Spring 2021

數位控制系統 Digital Control Systems

DCS-35 Input-Output Design - By Polynomial Approach

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Introduction: Model and Analysis and Design

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- Plant (DT):
 - Input-Output Model:

$$\frac{Y_d(z)}{U_d(z)} = G_d(z) = \frac{B_d(z)}{A_d(z)}$$

State-Space Model:

$$\mathbf{x}[k+1] = \mathbf{F}\mathbf{x}[k] + \mathbf{H}u[k]$$
$$y[k] = \mathbf{C}\mathbf{x}[k] + \mathbf{D}u[k]$$

System Properties:

Stability

- Controllability and Reachability
- Observability and Detectability



Discrete State-Space

- Internal model
 - Matrix calculations
- External model
 - Polynomial calculations



Outline

- Process and Controller Models
 - By Rational Transfer functions
- Poles and Zeros
- Command Signals
- Disturbance Response
- Case Study:
 - Double Integrator

Control System Design:

- Command signal following (reference)
- Load disturbance (actuator)
- Measurement noise (sensor)
- Process disturbance (un-modeled dynamics)
- Design Parameters:
 - Closed-loop characteristic polynomial
 - Sampling period

Block Diagram of a Typical Control System:



(z+0.5)(z-1)u

$$= (z+0.3)f - (z^2 - 0.2z + 1)y$$

 $\frac{(z+1)}{(z^2 - 0.2z + 1)}$

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A(z)y(k) = B(z)u(k)

Or, $y(k) = \frac{B(z)}{A(z)}u(k)$

- $A(z) = z^n + a_1 z^{n-1} + \dots + a_n$
- $B(z) = b_1 z^{n-1} + \dots + b_n$
- deg A(q) > deg B(q)
- A(q), B(q): no common factors

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 $\Rightarrow y(k) = \frac{BT}{AR + RS}f(k)$

Closed-Loop Characteristic Polynomial:

$$A_{cl}(z) = A(z)R(z) + B(z)S(z)$$

•
$$G(z) = \frac{B(z)}{A(z)}$$

•
$$G_{\mathsf{fb}}(z) = \frac{S(z)}{R(z)}$$

 \Rightarrow Diophantine equation

Pole-placement design:

 \succ Find R(z) & S(z),

such that **Diophantine equation** is satisfied!

Closed-Loop Characteristic Polynomial:

$$A_{cl}(z) = A(z)R(z) + B(z)S(z)$$
$$= A_c(z) \quad A_o(z)$$

 $\Rightarrow A_c(z) = \det(zI - F + HK)$

controller polynomial

 $\Rightarrow A_o(z) = \det(zI - F + LC)$

observer polynomial

- If controllable \Rightarrow any eig $\rightarrow A_c(z)$
- If observable \Rightarrow any eig $\rightarrow A_o(z)$

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- Algorithm 5.1: (Simple Pole-Placement Design)
 - Data: $\frac{B(z)}{A(z)}$ and $A_{cl}(z)$
 - A(z), B(z) do not have common factors
 - $A_{cl}(z)$ has desired specification
 - Step 1: Find R(z), S(z)
 - $\deg(S(z)) \leq \deg(R(z))$

Satisfy

 $A(z)R(z) + B(z)S(z) = A_{cl}(z)$

- Algorithm 5.1: (Simple Pole-Placement Design)
 - Step 2: Write $A_{cl}(z) = A_c(z)A_o(z)$

• $\deg(A_o(z)) \leq \deg(R(z))$

Select
$$T(z) = t_o A_o(z)$$

•
$$t_o = \frac{A_c(1)}{B(1)}$$

- Controller Law: R(z)u(k) = T(z)f(k) - S(z)y(k)
- Response to command signals: $A_c(z)y(k) = t_o B(z)f(k)$

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Example 5.1: (Control of a double integrator)

$$\frac{1}{s^2} \iff \frac{h^2(z+1)}{2(z-1)^2}$$
$$\Rightarrow A(z) = (z-1)^2$$
$$B(z) = \frac{h^2}{2}(z+1)$$

• Diophantine equation

$$A_{cl}(z) = (z-1)^2 R(z) + \frac{h^2}{2} (z+1) S(z)$$

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- Example 5.1: (Control of a double integrator)
 - Try $R(z) = 1, S(z) = s_0$

