · Incompressible Navier–Stokes equations · False diffusion error · Numerical dispersion error

# 1 Introduction

Conventionally, there are two major classes of numerical methods for solving the incompressible Navier–Stokes equations: mesh and meshless (particle) methods. Finite difference method (FDM), finite volume method (FVM) and finite element method (FEM) are categorized as the mesh methods. They are developed based on the Taylor series expansion, conservation in a control volume, and variational principles, respectively. On the other hand, the moving particle

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semi-implicit (MPS) and smoothed particle hydrodynamics (SPH) methods are the typical particle methods. MPS adopts the idea of particle interactions to model spatial differential operators, while Sph makes use of the concept of integral representation of a function and exploits the derivative of the kernel function to approximate spatial differential operators.

For the two major particle methods, MPS method was developed in 1996 [1] to deal with incompressible free surface flow problem and SPH method was firstly developed to solve astrophysics problems [2,3] in 1977. MPS solves the incompressible flows in a strong form and approximates differential operators within particle interaction framework. MPS adopts projection method to solve the Navier-Stokes equations and uses the concept of particle number density (PND) to model incompressibility [1]. However, the original particle methods suffer from the oscillating pressure fields. Several approaches have been proposed to tackle this difficulty [4-7]. Additionally, MPS method was also improved to tackle physically more complex flow problems such as the multi-resolution problem [8], multi-phase flow problem [9,10], and progressive water wave problem [11]. On the other hand, SPH solves the weakly compressible flow in

a weak form originally and is adopted to solve the incompressible flows [12-15]. SPH has also been applied to solve multi-resolution problem [16,17] or flexible fibers-viscous fluids interaction [18]. Both MPS and SPH methods suffer from the drawback of low accuracy and oscillatory pressure. Recently, accuracy has been improved by taking least squares or local polynomial reconstruction concepts into account [8,19,20], while pressure oscillations were eliminated by modifying the source terms in the pressure Poisson equation (PPE) [21-23]. Since PPE is an elliptic equation, Hwang proposed moving particle method with an embedded pressure mesh (MPPM) [4] to reduce pressure oscillations. Also, MPPM method was extended to solve multi-phase flow [10] or flows in complex flow domain [24]. In some of the recently proposed MPS and SPH methods, [20,25] and [26] show first-, second- and third-order accurate solutions for velocity, respectively. However, for the most important issue, they did not improve the accuracy order for pressure, which is most likely first-order accurate. This motivates us to develop a new particle method aiming primarily at increasing the spatial accuracy order for pressure to second. Moreover, the spatial accuracy order for velocity is improved to fourth as well.

In order to resolve low accuracy and inconsistency issues encountered in the conventional SPH and MPS methods, several refined methods have been proposed such as the updated Lagrangian particle hydrodynamics (ULPH) method [27], material point method (MPM) [28,29], corrective smoothed particle method (CSPM) [30], kernel gradient correction (KGC) method [31,32], finite particle method (FPM) [33], and decoupled finite particle method (DFPM) [34]. Most of these particle methods, except the MPM, are aimed at providing alternative ways, which are consistent and have higher accuracy compared with the conventional SPH and MPS methods, to approximate the gradient and the divergence operators. Also, it is noted that these numerical methods are actually solving weakly compressible Navier-Stokes equations such that the equation of state or a pressure transient equation shall be invoked for the calculation of pressure. In ULPH method, new formulations were proposed, while the others were constructed through the correction on the conventional SPH method. The accuracy order of velocity and pressure of these methods does not exceed second and first orders, respectively. In addition, ghost particles are adopted in these methods to model Dirichlet and Neumann boundary conditions for velocity. However, the proposed IMLE method exploits the advantage of the Eulerian method such that the spatial derivative terms can be approximated to yield an even higher accuracy order, e.g., sixth- and fifth-order accuracy for the pressure gradient and the velocity Laplacian terms, respectively. From the numerical tests, IMLE method shows fourth- and second-order accuracy for velocity and pressure fields, respectively. Also, with the aid of the interpolation procedure, there is no longer any need to take care of particle

penetrating across the wall. While the ULPH, CSPM, KGC, FPM, and DFPM are classified to be the fully Lagrangian methods, the MPM is also a hybrid method which combines the advantages of both Lagrangian and Eulerian methods such as the newly proposed IMLE method. Some key differences are that the primitive variables  $\mathbf{u}$  and p are transferred in two ways, namely they are interpolated from Lagrangian particles to Eulerian grids and in reverse at the integer time step levels n and n + 1. On the other hand, the IMLE method only interpolates the intermediate velocities from Lagrangian particles to Eulerian grids. It is noted that the source term of the PPE is calculated from the interpolated velocities. As a result, it is better to use the pressure gradient term to modify the interpolated velocities on Eulerian grids only but not those on Lagrangian particles. Therefore, at the beginning of the next time step, the velocity Laplacian term on Eulerian grids can be approximated using a higher accurate numerical scheme without adopting any interpolation procedure which will deteriorate the accuracy order. It is also noted that in the proposed IMLE method, unlike some other particle methods [29,31,33], no free parameters will be involved.

In the conventional finite difference and finite volume methods, the velocity terms are solved from the momentum equations which involve convection terms. When discretizing convection terms, upwinding scheme can effectively get rid of convective instability problem [35]. However, upwinding schemes introduce errors which can deteriorate the accuracy of the results. Firstly, discretizing convection terms shall introduce numerical dispersion error. Several dispersion-relation-preserving (DRP) schemes proposed in [35–38] can be applied to reduce such an error. Secondly, in most of the multi-dimensional flow situation, streamlines in a flow field are not aligned with the grid lines. Therefore, it is difficult to eliminate false diffusion error. Even though some numerical methods [39–42] can partly resolve this problem, within the framework of Eulerian approach it is still an open issue in the computational fluid dynamics (CFD) community.

The goal of this paper is aimed at developing an improved mixed Lagrangian-Eulerian (IMLE) method based on the previously proposed MLE method [43] to improve computational accuracy, enhance numerical stability and computational efficiency by combining advantages of both particle (meshless) and mesh methods. Diffusion terms present in the momentum equations are discretized on Eulerian grids to get high-order accuracy. It is emphasized that it is easier to get a better accuracy on Cartesian grid rather than on Lagrangian particles. Total derivative terms are approximated by simply advecting Lagrangian particles so that the procedure of discretizing convection terms can be avoided, thereby completely eliminating the convective instability problem. With the aid of the moving least squares (MLS) interpolation method [44], velocity components can be interpolated back to the Eulerian grid to calculate the source term of the PPE. It is noted that MLS method was recently adopted in high-order methods to solve fluid flow problems [45–48]. While the PPE is solved on the Eulerian grid which is suitable for elliptic property of PPE, pressure field shows no oscillations, thus eliminating a major source of numerical error generated in conventional particle methods. In this newly proposed IMLE method, as the most outstanding feature of Lagrangian particle methods, there are no numerical dispersion and dissipative errors since no upwinding schemes for the discretization of convection terms have been adopted. Additionally, accuracy order in calculating diffusion terms is as high as that of the numerical schemes adopted on Eulerian grids and is not restricted by the interpolation scheme presented in the original MLE method.

The rest of this paper is organized as follows. Section 2 describes the governing equations and boundary conditions prescribed in this paper. In Section 3, the previously proposed mixed Lagrangian–Eulerian (MLE) method [43] is reviewed. Then the spatial discretization schemes and the proposed improved mixed Lagrangian–Eulerian (IMLE) method are detailed. Section 4 presents numerical results obtained from some classical benchmark problems including the Taylor–Green vortex, the backward-facing step, and the lid-driven cavity flow problems. Conclusions are drawn in Sect. 5.

# 2 Governing equations

In this study, we consider the equations for two-dimensional incompressible fluid flow. Governing equations, namely continuity and Navier–Stokes equations, are solved in the physical domain  $\Omega$  whose boundary is  $\partial \Omega$ .

$$\nabla \cdot \mathbf{u} = 0 \tag{1}$$

$$\frac{d\mathbf{u}}{dt} \equiv \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \mathbf{u}$$
(2)

In the above equations, **u** is the velocity vector, *p* the pressure,  $\rho$  the density, and  $\nu$  the kinematic viscosity. Reynolds number is defined as  $Re = \frac{\rho UL}{\mu}$ , where *U* is the characteristic velocity, *L* the characteristic length, and  $\mu$  the dynamic viscosity. The solutions of Eqs. (1) and (2) are sought subjected to the initial condition for velocity specified in  $\Omega$  and the boundary conditions for velocity and pressure prescribed on  $\partial \Omega$ .

$$\mathbf{u}\left(\mathbf{x},t=0\right) = \mathbf{u}_{0}\left(\mathbf{x}\right) \quad \text{in} \quad \Omega \tag{3}$$

$$\alpha \phi + \beta \mathbf{n} \cdot \nabla \phi = \gamma \quad \text{on} \quad \partial \Omega \tag{4}$$

In Eq. (4),  $\phi$  can be velocity **u** or pressure *p* and **n** denotes the unit outward normal vector of the boundary  $\partial \Omega$ . For Dirichlet

and Neumann boundary conditions,  $(\alpha, \beta)$  equals (1, 0) and (0, 1), respectively.

