# Chapter 5: Modeling with Higher-Order Differential Equations

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October 22, 2013

1 Linear Models: Initial-Value Problems

2 Summary

# Modeling with Second Order Linear Differential Equation

We focus on two linear dynamical systems modeled by the following:

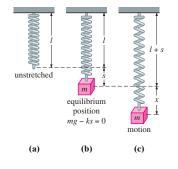
$$ay'' + by' + cy = g(t), \ y(0) = y_0, \ y'(0) = y_1,$$

where the initial conditions are at time t=0.

The two systems are:

- Spring/Mass Systems
- LRC Series Circuits

## Hooke's Law + Newton's Second Law



Assume that the equilibrium position is x=0, and x  $\phi$   $\cap$   $\wedge$   $\wedge$ 

Due to Hooke's Law, net force = mg - k(s + x).

Note that at equilibrium  $\not\ni \not$   $\not$   $\not$   $\Rightarrow$  mg = ks.

Hence, by Newton's Second Law,

$$mx'' = mg - k(s+x) = -kx$$

$$\implies mx'' + kx = 0$$

$$\implies x'' + \frac{k}{m}x = \boxed{x'' + \omega^2 x = 0}$$

where 
$$\omega = \sqrt{k/m}$$
.

## Free Undamped Motion

Solution to  $x'' + \omega^2 x = 0$ :

$$x(t) = c_1 \cos \omega t + c_2 \sin \omega t.$$

Free: No external force ← Homogeneous Equation

**Undamped**: Motion is periodic (period  $=\frac{2\pi}{\omega}$ ), no loss in energy.

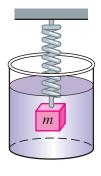
Alternative Representation of x(t):

$$x(t) = A\sin(\omega t + \phi)$$

where

- $A := \sqrt{c_1^2 + c_2^2}$  denotes the **amplitude** of the motion
- ullet  $\phi:= an^{-1}rac{c_1}{c_2}$  denotes the initial phase angle

## Free Damped Motion



Assume that the mass is in a surrounding medium with a resisting force proportional to the velocity.

Net force = 
$$mg - k(s + x) - \beta x'$$
.

Hence, by Newton's Second Law,

$$mx'' = mg - k(s+x) - \beta x' = -kx - \beta x'$$
$$\implies x'' + \frac{\beta}{m}x' + \frac{k}{m}x = \boxed{x'' + \frac{2\lambda x'}{2} + \omega^2 x = 0}$$

where 
$$\omega = \sqrt{k/m}$$
 and  $\lambda = \beta/2m$ .

# Solutions of Free Damped Motion

$$D^2 + 2\lambda D + \omega^2$$
 has two roots  $-\lambda \pm \sqrt{\lambda^2 - \omega^2}$ .

Solution to  $x'' + 2\lambda x' + \omega^2 x = 0$ :

■ Overdamped  $\lambda^2 > \omega^2$ :

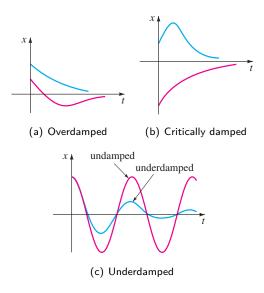
$$x(t) = e^{-\lambda t} \left( c_1 e^{\sqrt{\lambda^2 - \omega^2}t} + c_2 e^{-\sqrt{\lambda^2 - \omega^2}t} \right)$$

• Critically damped  $\lambda^2 = \omega^2$ :

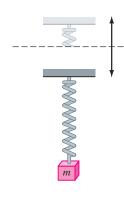
$$x(t) = e^{-\lambda t} \left( c_1 + c_2 t \right)$$

■ Underdamped  $\lambda^2 < \omega^2$ :

$$x(t) = e^{-\lambda t} \left( c_1 \cos \sqrt{\omega^2 - \lambda^2} t + c_2 \sin \sqrt{\omega^2 - \lambda^2} t \right)$$



#### **Driven Motion**



Assume that the certain external force f(t) is applied to the system. For example, the support is vertically oscillating.

Net force = 
$$mg - k(s + x) - \beta x + f(t)$$
.