 \Rightarrow This is P controller, b/c $G_{fb} = \frac{S(z)}{R(z)}$

$$\Rightarrow A_{cl}(z) = (z^2 - 2z + 1) + \frac{s_0 h^2}{2}(z + 1)$$

 \Rightarrow Impossible for any $A_{cl}(z)$ of 2nd order

Example 5.1: (Control of a double integrator)

• Try
$$R(z) = z + r_1, S(z) = s_0 z + s_1$$

 \Rightarrow This is 1st-order controller,

$$\Rightarrow A_{cl}(z) = (z^2 - 2z + 1)(z + r_1) + \frac{h^2}{2}(z + 1)(s_0 z + s_1)$$

$$\Rightarrow z^{3} + \left(r_{1} + \frac{h^{2}}{2}s_{0} - 2\right)z^{2} + \left(1 - 2r_{1} + \frac{h^{2}}{2}(s_{0} + s_{1})\right)z + r_{1} + s_{1}\frac{h^{2}}{2}z^{2}$$

 $\Rightarrow r_1, s_0, s_1$ for any $A_{cl}(z)$ of 3rd order

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Example 5.1: (Control of a double integrator)

• Try
$$R(z) = z + r_1, S(z) = s_0 z + s_1$$

$$\Rightarrow$$
 If $A_{cl}(z) = z^3 + p_1 z^2 + p_2 z + p_3$

$$\Rightarrow \qquad r_1 + \frac{h^2}{2} s_0 = p_1 + 2 \\ -2r_1 + \frac{h^2}{2} (s_0 + s_1) = p_2 - 1 \\ r_1 + s_1 \frac{h^2}{2} = p_3 \end{cases}$$

$$\Rightarrow r_1 = \frac{3+p_1+p_2-p_3}{4} \\ s_0 = \frac{5+3p_1+p_2-p_3}{2h^2} \\ s_1 = -\frac{3+p_1-p_2-3p_3}{2h^2}$$

Example 5.1: (Control of a double integrator)

• $T(z) = t_0 A_o(z)$

b/c:
$$A_{cl}(z) = A_c(z)A_o(z)$$

 $\Rightarrow A_c(z) \text{ of 2nd order}$ $A_o(z) \text{ of 1st order}$

And, let
$$t_0 = \frac{A(1)}{B(1)}$$



- A^+, B^+ are nice polynomials
 - i.e., their roots are inside the unit disci.e., they can be canceled

$$R = B^{+} \bar{R}$$

$$\Rightarrow S = A^{+} \bar{S}$$

$$T = A^{+} \bar{T}$$

$$\Rightarrow \frac{BT}{AR + BS}$$

$$\Rightarrow \frac{(B^{+}B^{-})(A^{+}\bar{T})}{(A^{+}A^{-})(B^{+}\bar{R}) + (B^{+}B^{-})(A^{+}\bar{S})}$$

$$\Rightarrow \frac{B^{-}\bar{T}}{A^{-}\bar{R} + B^{-}\bar{S}}$$

Feng-Li Lian © 2021 **Realistic Design Problem** DCS35-InOutDesign-19 $A = A^+ A^-$ Closed-Loop Characteristic Polynomial: $B = B^{+} B^{-}$ $R = B^+ \bar{R}$ • $A_{cl} = AR + BS$ $S = A^+ \bar{S}$ $T = A^+ \overline{T}$ $= (A^+A^-)(B^+\bar{R}) + (B^+B^-)(A^+\bar{S})$ $= A^+ B^+ \left(A^- \bar{R} + B^- \bar{S} \right)$ $= A^{+}B^{+}(\bar{A}_{cl})$ • b/c: $A_{cl} = A_c A_o = \left(B^+ \bar{A}_c\right) \left(A^+ \bar{A}_o\right)$ $\Rightarrow A^{-}\bar{R} + B^{-}\bar{S} = \bar{A}_{cl} = \bar{A}_c\bar{A}_o$

Minimum-Degree Causal Controller:

$$\Rightarrow A^{-}\bar{R} + B^{-}\bar{S} = \bar{A}_{cl} = \bar{A}_c\bar{A}_o$$

• It is unique, if $deg(\overline{S}) < deg(A^-)$





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 If unstable modes are canceled, they are uncontrollable or unobservable