## **3 Numerical methods**

# 3.1 Review of the previously proposed mixed Lagrangian-Eulerian method

In the previously proposed Mixed Lagrangian-Eulerian (MLE) method [43], the pressure oscillation problem usually encountered in the conventional particle methods, such as MPS [1,5,7,21,22,49-52] and SPH [2,3,12,53-55], has been resolved by solving the elliptic PPE on Eulerian grids. Moreover, the accuracy order of discretizing the diffusion terms on the Lagrangian particles has been improved from the zeroth [56] and first [57] orders to the second order. The key idea is that the diffusion terms were firstly calculated on Eulerian grids by using the high-order scheme given in [58] and then they were interpolated to the Lagrangian particles by second-order accurate bilinear interpolation. Thus, the accuracy order of approximating diffusion terms at the locations of moving particles is restricted by the adopted interpolation scheme After the step of advecting particles, intermediate velocities at the locations of moving particles are interpolated back to the Eulerian grids by using third-order MLS in order to calculate the source term shown in PPE. In the final step, the values of velocity components on the moving particles are updated by the pressure gradient term and these particles continue their journey again according to the final velocities.

In the original MLE method, the continuity equation is solved on Eulerian grids aiming to retain the elliptic property of the PPE. It is implied that the moving particles are regarded as observation points rather than mass points as usually used in the MPS and SPH methods. Because the particles are considered as mass points, they cannot penetrate physical boundaries and will generate less accurate or totally unacceptable solutions when particles become clustered. On the other hand, observation particles can be deleted from or added into a cell according to the particle distribution situation. For the case of highly clustered particles, the particle nearest the cell center will be kept and the others be deleted. On the contrary, a particle will be added into a cell without any particle. Also, for the balance of the computational efficiency and accuracy, a tuned parameter was proposed to control the maximum number of particles in a cell. In Sect. 4.3, numerical results obtained from the case of different values of tuned parameters in MLE methods will be compared with the proposed IMLE method.

The numerical procedure of the MLE method is shown below.

#### Algorithm 1 Algorithm of the MLE method

#### 1: Read input file

- 2: Calculate computational parameters
- 3: for time < MaxTime do
- 4: Calculate diffusion terms on Eulerian grids
- 5: Interpolate diffusion terms from Eulerian grids to Lagrangian particles
- 6: Calculate intermediate velocities on Lagrangian particles
- 7: Calculate intermediate locations of Lagrangian particles according to the intermediate velocities
- 8: Update link-list which manages particle distribution
- 9: Interpolate intermediate velocities from Lagrangian particles to Eulerian grids
- 10: Solve PPE
- 11: Modify intermediate velocities on Eulerian grids and Lagrangian particles
- 12: Modify particle locations according to updated velocities
- 13: Update link-list which manages particle distribution
- 14: Delete too-clustered particles/Add one particle into the empty cell/Delete particles from a cell which contains too many particles according to the value of *npcmax*
- 15: end for
- 16: Output results for post-processing

# 3.2 Cell-centered combined compact difference scheme in collocated grids

In this study, a cell-centered (CC) collocated grid system schematically shown in Fig. 1a is adopted. Red circles denote cell centers storing variables **u** and *p*, green diamonds denote boundary face centers storing boundary values of **u** and *p*, and blue diamonds denote corner points storing only boundary values of **u**. The reason for keeping corner points will be addressed in Sect. 3.4.2. The numbers of the grid points and the mesh cells are defined as np and nc = np - 1, respectively. The grid size is defined as  $h = \frac{L_x}{nc}$ , where  $L_x$  is the domain length along *x*-direction. For the sake of conve-



Fig. 1 Two-dimensional grid systems

nience, a uniform Cartesian mesh is used in this study. For spatial derivative terms in interior grid points, the sixth-order accurate combined compact difference (CCD) [58] schemes given below are applied to calculate the first and second derivative terms, respectively.

$$\frac{7}{16} \left( \phi'_{i+1} + \phi'_{i-1} \right) + \phi'_{i} - \frac{h}{16} \left( \phi''_{i+1} - \phi''_{i-1} \right) 
= \frac{15}{16h} \left( \phi_{i+1} - \phi_{i-1} \right)$$

$$\frac{9}{8h} \left( \phi'_{i+1} - \phi'_{i-1} \right) - \frac{1}{8} \left( \phi''_{i+1} + \phi''_{i-1} \right) + \phi''_{i} 
= \frac{3}{h^{2}} \left( \phi_{i+1} - 2\phi_{i} + \phi_{i-1} \right)$$
(5)
(6)

It is noted that the original CCD schemes were derived in the context of vertex-centered (VC) collocated grid system which is shown in Fig. 1b. In the cell-centered grid system, the schemes should be derived for calculating the first and second derivatives with Dirichlet and Neumann boundary conditions for boundary cells. Here the scheme for approximating the first derivative term subjected to Dirichlet boundary condition is taken as an example. The first derivative term at the first cell center, i = 1, is assumed to take the form of

$$\phi_1' + a_1\phi_2' + hb_1\phi_2'' = \frac{1}{h}\left(c_1\phi_L + c_2\phi_1 + c_3\phi_2\right) \tag{7}$$

In the above equation, *h* is the grid size and  $\phi_L$  is the value at the left Dirichlet boundary. After performing the modified equation analysis and eliminating the first few leading error terms, the five coefficients shown in Eq. (7) can be determined uniquely as:  $a_1 = \frac{8}{9}$ ,  $b_1 = -\frac{1}{6}$ ,  $c_1 = -\frac{16}{27}$ ,  $c_2 = -1$ , and  $c_3 = \frac{43}{27}$ . The leading error term of Eq. (7) is  $\frac{1}{240}h^4\phi^{(5)}$ . By repeating the above procedure, the discretization schemes for calculating the first and second derivative terms with Dirichlet, Neumann, and periodic boundary conditions can



be derived and the corresponding matrix equations, taking nc = 4 as an example, are shown below.

(1) Cell-centered combined compact difference scheme for the case with Dirichlet boundary condition (CC-CCD-D) is tridiagonal matrices. Note that the size of the sub-matrix is two because the number of unknowns at each mesh cell is two and the number of sub-matrices is *nc*. Advantages of adopting CC grids rather than VC grids are listed below.



- (2) Cell-centered combined compact difference scheme for the case with Neumann boundary condition (CC-CCD-N) is
- There are no geometrical singular points. As shown in Fig. 1b, the four corner points are the geometrically singular points upon which numerical implementation of

$$\begin{bmatrix} 1 & 0 & \left| \frac{31}{27} - \frac{2h}{9} \right| \\ 0 & 1 & \left| -\frac{8}{9h} - \frac{3}{1} \right| \\ \hline \frac{7}{16} & \frac{h}{16} \right| & 1 & 0 & \left| \frac{7}{16} - \frac{h}{16} \right| \\ -\frac{9}{8h} - \frac{1}{8} \right| & 0 & 1 & \left| \frac{9}{9h} - \frac{1}{8} \right| \\ \hline \frac{7}{16} & \frac{h}{16} \left| 1 & 0 & \left| \frac{7}{16} - \frac{h}{16} \right| \\ \hline \frac{9}{9h} - \frac{1}{8} \right| & 0 & 1 & \left| \frac{9}{9h} - \frac{1}{8} \right| \\ \hline \frac{9}{9h} - \frac{1}{8} \right| & 0 & 1 & \left| \frac{9}{9h} - \frac{1}{8} \right| \\ \hline \frac{1}{9h} - \frac{15}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{3}{h^2} - \frac{6}{h^2} & \frac{3}{h^2} \right| \\ \hline \frac{1}{9h} - \frac{15}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{3}{h^2} - \frac{6}{h^2} & \frac{3}{h^2} \right| \\ \hline \frac{1}{9h} - \frac{15}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{3}{h^2} - \frac{6}{h^2} & \frac{3}{h^2} \right| \\ \hline \frac{1}{9h} - \frac{15}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{3}{h^2} - \frac{6}{h^2} & \frac{3}{h^2} \right| \\ \hline \frac{1}{9h} - \frac{15}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{1}{9h} - \frac{1}{8h} & \frac{1}{10} & 0 & 1 \\ \hline \frac{1}{9h} - \frac{1}{8h} & \frac{1}{10} & 0 & 1 \\ \hline \frac{1}{9h} - \frac{1}{8h} & \frac{1}{10} & 0 & 1 \\ \hline \frac{1}{9h} - \frac{1}{8h} & \frac{1}{10} & 0 & 1 \\ \hline \frac{1}{9h} - \frac{1}{8h} & \frac{1}{10} & 0 & 1 \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{15}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & 0 & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} & \frac{1}{16h} \\ \hline \frac{1}{9h} - \frac{1}{16h} \\ \hline$$

(3) Cell-centered combined compact difference scheme for the case with periodic boundary condition (CC-CCD-P) is Neumann boundary conditions is not easy while solving PPE. The Neumann boundary conditions can be discretized along the diagonal grid points as stated in [59]



As shown in Eqs. (8) and (9), the coefficient matrices for the calculation of the first and second derivative terms are block-

for uniform mesh or using the modified Collatz difference scheme given in [60]. However, one can avoid discretiz-

ing the Neumann boundary conditions at the corner points in CC grids.