Hence, by Newton's Second Law,

$$mx'' = mg - k(s+x) - \beta x' + f(t) = -kx - \beta x' + f(t)$$
$$\implies x'' + \frac{\beta}{m}x' + \frac{k}{m}x = \boxed{x'' + 2\lambda x' + \omega^2 x = F(t)}$$

where 
$$\omega = \sqrt{k/m}$$
,  $\lambda = \beta/2m$ , and  $F(t) = f(t)/m$ .

# When F(t) is Periodic

Solve

$$x'' + 2\lambda x' + \omega^2 x = F_0 \sin \gamma t.$$

Find the complementary solution:

$$x_c(t) = \begin{cases} e^{-\lambda t} \left( c_1 e^{\sqrt{\lambda^2 - \omega^2} t} + c_2 e^{-\sqrt{\lambda^2 - \omega^2} t} \right), & \lambda^2 > \omega^2 \\ e^{-\lambda t} \left( c_1 + c_2 t \right), & \lambda^2 = \omega^2 \\ e^{-\lambda t} \left( c_1 \cos \sqrt{\omega^2 - \lambda^2} t + c_2 \sin \sqrt{\omega^2 - \lambda^2} t \right), & \lambda^2 < \omega^2 \end{cases}$$

2 Find a particular solution:

$$x_p(t) = \begin{cases} A\sin\gamma t + B\cos\gamma t, & \lambda \neq 0 \\ A\sin\gamma t + B\cos\gamma t, & \lambda = 0, \ \omega^2 \neq \gamma^2 \\ At\sin\gamma t + Bt\cos\gamma t, & \lambda = 0, \ \omega^2 = \gamma^2 \end{cases}$$

## Driven Damped Motion: Steady-State vs. Transient

When  $\lambda \neq 0$ , it is a **damped** system, and the general solution is  $x(t) = x_c(t) + A \sin \gamma t + B \cos \gamma t$ , where

$$x_c(t) = e^{-\lambda t} \begin{cases} \left( c_1 e^{\sqrt{\lambda^2 - \omega^2} t} + c_2 e^{-\sqrt{\lambda^2 - \omega^2} t} \right), & \lambda^2 > \omega^2 \\ \left( c_1 + c_2 t \right), & \lambda^2 = \omega^2 \\ \left( c_1 \cos \sqrt{\omega^2 - \lambda^2} t + c_2 \sin \sqrt{\omega^2 - \lambda^2} t \right), & \lambda^2 < \omega^2 \end{cases}$$

Note that if  $\lambda > 0$ ,  $x_c(t) \to 0$  as  $t \to \infty$ .

 $\therefore x(t) \to A \sin \gamma t + B \cos \gamma t$  as  $t \to \infty$ . Decompose x(t) into two parts:

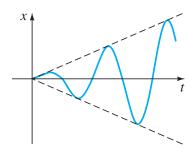
$$x(t) = \underbrace{x_c(t)}_{\text{transient}} + \underbrace{A \sin \gamma t + B \cos \gamma t}_{\text{steady-state}}$$

#### Pure Resonance

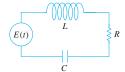
When  $\lambda=0$  and  $\omega^2=\gamma^2$  , it is a  ${\bf undamped}$  system, and the general solution is

$$x(t) = c_1 \cos \omega t + c_2 \sin \omega t + At \sin \omega t + Bt \cos \omega t$$

Note that  $x(t) \to \infty$  as  $t \to \infty$ , which is because of **resonance**.



#### Series Circuit



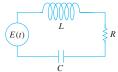
Recall from Chapter 1 that the voltage drop across the three elements are  $L\frac{dI}{dt}$ , IR, and  $\frac{q}{C}$  respectively.

Using the fact that  $I=\frac{dq}{dt}$  and Kirchhoff's Law, we have

$$Lq'' + Rq' + q/C = E(t).$$

- $\blacksquare \ \, {\bf Overdamped} \,\, R^2 > 4L/C$
- Critically damped  $R^2 = 4L/C$
- Underdamped  $R^2 < 4L/C$

# Steady-State Current



#### Example

For the external voltage  $E(t)=E_0\sin\gamma t$ , find the steady-state charge across the capacitor and the steady-state current.

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