Disturbance & Command Signal Response:

$$A = A^+ A^-$$

$$B = B^+ B^- \quad \bullet A^+, B^+ \text{ are canceled nice polynomials}$$

Desired response to command signals:

$$y_m = G_m f = \frac{B_m}{A_m} f$$

For <u>perfect model following:</u>

$$B_m = \bar{B}_m B^-$$



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$$R = A_m B^+ \bar{R}$$

$$S = A_m A^+ \bar{S}$$

$$T = \bar{B}_m \bar{A}_o \bar{A}_c A^+$$

 $A = A^+ A^ B = B^+ B^ B_m = \bar{B}_m B^-$

Control Law:

$$R(z)u(k) = T(z)f(k) - S(z)y(k)$$

$$\Rightarrow u(k) = \frac{T(z)}{R(z)}f(k) - \frac{S(z)}{R(z)}y(k)$$

$$\Rightarrow u(k) = \frac{A^+}{B^+} \left(\frac{\bar{B}_m\bar{A}_o\bar{A}_c}{A_m\bar{R}}f - \frac{\bar{S}}{\bar{R}}y\right)$$



Feng-Li Lian © 2021 **Realistic Design Problem** DCS35-InOutDesign-24 $A = A^+ A^-$ From the Diophantine equation: $B = B^+ B^ B_m = \overline{B}_m B^ R = A_m B^+ \bar{R}$ $\bar{A}_{o} \bar{A}_{c} = A^{-} \bar{R} + B^{-} \bar{S}$ $S = A_m A^+ \bar{S}$ $T = \bar{B}_m \bar{A}_o \bar{A}_c A^+$ Hence: $\Rightarrow \frac{\bar{B}_m}{A_m\bar{R}}(\bar{A}_o\bar{A}_c) = \frac{\bar{B}_m(A^-\bar{R}+B^-\bar{S})}{A_m\bar{R}}$ $= \frac{B_m A^-}{A_m} + \frac{B_m B^- S}{\overline{R}}$ $y_m = \frac{B_m}{A_m} f$ $= \frac{B_m A^-}{A_m B^-} + \frac{B_m \bar{S}}{A_m \bar{R}}$ $\Rightarrow u(k) = \frac{A^+}{B^+} \left(\left(\frac{B_m A^-}{A_m B^-} + \frac{B_m \bar{S}}{A_m \bar{R}} \right) f - \frac{\bar{S}}{\bar{R}} y \right)$

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Control Law:

$$\Rightarrow u(k) = \frac{A^+}{B^+} \left(\left(\frac{B_m A^-}{A_m B^-} + \frac{B_m \bar{S}}{A_m \bar{R}} \right) f - \frac{\bar{S}}{\bar{R}} y \right) \qquad \begin{array}{l} A = A^+ A^- \\ B = B^+ B^- \\ B_m = \bar{B}_m B^- \\ R = A_m B^+ \bar{R} \\ S = A_m A^+ \bar{S} \\ T = \bar{B}_m \bar{A}_o \bar{A}_c A^+ \end{array}$$

$$\Rightarrow u = \frac{B_m A}{A_m B} f + \frac{A^+ \bar{S}}{B^+ \bar{R}} (y_m - y)$$

$$G_{\mathsf{ff}} = \frac{B_m A}{A_m B} = \frac{B_m A}{A_m B^+}$$

• Feedback from $e = (y_m - y)$:

$$G_{\mathsf{fb}} = \frac{A^+ \,\bar{S}}{B^+ \,\bar{R}}$$

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Example 5.5 (motor with zero cancellation):

$$G(z) = \frac{K(z-b)}{(z-1)(z-a)}$$
$$G_m(z) = \frac{z(1+p_1+p_2)}{(z^2+p_1z+p_2)}$$

$$A = A^{+} A^{-}$$
$$B = B^{+} B^{-}$$
$$B_{m} = \bar{B}_{m} B^{-}$$
$$R = A_{m} B^{+} \bar{R}$$
$$S = A_{m} A^{+} \bar{S}$$
$$T = \bar{B}_{m} \bar{A}_{o} \bar{A}_{c} A^{+}$$

Cancel the zero z = b:

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Example 5.5 (motor with zero cancellation):

Control Law:

$$A_{cl} = AR + BS$$

$$\Rightarrow A_m = A\bar{R} + B^-\bar{S}$$

Try $\bar{R} = r_0, \bar{S} = s_0 z + s_1$

$$\Rightarrow r_0 = 0$$

$$s_0 = \frac{1+a+p_1}{K}$$

$$s_1 = -\frac{p_2-a}{K}$$

And $T(z) = A_o(z)\overline{B}_m$

$$\Rightarrow \frac{z(1+p_1+p_2)}{K} = t_0 z$$

$$A = A^{+} A^{-}$$

$$B = B^{+} B^{-}$$

$$B_{m} = \bar{B}_{m} B^{-}$$

$$R = A_{m} B^{+} \bar{R}$$

$$S = A_{m} A^{+} \bar{S}$$

$$T = \bar{B}_{m} \bar{A}_{o} \bar{A}_{c} A^{+}$$

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Example 5.5 (motor with zero cancellation):