- 2. The matrix size, LDA, is constant for the cases with Dirichlet, Neumann, and periodic boundary conditions, while LDA is different under different boundary conditions if VC grid is adopted. For example, LDA =  $2 \times nc$  for calculating the first and second derivative values with the given function values and LDA =  $3 \times nc$  for solving partial differential equation (PDE) if CC-CCD scheme is adopted. On the other hand, LDA equals  $2 \times np$  and  $2 \times nc$  for VC-CCD scheme to deal with problems with Dirichlet and Neumann/periodic boundary conditions. Therefore, the complexity in programming can be reduced in using the CC-CCD scheme.
- 3. In the original VC-CCD scheme, three function values at grid points i = 1-3 are taken into account in calculating the derivative terms at grid point i = 1. The derived block-tridiagonal matrix property is broken when solving a PDE as shown below.

$$\begin{bmatrix} \mathbf{B}_{1} \ \mathbf{C}_{1} & & \\ \mathbf{A}_{2} \ \mathbf{B}_{2} \ \mathbf{C}_{2} & & \\ \mathbf{A}_{3} \ \mathbf{B}_{3} & \mathbf{C}_{3} & & \\ & \ddots & \ddots & \ddots & \\ & \mathbf{A}_{nc-1} \ \mathbf{B}_{nc-1} \ \mathbf{C}_{nc-1} \\ & \mathbf{A}_{nc} & \mathbf{B}_{nc} \end{bmatrix} \begin{bmatrix} \mathbf{x}_{1} \\ \mathbf{x}_{2} \\ \mathbf{x}_{3} \\ \vdots \\ \mathbf{x}_{nc-1} \\ \mathbf{x}_{nc} \end{bmatrix}$$

$$= \begin{cases} \mathbf{b}_{1} \\ \mathbf{b}_{2} \\ \mathbf{b}_{3} \\ \vdots \\ \mathbf{b}_{nc-1} \\ \mathbf{b}_{nc} \end{bmatrix}$$
(12)

In the above matrix equation,  $\mathbf{A}_i$ ,  $\mathbf{B}_i$ , and  $\mathbf{C}_i$  are the submatrices shown in the coefficient matrix and their dimension is 2 for calculating the first and second derivative terms and 3 for solving a PDE.  $\mathbf{x}_i$  and  $\mathbf{b}_i$  are the solution vector and source vector corresponding to the cell index *i*. The coefficient matrix is firstly decomposed into  $\mathbf{L}$  and  $\mathbf{U}$  matrices as shown below.

| $\begin{array}{c ccccccccccccccccccccccccccccccccccc$  | $egin{pmatrix} \phi_1 \ \phi_1' \ \phi_1'' \ \phi_1'' \ \end{pmatrix}$            |       | $\begin{cases} src_1 \\ \gamma \\ 0 \end{cases}$    |   |      |
|--|---|-------|---|---|------|
| $ \begin{array}{c c c c c c c c c c c c c c c c c c c $  | $\overline{\begin{array}{c}\phi_2\\\phi_2'\\\phi_2''\\\phi_2''\end{array}}$       |       | $ \begin{array}{c} src_2 \\ 0 \\ 0 \\ \end{array} $ |   |      |
| $ \begin{vmatrix} 0 & 0 & 0 \\ \frac{15}{16h} & \frac{7}{16} & \frac{h}{16} \\ -\frac{3}{h^2} & -\frac{9}{8h} & -\frac{1}{8} \end{vmatrix} \begin{vmatrix} a & b & c \\ 0 & 1 & 0 \\ \frac{6}{h^2} & 0 & 1 \end{vmatrix} - \frac{15}{16h} \frac{7}{16} - \frac{h}{16} \\ -\frac{3}{h^2} & \frac{9}{8h} - \frac{1}{8} \end{vmatrix} $ | $ \begin{array}{c} \phi_3 \\ \phi_3' \\ \phi_3'' \\ \phi_3'' \end{array} $        | · = · | $\begin{bmatrix} src_3 \\ 0 \\ 0 \end{bmatrix}$     | } | (11) |
| $\begin{vmatrix} 0 & 0 & 0 \\ \frac{15}{16h} & \frac{7}{16} & \frac{h}{16} \\ -\frac{3}{h^2} & -\frac{9}{8h} & -\frac{1}{8} \end{vmatrix} \begin{vmatrix} a & b & c \\ 0 & 1 & 0 \\ \frac{b}{h^2} & 0 & 1 \end{vmatrix} - \frac{15}{16h} \frac{7}{16} - \frac{h}{16} \\ -\frac{3}{h^2} \frac{9}{8h} - \frac{1}{8} \end{vmatrix}$     | $\phi_4 \ \phi_4' \ \phi_4'' \ \phi_4''$  |       | <i>src</i> <sub>4</sub><br>0<br>0                   |   |      |
| $\begin{bmatrix} 0 & 0 & 0 & a & b & c \\ 0 & 0 & 0 & \alpha & \beta & 0 \\ -\frac{1}{h} & \frac{32}{h} & 16 & 4h & -\frac{31}{h} & 14 & -2h \end{bmatrix}$  | $\left( egin{array}{c} \phi_5 \ \phi_5' \ \phi_5'' \ \phi_5'' \end{array}  ight)$ |       | $\begin{array}{c} src_5\\ \gamma\\ 0\end{array}$    |   |      |

In the above matrix equation, coefficients *a*, *b*, and *c* are used in the governing equation, which is  $a\phi + b\phi' + c\phi'' = src$ , and  $\alpha$ ,  $\beta$ , and  $\gamma$  are shown in Eq. (4). From the computational point of view, the currently adopted CC-CCD scheme retains the block-tridiagonal matrix property while calculating derivative values and solving a PDE. Therefore, the block-tridiagonal LU decomposition method can be adopted in this paper for solving Eqs. (8) and (9).

LU decomposition of the block-tridiagonal matrix is illustrated below. Equations (8) and (9) can be written in the following form.

$$\begin{bmatrix} B_{1} & C_{1} \\ A_{2} & B_{2} & C_{2} \\ A_{3} & B_{3} & C_{3} \\ & \ddots & \ddots & \ddots \\ & A_{nc-1} & B_{nc-1} & C_{nc-1} \\ & & A_{nc} & B_{nc} \end{bmatrix}$$



| Algorithm 2 Block-tridiagonal LU decomposition                   |
|--|
| 1: $\mathbf{U}_1 = \mathbf{B}_1$                                 |
| 2: <b>for</b> $i = 2$ to <i>nc</i> <b>do</b>                     |
| 3: Solve $\mathbf{L}_i \mathbf{U}_{i-1} = \mathbf{A}_i$          |
| 4: $\mathbf{U}_i = \mathbf{B}_i - \mathbf{L}_i \mathbf{C}_{i-1}$ |
| 5: end for   |
|  |

Through a forward substitution, vector  $\mathbf{y} = \mathbf{U}\mathbf{x}$  is obtained.

$$\begin{bmatrix} \mathbf{I}_{1} & & & \\ \mathbf{L}_{2} & \mathbf{I}_{2} & & \\ & \mathbf{L}_{3} & \mathbf{I}_{3} & & \\ & \ddots & \ddots & & \\ & & \mathbf{L}_{nc-1} & \mathbf{I}_{nc-1} \\ & & & & \mathbf{L}_{nc} & \mathbf{I}_{nc} \end{bmatrix} \begin{bmatrix} \mathbf{y}_{1} \\ \mathbf{y}_{2} \\ \mathbf{y}_{3} \\ \vdots \\ \mathbf{y}_{nc-1} \\ \mathbf{y}_{nc} \end{bmatrix} = \begin{bmatrix} \mathbf{b}_{1} \\ \mathbf{b}_{2} \\ \mathbf{b}_{3} \\ \vdots \\ \mathbf{b}_{nc-1} \\ \mathbf{b}_{nc} \end{bmatrix}$$
(14)

| Algorithm 3 Block-tridiagonal forward substitution               |   |
|--|---|
| 1: $y_1 = b_1$   |   |
| 2: <b>for</b> $i = 2$ to <i>nc</i> <b>do</b>                     |   |
| 3: $\mathbf{y}_i = \mathbf{b}_i - \mathbf{L}_i \mathbf{y}_{i-1}$ |   |
| 4: end for   |   |
|  | - |

Finally, the solution vector **x** can be obtained after a backward substitution step.

$$\begin{bmatrix} \mathbf{U}_{1} \ \mathbf{C}_{1} \\ \mathbf{U}_{2} \ \mathbf{C}_{2} \\ \mathbf{U}_{3} \ \mathbf{C}_{3} \\ \ddots \\ \mathbf{U}_{nc-1} \ \mathbf{C}_{nc-1} \\ \mathbf{U}_{nc} \end{bmatrix} \begin{bmatrix} \mathbf{x}_{1} \\ \mathbf{x}_{2} \\ \mathbf{x}_{3} \\ \vdots \\ \mathbf{x}_{nc-1} \\ \mathbf{x}_{nc} \end{bmatrix} = \begin{bmatrix} \mathbf{y}_{1} \\ \mathbf{y}_{2} \\ \mathbf{y}_{3} \\ \vdots \\ \mathbf{y}_{nc-1} \\ \mathbf{y}_{nc} \end{bmatrix}$$
(15)

| Algorithm 4 Block-tridiagonal backward substitution                                 |
|---|
| 1: Solve $\mathbf{U}_{nc}\mathbf{x}_{nc} = \mathbf{y}_{nc}$                         |
| 2: for $i = nc - 1$ to 1 do   |
| 3: Solve $\mathbf{U}_i \mathbf{x}_i = \mathbf{y}_i - \mathbf{C}_i \mathbf{x}_{i+1}$ |
| 4: end for  |

In the execution of computer code, the coefficient matrices can be constructed and decomposed only once in the preprocessing step. During the time-stepping procedure, only forward and backward substitutions are needed to be performed to get the solution vector  $\mathbf{x}$ . When solving the periodic boundary value problem, the same procedures can be applied only with a slight modification of the matrices  $\mathbf{L}$  and  $\mathbf{U}$ .

# 3.3 The newly proposed improved mixed Lagrangian–Eulerian method

To solve Eqs. (1) and (2), the projection method proposed by Chorin [61] is adopted. First of all, Eq. (2) is temporally discretized by the first-order Euler explicit method without considering the pressure gradient term under the Lagrangian framework to get the intermediate velocities.

$$\mathbf{u}^* = \mathbf{u}^n + \Delta t \, \nu \, \nabla^2 \mathbf{u}^n \tag{16}$$

In the above equation, the superscripts n and \* represent the solutions obtained at the time level n and the intermediate level between levels n and n + 1. Since the momentum equation is solved under the Lagrangian framework, fluid particles should be advected with the intermediate velocities to the intermediate locations.

$$\mathbf{r}^* = \mathbf{r}^n + \Delta t \mathbf{u}^* \tag{17}$$

It is to be noted that  $\mathbf{r}^n$  is the location of the center of a cell which is uniformly distributed because Cartesian grid is considered in this study. It is obvious that the intermediate location  $\mathbf{r}^*$  is not uniformly distributed any longer after the advection step. Therefore, a third-order accurate moving least squares (MLS) interpolation method is adopted to interpolate the intermediate velocities from  $\mathbf{r}^*$  back to  $\mathbf{r}^n$ . In this study,  $3^2 = 9$  particles are taken into account for interpolation purpose as shown in Fig. 2. Red circles represent cell center  $(\mathbf{r}^n)$ , and blue triangles represent intermediate locations  $(\mathbf{r}^*)$ . It is noted that the red solid circle is the reference location to interpolate velocities from the surrounding nine blue solid triangles. Red and blue hollow symbols are not taken into account in the interpolation procedure. The green circle is the influence domain for the red solid circle, and the radius is twice the diagonal length of the mesh cell such that  $r_e = 2\sqrt{2}h$ . The influence radius  $r_e$  is used in calculating the weighting function  $\omega_i$  ( $r_i, r_e$ ) which is used in the

$$f(x, y) = a_0 + a_1 x + a_2 y + a_3 x^2 + a_4 x y + a_5 y^2 + O(x^3, y^3)$$
(19)

The coefficients  $a_0-a_5$  can be determined by using the MLS method to minimize the error and solving the following linear system.

$$\begin{bmatrix} \sum \omega_i^2 & \sum \omega_i^2 x_i & \sum \omega_i^2 y_i & \sum \omega_i^2 y_i & \sum \omega_i^2 x_i^2 & \sum \omega_i^2 x_i y_i & \sum \omega_i^2 x_i^2 y_i \\ \sum \omega_i^2 x_i & \sum \omega_i^2 x_i y_i & \sum \omega_i^2 x_i^2 & \sum \omega_i^2 x_i^2 y_i & \sum \omega_i^2 x_i^2 y_i & \sum \omega_i^2 x_i^2 y_i \\ \sum \omega_i^2 x_i^2 & \sum \omega_i^2 x_i^2 & \sum \omega_i^2 x_i^2 & \sum \omega_i^2 x_i^2 y_i & \sum \omega_i^2 x_i^2 y_i & \sum \omega_i^2 x_i^2 y_i \\ \sum \omega_i^2 x_i y_i & \sum \omega_i^2 x_i^2 & \sum \omega_i^2 x_i^2 y_i \\ \sum \omega_i^2 x_i y_i & \sum \omega_i^2 x_i^2 y_i^2 & \sum \omega_i^2 x_$$

$$\omega_{i}(r_{i}, r_{e}) = \begin{cases} \frac{a}{r_{e}} + \frac{b}{r_{e}} \left(\frac{r_{i}}{r_{e}}\right) + \frac{c}{r_{e}} \left(\frac{r_{i}}{r_{e}}\right)^{2} + \frac{d}{r_{e}} \left(\frac{r_{i}}{r_{e}}\right)^{3} + \frac{e}{r_{e}} \left(\frac{r_{i}}{r_{e}}\right)^{4} & 0 \le r_{i} \le r_{e} \\ 0 & r_{e} > r_{i} \end{cases}$$

$$(18)$$

tance between the interpolated point and the reference point (red solid circle). In this study, the weighting function given

below takes the form from [62].

where

$$a = \frac{480\sqrt{2} - 705}{512\sqrt{2} - 745} \quad b = 0 \quad c = \frac{-960\sqrt{2} + 1515}{512\sqrt{2} - 745}$$
$$d = \frac{-210}{512\sqrt{2} - 745} \quad e = \frac{480\sqrt{2} - 600}{512\sqrt{2} - 745}$$

For performing the MLS interpolation method, a local polynomial given below is constructed.



**Fig. 2** Schematic of the interpolation process (circles denote Eulerian grids; triangles denote Lagrangian particles; solid symbols denote the points participating in interpolation process; hollow symbols denote the points not participating in interpolation process)

It is worth noting that the function value can be evaluated at the reference point by calculating f(0, 0) which is actually  $a_0$ . Therefore, the above linear system does not need to be solved entirely to get all the six coefficients. Instead, backward elimination procedure can be adopted to get  $a_0$  without computing the values of  $a_1-a_5$  and it is more computationally efficient.

Since the interpolated intermediate velocity field is not divergence free, a modified step is introduced to obtain the divergence free velocity field.

$$\mathbf{u}^{n+1} = \mathbf{u}_{\text{interpolated}}^* - \frac{\Delta t}{\rho} \nabla p^{n+1}$$
(21)

Taking divergence operation on the above equation and using Eq. (1), the pressure Poisson equation (PPE) can be obtained as following.

$$\nabla^2 p^{n+1} = \frac{\rho}{\Delta t} \nabla \cdot \mathbf{u}_{\text{interpolated}}^*$$
(22)

In this study, the above PPE is discretized by the second-order central difference scheme, leading to

$$p_{i+1,j} + p_{i-1,j} + p_{i,j+1} + p_{i,j-1} - 4p_{i,j}$$

$$= \frac{\rho h}{2\Delta t} \left( u_{i+1,j}^* - u_{i-1,j}^* + v_{i,j+1}^* - v_{i,j-1}^* \right)$$
(23)

The resulting linear system is solved by the conjugate gradient (CG) iterative solver. The iterative solution of p is said to be convergent if the root mean square (RMS) of the residuals is smaller than a convergence criteria, said RMS (res<sup>k</sup>) <  $\varepsilon = 10^{-15}$ , where res<sup>k</sup> is the residual of the *k*-th step iterative solutions and the function RMS is defined as

RMS 
$$(\phi) = \left(\frac{1}{nc_x} \frac{1}{nc_y} \sum_{i=1}^{nc_x} \sum_{j=1}^{nc_y} \phi_{i,j}^2\right)^{\frac{1}{2}}$$
 (24)

In the above equation,  $nc_x$  and  $nc_y$  are the number of mesh cells in x- and y-directions, respectively.

For the sake of completeness, the numerical procedure for the IMLE method is summarized below.

#### Algorithm 5 Algorithm of the IMLE method

1: Read input file

- 2: Calculate computational parameters
- 3: for time < MaxTime do
- 4: Calculate the diffusion terms on Eulerian grids, Eq. (8) or (9) or (10).
- 5: Calculate the intermediate velocities on Eulerian grids, Eq. (16).
- 6: Calculate the intermediate locations of the Lagrangian particles departing from the Eulerian grids according to the intermediate velocities, Eq. (17).
- 7: Interpolate the intermediate velocities from Lagrangian particles to Eulerian grids, Eqs. (20).
- 8: Solve the PPE, Eq. (23).
- 9: Update the intermediate velocities on Eulerian grids, Eq. (21).
- 10: end for
- 11: Output results for post-processing

Some key differences between the MLE and IMLE methods are summarized below.