Control Law:

$$\Rightarrow u(k) = \frac{A^+}{B^+} \left(\frac{\overline{B}_m \overline{A}_o \overline{A}_c}{A_m \overline{R}} f - \frac{\overline{S}}{\overline{R}} y \right)$$

 $A = A^{+} A^{-}$ $B = B^{+} B^{-}$ $B_{m} = \bar{B}_{m} B^{-}$ $R = A_{m} B^{+} \bar{R}$ $S = A_{m} A^{+} \bar{S}$ $T = \bar{B}_{m} \bar{A}_{o} \bar{A}_{c} A^{+}$

 $\Rightarrow u(k) = t_0 f(k) - s_0 y(k) - s_1 y(k-1) + bu(k-1)$

- Example 5.5 (motor with zero cancellation):
- Step response for a motor with pole-placement control
- Specification: $\zeta = 0.7$, $\omega = 1$
- (a) h = 0.25, (b) h = 1.0.



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Example 5.6 (motor w/o zero cancellation):

$$G(z) = \frac{K(z-b)}{(z-1)(z-a)}$$
$$G_m(z) = \frac{(1+p_1+p_2)}{(1-b)} \frac{(z-b)}{(z^2+p_1z+p_2)}$$

$$egin{aligned} B^+ &= 1 \ B^- &= K(z-b) \ A^+ &= 1 \ A_c &= A_m \ A_o &= z \ ar{B}_m &= rac{1+p_1+p_2}{K(1-b)} \end{aligned}$$

Example 5.6 (motor w/o zero cancellation):

Control Law:

$$AR + B^{-}S = A_{m}A_{o}$$

 $\deg S = 1, \deg R = 1$
Try $R = z + r_{1}, S = s_{0}z + s_{1}$
 $u(k) = t_{0}u_{c}(k) - s_{0}y(k) - s_{1}y(k-1) - r_{1}u(k-1)$

- Example 5.6 (motor wi/0 zero cancellation):
- Step response for a motor with pole-placement control
- Specification: $\zeta = 0.7$, $\omega = 1$
- (a) h = 0.25, (b) h = 1.0.



Algorithm 5.3 (General Pole-Placement Design):

Block Diagram:



Algorithm 5.3 (General Pole-Placement Design):

Data:

- Process Model: $\frac{B(z)}{A(z)}$ A(z), B(z): no common factors
- Closed-Loop Chac. Poly.: $A_{cl}(z)$
- Desired Response: $\frac{B_m(z)}{A_m(z)}$
- $R_d(z), S_d(z)$ specify R(z), S(z)

 $G_{\mathsf{ff}} = \frac{B_m A}{A_m R}$

- Algorithm 5.3 (General Pole-Placement Design):
- Pole Excess Condition:
 - $\deg A_m(z) \deg B_m(z) \ge \deg A(z) \deg B(z)$
 - $\deg A_m(z) + \deg B(z) \ge \deg A(z) + \deg B_m(z)$
- Model Following Condition:
 - $B_m = B^- \bar{B}_m$
- Degree Condition:
 - $\deg A_{cl} = 2 \deg A + \deg A_m + \deg R_d + \deg S_d 1$

Algorithm 5.3 (General Pole-Placement Design):

• <u>Step 1:</u>

•
$$A = A^+ A^-$$

• $B = B^+ B^ A^+, B^+$: can be canceled by the controller

Step 2:

• Solve the Diophantine eqn:

$$A^{-} R_{d} \bar{R} + B^{-} S_{d} \bar{S} = \bar{A}_{cl}$$

Algorithm 5.3 (General Pole-Placement Design):