- 1. After the interpolation step (step 9 in Algorithm 1), MLE particles keep moving from the intermediate locations  $\mathbf{r}^*$  to the next locations  $\mathbf{r}^{n+1}$  (step 12 in Algorithm 1) until they are deleted. This means that MLE particles are not always located at the Eulerian grid points. Thus another interpolation (step 5 in Algorithm 1) is required to interpolate diffusion terms from Eulerian grids to Lagrangian particles after those terms have been calculated on Eulerian grids by the VC-CCD scheme. Therefore, the accuracy order of the diffusion terms on Lagrangian particles is restricted by the accuracy order of the adopted interpolation scheme.
- 2. In every advection step (step 7 in Algorithm 1 and step 6 in Algorithm 5), the departing points of the MLE particles may not be located on the Eulerian grid points, while the IMLE particles are exactly departed from the Eulerian grid points (shown in Fig. 3). In other words, distance between a Lagrangian particle and its corresponding Eulerian grid point may be longer for MLE method, i.e.,  $|\mathbf{r}^n - \mathbf{r}^{n-2}|_{\text{MLE}} > |\mathbf{r}^* - \mathbf{r}^n|_{\text{IMLE}}$ . As a result, IMLE particles can be expected to be distributed more uniformly than MLE particles (a small/large deviation from the uniform Eulerian grids in IMLE/MLE methods). In this situation, particle distribution with smaller deviation can reduce MLS interpolation error [63] in IMLE method.
- 3. The source term of the PPE is calculated from the interpolated velocities. It is therefore better to update the

interpolated velocities on Eulerian grids only rather than to update the velocities on Lagrangian particles as well.

- 4. The interpolation procedure (step 7 in Algorithm 5) can be seen as a re-meshing step which re-meshes the slightly non-uniform Lagrangian particles to the uniform Eulerian grids.
- 5. MLE uses semi-staggered grid which requires one (two) additional interpolation procedures to interpolate the pressure gradient terms from cell faces to cell vertices in two (three)-dimensional grids [64]. IMLE adopts collocated grid, and there is no need for interpolating pressure gradient term; therefore, IMLE is more efficient.

With respect to the conventional finite difference method, which directly discretizes the convection terms, the previous process (steps 6–7 in Algorithm 5) can be seen as an alternative way to compute the addition of particle derivative and convection terms at grid points. As shown above, the particles move along the streamlines so that there is no false diffusion error generated from using any local one-dimensional upwinding scheme. Also, since there is no convection terms used in IMLE method, there is no dispersion error as well resulting from the discretization of the convection terms.

#### 3.4 Boundary treatments for intermediate velocity

# 3.4.1 Boundary treatment for Neumann boundary condition

For the cases of Dirichlet boundary condition, intermediate velocity components located on boundary face centers (green diamonds in Fig. 1a) are directly assigned as velocity components at next time step, i.e.,  $\mathbf{u}^* = \mathbf{u}^{n+1}$ . On the other hand, if Neumann boundary condition is considered, velocity components and tangential location of a boundary face particle are determined by the boundary cell particle. As shown in Fig. 4, tangential location of a boundary face particle is the projection of the location of a boundary cell particle, i.e.,  $y_{\text{face particle}}^* = y_{\text{cell particle}}^*$ . Similarly, velocity vector is duplicated from the boundary cell particle to its corresponding boundary face particle, i.e.,  $\mathbf{u}_{\text{face particle}}^* = \mathbf{u}_{\text{cell particle}}^*$ .

It is worth noting that there are no ghost/dummy particles adopted in IMLE method, while they are usually used in conventional SPH [25,65,66] and MPS [8,9,67] methods. Ghost particle or repulsive force model, which may deteriorate computational accuracy, was developed to prevent particles from penetrating rigid walls. However, in the proposed IMLE method, Lagrangian particles are reset to uniform Eulerian mesh points, thus ensuring that no particle can penetrate across the walls.



Fig. 3 Schematic of MLE (left) and IMLE (right) particles. Red circles denote cell centers, blue diamonds denote MLE particles at different time step, and blue triangle denotes intermediate location of IMLE particle. (Color figure online)



**Fig. 4** Schematic of determining location and velocity of boundary face particles (green solid diamond) for Neumann boundary condition in x-direction. (Color figure online)

#### 3.4.2 Boundary treatment for interpolation procedure

When using MLS interpolation, there are six unknowns in Eqs. (19) and (20). It is obvious that there should be at least six particles needed to be taken into account for MLS interpolation. Also, MLS interpolation is more accurate if there are more particles when solving Eq. (20). In this study, nine (=  $3^d$ , where d = 2 is the problem dimension) particles are considered in MLS interpolation for interior, boundary and

corner cells. Three interpolation cases are shown in Figs. 2 and 5a, b. In order to maintain nine-particle property, four corner particles are reserved for utilizing MLS interpolation of intermediate velocities. As mentioned previously, solid triangles and diamonds are used to interpolate intermediate velocities to solid circle while not for hollow symbols.

A most important point in using MLS interpolation is that there is no need to tackle the difficulty of the unity (normalization) condition which is the inevitable condition for SPH kernel function. In SPH, a physical quantity  $\phi$  on a particle can be calculated in continuous sense by means of

$$\phi_i = \int \phi_j W\left(\mathbf{x}_j - \mathbf{x}_i, h\right) \mathrm{d}V_j \tag{25}$$

The discrete form of the above equation is

$$\phi_i = \sum_{j=1}^{NP} \phi_j W \left( \mathbf{x}_j - \mathbf{x}_i, h \right) \Delta V_j$$
(26)

In the above two equations, the quantity  $\phi$  is interpolated from the surrounding particles  $j = 1 \sim NP$  to the particle *i*, where NP is the number of particles within the influence domain of particle *i*, *W* the kernel function, *h* the smoothing length, and  $\Delta V_j$  the volume of particle *j*. One of the several important properties for *W* is that *W* should satisfy the unity (normalization) condition. Continuous and discrete forms of the unity condition are shown below.

$$\int W\left(\mathbf{x}_{j}-\mathbf{x}_{i},h\right)\mathrm{d}V_{j}=1$$
(27)



(a) Dirichlet boundary condition





| Table 1 Spatial rates of | Mesh size | и            |        | ν            |        | р            |        |
|--------------------------|-----------|--------------|--------|--------------|--------|--------------|--------|
| Taylor–Green flow        |           | RMS of error | sroc   | RMS of error | sroc   | RMS of error | sroc   |
|                          | 1.00E-1   | 2.6800E-03   | _      | 2.6800E-03   | _      | 9.6307E-02   | _      |
|                          | 5.00E-2   | 1.7601E-04   | 3.9285 | 1.7601E-04   | 3.9285 | 2.2690E-02   | 2.0856 |
|                          | 2.50E-2   | 1.1140E-05   | 3.9818 | 1.1147E-05   | 3.9809 | 4.1758E-03   | 2.4419 |
|                          | 1.25E-2   | 7.0159E-07   | 3.9890 | 7.6250E-07   | 3.8698 | 1.2609E-03   | 1.7276 |
|                          |           |              |        |              |        |              |        |



Fig. 6 Numerical results of the Taylor–Green vortex flow considered at  $Re = 10^6$ 

$$\sum_{j=1}^{NP} W\left(\mathbf{x}_{j} - \mathbf{x}_{i}, h\right) \Delta V_{j} = 1$$
(28)

However, most of the kernel functions were developed in continuous sense, i.e., satisfying Eq. (27) only, they are not guaranteed to satisfy the discrete unity condition as stated

in [68–70]. In order to enforce the unsatisfied discrete unity condition, reproducing kernel particle method (RKPM) [71] or improved SPH methods [69,70] were developed.

On the other hand, MLS interpolation method is based on the concept of surface reconstruction. By using Eq. (19), a smooth function can be reconstructed accurately under the nine-particle framework. In a short summary, under the

Fig. 7 Evolution of decayed maximum velocity







ι





Fig. 10 Streamlines for different Reynolds numbers of the backward-facing step flow problem

weakly non-uniform particle distribution (stated in Sect. 3.3), the nine-particle compact framework (shown in this section) adopted in MLS interpolation method can improve computational accuracy.

#### 4 Verification and validation studies

#### 4.1 Taylor–Green vortex flow

In the first test case, the IMLE method is verified using the analytical solutions of the Taylor–Green vortex flow problem. The considered flow domain is a unit square consisting of a series of decaying vortices. The analytical solutions of the flow variables can be expressed as follows.

$$u_{\text{analytic}}(x, y, t) = -Ue^{bt}\cos\left(2\pi x\right)\sin\left(2\pi y\right)$$
(29)

$$v_{\text{analytic}}(x, y, t) = Ue^{bt} \sin(2\pi x) \cos(2\pi y)$$
(30)

$$p_{\text{analytic}}(x, y, t) = -\frac{U^2}{4}e^{2bt}\left[\cos\left(4\pi x\right) + \cos\left(4\pi y\right)\right] (31)$$

In the above equations, *b* is the decaying rate expressed as  $-8\pi^2/Re$ , where  $Re \ (= \rho UL/\mu)$  is the Reynolds number (= 100). The density  $\rho$ , reference velocity *U*, and characteristic length *L* are set to unity similar to those reported in the previous works [24,55,72].

First of all, in order to test the spatial rates of convergence (sroc) of velocity and pressure, periodic boundary conditions are applied in both x- and y-directions. Different grid resolutions ( $nc^2 = 10^2$ ,  $20^2$ ,  $40^2$ , and  $80^2$ ) are adopted. In order to evaluate the sroc, a relatively small time step size ( $\Delta t = 10^{-5}$ ) is considered and the simulations are executed for only one time step in order to eliminate the influence of time discretization error on the computed sroc [73]. Table 1 shows the RMS of the errors and the sroc at different grid resolutions. It is observed that velocity and pressure exhibit fourth- and second-order accuracy, respectively. Therefore, the current IMLE method is more accurate than our previous MLE method [43], as the latter reported that the sroc values of velocity and pressure are merely 2.0 and 1.4, respectively.

Secondly, in order to examine the numerical dissipation of the proposed IMLE method, the Reynolds number is



Fig. 11 Normalized velocity profiles of the backward-facing step flow problem

increased to  $10^6$ . The grid resolution is set to  $nc^2 = 400^2$  In this case, time step size  $\Delta t = 0.1 \Delta x$  is considered such that the initial Courant–Friedrichs–Lewy (CFL) number is about 0.1. Figure 6 shows the contours of velocities u, v, and pressure p. It is appealing to note that the predicted numerical solutions vary smooth even at high Re flow condition.

Finally, the proposed IMLE solutions are compared with the ISPH solutions [72]. Here, the time evolutions of maximum velocity (Eq. 32) are compared. The maximum relative error is defined in Eq. (33).

$$u_{\max}(t) = \max\left(\operatorname{abs}\left(\mathbf{u}\left(t\right)\right)\right) \tag{32}$$

$$L_{\infty} = \max_{t=0}^{5} \frac{|u_{\max}(t) - e^{bt}|}{e^{bt}}$$
(33)

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For this comparative study, simulations are performed using grid resolutions  $nc^2 = 10^2$ ,  $20^2$ ,  $40^2$ , and  $80^2$  and the CFL number is again set as 0.1. The Reynolds number is 100, and the simulation time is 5 s. Figure 7 shows that  $u_{\text{max}}(t)$  comes closer to the analytical solution if a finer mesh is adopted. Figure 8 shows that the proposed IMLE method outperforms the ISPH method [72].

# 4.2 Backward-facing step flow

In this subsection, a backward-facing step flow problem is considered to show the ability of applying IMLE to simulate inflow-outflow problem. The physical domain, the initial, and the boundary conditions are shown in Fig. 9. In this

| Table 2   Comparisons of  | Program name |                      | MLE              |           | IMLE       |         |           |             |
|---|--------------|----------------------|------------------|-----------|------------|---------|-----------|-------------|
| MLE and IMLE of the   |              |                      | Comput. time (s) | Prop. (%) | Comput. ti | ime (s) | Prop. (%) | Improvement |
| backward-facing step flow problem at $Re = 100$                       | Main         |                      | 1.2561E+05       | 100.00    | 3.1721E+   | 04      | 100.00    | 3.96        |
| I   | u*           |                      | 1.1621E+03       | 0.93      | 6.1763E+   | 02      | 1.95      | 1.88        |
|   | MLS in       | trpl.                | 2.7307E+04       | 21.74     | 2.0338E+   | 03      | 6.41      | 13.43       |
|   | $p^{n+1}$    |                      | 9.4545E+04       | 75.27     | 2.8761E+   | 04      | 90.67     | 3.29        |
| Table 3         Velocity u at different                               |              | r / H                |                  |           |            |         |           |             |
| cross sections of the backward-facing step flow problem at $Re = 100$ | y.           | $\frac{x/H_s}{0.00}$ | 2.55             | 3.06      | 3.57       | 4.18    | 4.80      | 5.41        |
|   | 0.0          | 0.0000               | 0.0000           | 0.0000    | 0.0000     | 0.0000  | 0.0000    | 0.0000      |
|   | 0.5          | 0.0000               | - 0.0492         | 0.0006    | 0.0517     | 0.0996  | 0.1277    | 0.1407      |
|   | 1.0          | 0.0000               | - 0.0500         | 0.0341    | 0.1179     | 0.1966  | 0.2444    | 0.2673      |
|   | 1.5          | 0.0000               | - 0.0064         | 0.0993    | 0.2005     | 0.2946  | 0.3525    | 0.3814      |
|   | 2.0          | 0.0000               | 0.0760           | 0.1932    | 0.2996     | 0.3958  | 0.4546    | 0.4842      |
|   | 2.5          | 0.0000               | 0.1903           | 0.3106    | 0.4126     | 0.5001  | 0.5515    | 0.5767      |
|   | 3.0          | 0.0000               | 0.3288           | 0.4442    | 0.5338     | 0.6042  | 0.6419    | 0.6583      |
|   | 3.5          | 0.0000               | 0.4826           | 0.5846    | 0.6549     | 0.7020  | 0.7219    | 0.7269      |
|   | 4.0          | 0.0000               | 0.6420           | 0.7215    | 0.7660     | 0.7861  | 0.7866    | 0.7795      |
|   | 4.5          | 0.0000               | 0.7955           | 0.8434    | 0.8574     | 0.8487  | 0.8305    | 0.8127      |
|   | 5.0          | 0.1151               | 0.9299           | 0.9387    | 0.9197     | 0.8834  | 0.8493    | 0.8236      |
|   | 5.5          | 0.6115               | 1.0310           | 0.9970    | 0.9458     | 0.8861  | 0.8408    | 0.8109      |
|   | 6.0          | 1.0001               | 1.0856           | 1.0103    | 0.9320     | 0.8557  | 0.8050    | 0.7752      |
|   | 6.5          | 1.2775               | 1.0846           | 0.9751    | 0.8783     | 0.7943  | 0.7443    | 0.7185      |
|   | 7.0          | 1.4439               | 1.0249           | 0.8937    | 0.7895     | 0.7070  | 0.6631    | 0.6444      |
|   | 7.5          | 1.4993               | 0.9110           | 0.7737    | 0.6734     | 0.6007  | 0.5671    | 0.5570      |
|   | 8.0          | 1.4439               | 0.7542           | 0.6268    | 0.5404     | 0.4833  | 0.4617    | 0.4602      |
|   | 8.5          | 1.2774               | 0.5710           | 0.4667    | 0.4009     | 0.3620  | 0.3519    | 0.3572      |
|   | 9.0          | 1.0001               | 0.3793           | 0.3066    | 0.2640     | 0.2423  | 0.2409    | 0.2497      |
|   | 9.5          | 0.6117               | 0.1953           | 0.1568    | 0.1361     | 0.1279  | 0.1306    | 0.1385      |
|   | 10.1         | 0.0000               | 0.0000           | 0.0000    | 0.0000     | 0.0000  | 0.0000    | 0.0000      |

problem, the heights of step and inlet channel are set at  $H_{\rm s} = 4.9$  and  $H_{\rm i} = 5.2$ , respectively. For simplicity, a fully developed parabolic inflow profile is prescribed at the inlet channel. The length of the physical domain is twenty times the step height. At outflow boundary, Neumann and Dirichlet boundary conditions for velocity and pressure are specified. Reynolds number for this problem is defined as  $Re = \rho U L / \nu$ , where  $\rho$  is the density which is 1, U the average inflow velocity which is 1 and L the hydraulic length which is equal to  $2H_i$ . Two different Reynolds numbers are considered in this study which are 100 and 389. Uniform grid size  $\Delta x = \Delta y = 0.1$  and time step  $\Delta t = 0.01$  are adopted so that Courant number is approximately equal to 0.15. Simulation times of T = 500.0 and 1000.0 are taken for the cases with low and high Reynolds numbers, respectively.

In order to compare the numerical results with the experimental data [74], normalized velocity profiles (u/U) at different cross sections  $(x/H_s)$  are compared. First of all, streamlines near the inlet channel are shown in Fig. 10. IMLE results show very smooth streamlines of the primary vortices. Secondly, Fig. 11 shows that velocity profiles obtained from IMLE method are in good agreement with the reference experimental data. Finally, Table 2 shows that IMLE outperforms MLE in terms of the computational efficiency. Some discussions about efficiency are detailed in the next section. The degree of computational improvement is defined as the fraction of computation time of MLE method over that of IMLE method. The numerical data of velocity u at different cross sections are shown in Tables 3 and 4 for different Reynolds numbers.