• <u>Step 3:</u>

• the controller:

$$Ru = Tu_c - Sy$$

$$R = A_m B^+ R_d \bar{R}$$

$$S = A_m A^+ S_d \bar{S}$$

$$T = \bar{B}_m A^+ \bar{A}_{cl}$$

$$A = A^+ A^-$$

$$B = B^+ B^-$$

$$B_m = \bar{B}_m B^-$$

$$A_{cl} = A^+ B^+ A_m \bar{A}_{cl}$$

• the C.L. Chac. Poly.:

$$A_{cl} = A^+ B^+ A_m \bar{A}_{cl}$$

- Algorithm 5.3 (General Pole-Placement Design):
- Calculating the Control Law:

$$A^{-} R_{d} \bar{R} + B^{-} S_{d} \bar{S} = \bar{A}_{cl}$$

• $AR^0 + BS^0 = A_{cl}^0$

$$R = A_m B^+ R_d \bar{R}$$

$$S = A_m A^+ S_d \bar{S}$$

$$T = \bar{B}_m A^+ \bar{A}_{cl}$$

$$A = A^+ A^-$$

$$B = B^+ B^-$$

$$B_m = \bar{B}_m B^-$$

$$A_{cl} = A^+ B^+ A_m \bar{A}_{cl}$$

• AU + BV = 0 U, V: min-degree sol

• Define
$$\begin{cases} R = XR^0 + YU \\ S = XS^0 + YV \\ X: \text{ stable monic poly.} \end{cases}$$

$$\Rightarrow AR + BS = XA_{cl}^0 = A_{cl}$$

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Process Model:

•
$$G(s) = \frac{K}{s^2}$$
 $K = 1$

•
$$G(z) = \frac{h^2}{2} \frac{z+1}{(z-1)^2}$$

Specifications:

Closed-Loop Polynomial A_c:

•
$$A_c(s) \Rightarrow s^2 + 2\zeta ws + w^2$$

• $A_c(z) = z^2 - 2ze^{-\zeta wh} \cos\left(wh\sqrt{1-\zeta^2}\right) + e^{-2\zeta wh}$

$$= z^2 + a_{c1}z + a_{c2}$$

Closed-Loop Polynomial A_o:

•
$$A_o(s) \Rightarrow (s+\alpha)^2$$

•
$$A_o(z) = (z - e^{-\alpha h})^2$$

= $z^2 + a_{o1}z + a_{o2}$

- Controller Design:
- The Diophantine Eqn:

$$(z-1)^2 R(z) + \frac{h^2}{2}(z+1)S(z) = A_o(z)A_c(z) = A_{cl}(z)$$

For Integral Action:

• R(z) should have (z-1)

For Minimum-Degree Solution:

- $R(z) \Rightarrow$ 2nd order
- $S(z) \Rightarrow$ 2nd order

Controller Design:

•
$$R(z) = (z+r)(z-1) = z^2 + r_1 z + r_2$$

• $S(z) = s_0 z^2 + s_1 z + s_2$

Straightforward calculations give:



And: $\overline{\bullet} T(z) = \frac{A_c(1)A_o(z)}{B(1)} = \frac{(1+a_{c1}+a_{c2})A_o(z)}{h^2}$

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- Nominal Design:
- Closed-Loop Parameters:
 - $\begin{array}{rcl} \zeta & = & 0.707 \\ w & = & 0.2 \end{array}$
 - $\alpha = 2$
 - h = 1

$$e^{-\alpha h} = 0.135$$

Simulation Study:

Command Signal:

Load Disturbance:

Measurement Noise:

Unit Step Step of -0.05 at time 50 $0.01 \sin 2t$ at time 100

Frequency Folding:

Nyquist Frequency: $0.5 \text{ Hz} = \pi \text{ rad/s}$ Measurement Noise:2 rad/s



- Changing Natural Frequency ω:
- (a) ω = 0.2, 0.1, 0.4; [ζ = 0.707; α = 2; h = 1]
- (b) control signal when $\omega = 0.1$
- (c) control signal when $\omega = 0.4$



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- Changing Damping Ratio ζ:
- (a) $\zeta = 0.707$, 0.5, 1.0; [$\omega = 0.2$; $\alpha = 2$; h = 1]
- (b) control signal when $\zeta = 0.5$
- (c) control signal when $\zeta = 1.0$



- Changing Observer Poles α : $z = e^{-\alpha h}$
- (a) $\alpha = 2, 0.5, 10; [\zeta = 0.707; \omega = 0.2; h = 1]$
- (b) control signal when $\alpha = 0.5$
- (c) control signal when $\alpha = 10$



- Changing Sampling Period h: $z = e^{-\alpha h}$
- (a) $h = 1, 2, 0.1; [\zeta = 0.707; \omega = 0.2; \alpha = 2]$
- (b) control signal when h = 2
- (c) control signal when h = 0.1