**Table 4** Velocity u at differentcross sections of thebackward-facing step flowproblem at Re = 389

| у    | $x/H_{\rm s}$ |          |         |         |         |         |         |
|------|---------------|----------|---------|---------|---------|---------|---------|
|      | 0.00          | 2.55     | 3.06    | 3.57    | 4.18    | 4.80    | 5.41    |
| 0.0  | 0.0000        | 0.0000   | 0.0000  | 0.0000  | 0.0000  | 0.0000  | 0.0000  |
| 0.5  | 0.0000        | -0.0749  | -0.0865 | -0.0967 | -0.1072 | -0.1150 | -0.1183 |
| 1.0  | 0.0000        | -0.1251  | -0.1409 | -0.1536 | -0.1643 | -0.1685 | -0.1629 |
| 1.5  | 0.0000        | -0.1475  | -0.1606 | -0.1685 | -0.1705 | -0.1621 | -0.1397 |
| 2.0  | 0.0000        | - 0.1399 | -0.1443 | -0.1419 | -0.1290 | -0.1028 | -0.0602 |
| 2.5  | 0.0000        | -0.1014  | -0.0927 | -0.0764 | -0.0455 | -0.0000 | 0.0622  |
| 3.0  | 0.0000        | - 0.0314 | -0.0069 | 0.0248  | 0.0742  | 0.1377  | 0.2165  |
| 3.5  | 0.0000        | 0.0721   | 0.1132  | 0.1603  | 0.2264  | 0.3045  | 0.3948  |
| 4.0  | 0.0000        | 0.2140   | 0.2701  | 0.3300  | 0.4085  | 0.4953  | 0.5896  |
| 4.5  | 0.0000        | 0.4003   | 0.4662  | 0.5329  | 0.6156  | 0.7018  | 0.7896  |
| 5.0  | 0.1139        | 0.6311   | 0.6966  | 0.7601  | 0.8352  | 0.9086  | 0.9773  |
| 5.5  | 0.6118        | 0.8882   | 0.9403  | 0.9890  | 1.0434  | 1.0916  | 1.1296  |
| 6.0  | 1.0001        | 1.1306   | 1.1600  | 1.1854  | 1.2094  | 1.2238  | 1.2239  |
| 6.5  | 1.2775        | 1.3119   | 1.3161  | 1.3152  | 1.3056  | 1.2833  | 1.2437  |
| 7.0  | 1.4439        | 1.4023   | 1.3829  | 1.3569  | 1.3149  | 1.2580  | 1.1827  |
| 7.5  | 1.4994        | 1.3907   | 1.3503  | 1.3021  | 1.2324  | 1.1472  | 1.0451  |
| 8.0  | 1.4439        | 1.2754   | 1.2177  | 1.1527  | 1.0636  | 0.9614  | 0.8468  |
| 8.5  | 1.2775        | 1.0609   | 0.9935  | 0.9209  | 0.8264  | 0.7235  | 0.6142  |
| 9.0  | 1.0001        | 0.7635   | 0.6996  | 0.6339  | 0.5519  | 0.4664  | 0.3801  |
| 9.5  | 0.6118        | 0.4176   | 0.3742  | 0.3310  | 0.2787  | 0.2262  | 0.1756  |
| 10.1 | 0.0000        | 0.0000   | 0.0000  | 0.0000  | 0.0000  | 0.0000  | 0.0000  |

$$u = 0$$

$$v = 0$$

$$\frac{\partial p}{\partial x} = 0$$

$$\vec{u}(x, y, t = 0) = 0$$

$$u = 0, v = 0, \frac{\partial p}{\partial y} = 0$$

$$(x, y) = (1,1)$$

$$u = 0$$

$$v = 0$$

$$\frac{\partial p}{\partial x} = 0$$

Fig. 12 Schematic of the lid-driven cavity flow problem

#### 4.3 Lid-driven cavity flow

In the last validation study, a lid-driven cavity flow in a unit square domain is considered. The physical domain, the initial, and the boundary conditions are schematically shown in Fig. 12. For this problem, characteristic velocity and length and density are 1. Reynolds numbers of 1000 and 10,000 are considered, while the corresponding simulation times are 400.0 and 1000.0. Grid numbers  $81^2$  and  $201^2$  are adopted for

| Table 5         Grid settings for MLE           simulations of the lid-driven |    | Re      |          |  |
|---|----|---------|----------|--|
| cavity flow problem   |    | 1000    | 10,000   |  |
|   |    | 41-1(1) | 81-1(4)  |  |
|   | *  | 41-4(2) | 81-4(5)  |  |
|   | ** | 81-1(3) | 201-1(6) |  |

low and high Reynolds numbers, respectively. For comparisons, grid settings for MLE simulations are listed in Table 5, where the notation m - n means that m is the number of grid points and n is the maximum number of particles in a cell. The row with symbol \* indicates that the numerical results of those settings are shown in [43]. The row with symbol \*\* indicates that under these settings, particle numbers are the same for both MLE and IMLE methods. Case numberings are shown in parentheses.

Usually, velocity profiles u(y) and v(x) at the cross sections x = 0.5 and y = 0.5 are compared with the reference data obtained from [75] for benchmarking purpose. As shown in Fig. 13, under the same number of grids/particles  $(nc^2 = 80^2, 200^2)$ , IMLE method shows better accuracy than that of MLE(3)/MLE(6). Both high accuracy order of approximating diffusion terms and smaller MLS interpolation error can result in this result. On the other hand, for the previously proposed MLE method, larger values of *n* 



Fig. 13 Comparison of the velocity profiles between IMLE, MLE and Ghia [75] for different Reynolds numbers of the lid-driven cavity flow problem

(more particles) with the same number of grid points show more accurate velocity profiles (comparing cases (1)-(2) and (4)–(5)). Even though cases (2) and (5) show more accurate velocity profiles, velocity and pressure contours exhibit oscillations near the walls (top row of Figs. 14 and 15). The problems of oscillating contours can be partly resolved by increasing grid number as shown in the middle row of Figs. 14 and 15. Contours of u and v show no oscillations, while pressure contours remain unsmooth. However, by comparing the middle and bottom rows in Figs. 14 and 15, IMLE method shows even more smoother contours both for u, v, and punder the same number of grids/particles. For the comparison with the conventional particle method, IMLE method shows accurate results with smaller number of grids and particles  $(nc^2 = 80^2 \text{ and } 200^2)$ , while SPH method reported in [55] used much larger number of particles ( $N = 200^2$  and  $400^2$ ). Therefore, IMLE method is said to be more accurate than the previously proposed MLE method. The numerical data of velocities u(y) and v(x) at the cross sections x = 0.5and y = 0.5 are shown in Table 6.

On the other hand, computational times for cases (3), (6) and IMLE are shown in Tables 7 and 8. There are some major issues regarding to the save of computational times in each step. For calculating intermediate velocities  $\mathbf{u}^*$ , MLE method requires an additional work to accomplish the interpolation (step 5 in algorithm 1). Therefore, it is less efficient than IMLE method. In MLS interpolation, because the neighboring particles are explicitly known in IMLE method, no additional work is needed to proceed particle searching procedure in IMLE method. For solving the MLS matrix equation, only backward elimination procedure is taken in MLS interpolation in IMLE method. In addition, since Lagrangian particles are re-meshed to Eulerian grids, there is no need to use linked-list to manage particle distribution. Therefore, IMLE method is more efficient than the previously proposed MLE method.

The most important observation to be pointed out is that both conventional particle and MLE and IMLE methods solve transient Navier-Stokes equations because the total derivative terms are solved. It is not easy for MLE method to reach steady state solutions, while application of IMLE method can get nominally steady state solutions. In other words, by computing  $L2_u = RMS(\mathbf{u}^n - \mathbf{u}^{n-1})$  or  $L2_p = RMS(p^n - p^{n-1})$ , where the superscripts n and n-1 are time levels, IMLE method shows nominally steady state solution for lid-driven cavity flow problem at both low and high Reynolds numbers as shown in Fig. 16, while MLE method does not. To the authors' best knowledge, it is the first time for a Lagrangian-based method to show the convergence profiles. On the other hand, while the computed solutions are reaching steady state, iteration number for the application of CG to solve the PPE is decreased to one. A considerable amount of computing time can be saved. Figure 17 shows the results mentioned above. Therefore, IMLE method is clearly shown to be more stable than the previously proposed MLE method.

After illustrating the advantages of the proposed IMLE method in terms of the accuracy, efficiency and stability, the effects of numerical schemes on the accuracies of the



**Fig. 14** Numerical results of the lid-driven cavity flow problem at Re = 1000 (left column: contour u; middle column: contour v; right column: contour p; top row: case of MLE(2); middle row: case of MLE(3); bottom row: case of IMLE

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**Fig. 15** Numerical results of the lid-driven cavity flow problem at Re = 10,000 (left column: contour *u*; middle column: contour *v*; right column: contour *p*; top row: case of MLE(5); middle row: case of MLE(6); bottom row: case of IMLE

**Table 6** Velocities *u* and *v* at the cross sections x = 0.5 and y = 0.5 for different Reynolds numbers of the lid-driven cavity flow problem

| y or x | Re = 1000     |               | Re = 10,000   |               |
|--------|---------------|---------------|---------------|---------------|
|        | u(x = 0.5, y) | v(x, y = 0.5) | u(x = 0.5, y) | v(x, y = 0.5) |
| 0.00   | 0.0000        | 0.0000        | 0.0000        | 0.0000        |
| 0.05   | -0.1558       | 0.2257        | -0.4094       | 0.4012        |
| 0.10   | -0.2706       | 0.3120        | -0.3627       | 0.3833        |
| 0.15   | -0.3483       | 0.3485        | - 0.3139      | 0.3243        |
| 0.20   | -0.3565       | 0.3399        | -0.2676       | 0.2730        |
| 0.25   | -0.3090       | 0.2956        | -0.2239       | 0.2248        |
| 0.30   | - 0.2493      | 0.2378        | -0.1818       | 0.1791        |
| 0.35   | -0.1976       | 0.1812        | -0.1411       | 0.1352        |
| 0.40   | -0.1513       | 0.1285        | -0.1012       | 0.0927        |
| 0.45   | -0.1058       | 0.0777        | - 0.0619      | 0.0510        |
| 0.50   | -0.0597       | 0.0272        | -0.0226       | 0.0097        |
| 0.55   | - 0.0129      | -0.0237       | 0.0169        | -0.0317       |
| 0.60   | 0.0349        | -0.0754       | 0.0576        | -0.0738       |
| 0.65   | 0.0846        | -0.1285       | 0.0999        | -0.1174       |
| 0.70   | 0.1374        | -0.1827       | 0.1446        | -0.1630       |
| 0.75   | 0.1945        | -0.2373       | 0.1929        | - 0.2113      |
| 0.80   | 0.2553        | -0.2976       | 0.2458        | -0.2631       |
| 0.85   | 0.3138        | -0.3901       | 0.3049        | - 0.3191      |
| 0.90   | 0.3579        | -0.4918       | 0.3724        | -0.3808       |
| 0.95   | 0.4350        | -0.3511       | 0.4445        | -0.4620       |
| 1.00   | 1.0000        | 0.0000        | 1.0000        | 0.0000        |
|        |               |               |               |               |

| <b>Table 7</b> Computational times |
|------------------------------------|
| for the MLE(3) and IMLE at         |
| Re = 1000 of the lid-driven        |
| cavity flow problem                |

| Program     | Case (3)  |           | IMLE      |           |         |
|-------------|-----------|-----------|-----------|-----------|---------|
|             | Time (s)  | Prop. (%) | Time (s)  | Prop. (%) | Speedup |
| Main        | 1.1796E+4 | 100.00    | 2.4201E+3 | 100.00    | 4.87    |
| u*          | 2.7623E+2 | 2.34      | 1.3733E+2 | 5.67      | 2.01    |
| MLS intrpl. | 6.8301E+3 | 57.90     | 8.8174E+2 | 36.43     | 7.75    |
| $p^{n+1}$   | 4.0819E+3 | 34.60     | 1.2853E+3 | 53.11     | 3.18    |

**Table 8** Computational timesfor the MLE(6) and IMLE atRe = 10,000 of the lid-drivencavity flow problem

| Program     | Case (6)  |           | IMLE      |           |         |  |  |  |
|-------------|-----------|-----------|-----------|-----------|---------|--|--|--|
|             | Time (s)  | Prop. (%) | Time (s)  | Prop. (%) | Speedup |  |  |  |
| Main        | 7.2239E+5 | 100.00    | 3.2185E+5 | 100.00    | 2.24    |  |  |  |
| <b>u</b> *  | 1.1835E+4 | 1.64      | 9.8702E+3 | 3.07      | 1.20    |  |  |  |
| MLS intrpl. | 2.7382E+5 | 37.90     | 3.3744E+4 | 10.48     | 8.11    |  |  |  |
| $p^{n+1}$   | 4.1168E+5 | 56.99     | 2.7375E+5 | 85.06     | 1.50    |  |  |  |



Fig. 16 Convergence profiles of the MLE (top row) and IMLE (bottom row) methods of the lid-driven cavity flow problem

velocity Laplacian (physical diffusion) term and the interpolation procedure based on the MLS method is examined. In the calculation of velocity Laplacian terms, the conventional second-order centered finite difference (FD2), the fourthorder compact finite difference (CD4), and the proposed sixth-order cell-centered combined compact finite difference (CC-CCD6) schemes are adopted. When performing MLS interpolation, the first- and the second-order accurate schemes can be similarly derived as the derivation of the third-order scheme given in Eq. (19) by setting the truncation errors to be O(x, y) and  $O(x^2, y^2)$ , respectively. Figure 18 compares the numerical results obtained from different Laplacian and MLS interpolation schemes. Note the notation Lapn and MLSm signify that the velocity Laplacian term and the MLS interpolation scheme are calculated by 2n-th- and m-th-order accurate schemes, where n and m are



Fig. 17 Number of CG iterations of the MLE (top row) and IMLE (bottom row) methods of the lid-driven cavity flow problem

1, 2 or 3. For the same velocity Laplacian schemes (left column in Fig. 18), a slight improvement is seen when using the second-order accurate MLS interpolation scheme, as compared with the solutions obtained from the first-order scheme. In addition, there is a significant improvement in accuracy when the third-order accurate MLS interpolation scheme is adopted. On the other hand, the order of accuracy of Laplacian scheme has no significant effect on the numerical results when the same MLS interpolation scheme is adopted. Seemingly, the accuracy of the numerical solution is more sensitive to the MLS interpolation scheme.

Finally, the proposed IMLE method is compared with the collocated finite volume solutions obtained from Open-Foam [76]. Here, the flow case with high Reynolds number



**Fig. 18** Numerical results of adopting different schemes in calculating velocity Laplacian terms and performing MLS interpolation for the liddriven cavity flow problem investigated at Re = 10,000. Lap1, Lap2, and Lap3 indicate that the velocity Laplacian term is approximated by

FD2, CD4, and CC-CCD6 scheme, respectively. MLS1, MLS2, and MLS3 indicate that the MLS interpolation scheme is first-, second-, and third-order accurate, respectively

Fig. 19 Velocity profiles obtained from the proposed IMLE method and the OpenFoam *icoFoam* module with different convective schemes



(Re = 10,000) is considered. The module *icoFoam* in OpenFoam is used, whereby it solves the unsteady incompressible laminar Navier-Stokes equations using the PISO scheme. In this study, cell number  $nc^2 = 200^2$ , time step  $\Delta t = 0.1 \Delta x$  and simulation time T = 1000 are chosen. For OpenFoam, the numerical scheme used for discretizing the partial derivative term  $(\partial \mathbf{u}/\partial t)$  is Euler implicit scheme. For spatial discretization, the pressure gradient term  $(\nabla p)$  and the velocity Laplacian term  $(\nabla^2 \mathbf{u})$  are discretized using the second-order central difference scheme, while the convection term  $(\nabla \cdot \mathbf{u}\mathbf{u})$  is discretized using either the first-order upwind (FUD) or the QUICK schemes. In order to generate the structured Cartesian mesh, the blockMesh module is executed. Figure 19 shows that the predicted velocity profiles using the proposed IMLE method come closer to those of the third-order QUICK scheme, while the FUD scheme (from OpenFoam) shows significant numerical diffusion. Meanwhile, Fig. 20 shows that the contours of velocity components u and v predicted from the proposed IMLE method are significantly smoother than those obtained by using the QUICK scheme. Figure 21 shows the streamlines and the vorticity contours obtained from the proposed IMLE method and the OpenFoam with the QUICK convective scheme. It is obviously seen that use of the proposed IMLE method can get non-oscillating numerical solutions. The unphysical oscillations are mainly arisen due to erroneously introduced phase

and amplitude errors (or false diffusion error). However, the proposed IMLE method is more physically reasonable since moving particles are purely advected along the streamlines at their respective velocities.

# **5** Concluding remarks

In this study, an improved mixed Lagrangian-Eulerian (IMLE) method is successfully developed to solve incompressible Navier-Stokes equations on the basis of the previously proposed MLE method. IMLE method combines the advantages of mesh and meshless methods to yield a higher accuracy order and avoid discretizing convection terms, respectively. The key feature is that by adopting a remeshing procedure, intermediate velocities on Eulerian grids can be calculated without introducing convection terms but with advecting Lagrangian particles and adopting MLS interpolation. In this way, high-order accurate finite difference scheme is adopted to calculate diffusion terms on Eulerian grids, while Lagrangian methods usually show low accuracy order. In order to utilize MLS interpolation for both interior, boundary, and corner cells, some special treatments for calculating intermediate velocities on boundary face particles are proposed. Weakly non-uniform and 3<sup>2</sup>-particle compact



Fig. 20 Contours of the velocity components *u* and *v* obtained from the proposed IMLE method and the OpenFoam *icoFoam* module with FUD and QUICK convective schemes





**Fig. 21** Streamlines and vorticity contours obtained from the solutions of the IMLE method and the OpenFoam with the QUICK scheme. **a** Streamlines obtained from the computed IMLE solutions; **b** streamlines obtained from the OpenFoam solutions using QUICK scheme; **c** vor-

ticity contours obtained from the computed IMLE solutions; **d** vorticity contours obtained from the OpenFoam solutions using QUICK scheme

framework can improve MLS interpolation accuracy. In the last section, IMLE method shows good numerical results for different kinds of problems including periodic, inflowoutflow, and bounded flows. Finally, it is reiterated that for the fully periodic problem, our proposed IMLE method can render fourth- and second-order accuracies for velocity and pressure, respectively. Also, IMLE outperforms MLE in terms of the numerical accuracy, computational efficiency, and scheme stability. It is also worth noting that the proposed method can be easily extended to solve the three-dimensional problems and it is under our current development. Since the proposed IMLE method adopts Eulerian grids to calculate the velocity Laplacian term and solve the PPE, one has to put in extra efforts to confront with difficulties of complex geometries inside the flow domain, irregular physical domain, and distorted free surface. For these issues, the coupled immersed boundary (IB) [77] and IMLE method, and the coupled level-set [10,29] and IMLE method are under our current development. **Funding** This work was supported by the Ministry of Science and Technology (MOST) of Republic of China (R.O.C.) under the Grants MOST 106-2221-E-002-107-MY2.

### **Compliance with ethical standards**

**Conflict of interest** The authors declare that there is no conflict of interest.

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