Solution of HW04 for Units 4A, 4B: Feedback Analysis, Control Systems

Assigned: October 22, 2021

Due: November 4, 2021 (23:59)

1. (U4A: Sensitivity)

4. A unity feedback control system has the open-loop transfer function

$$G(s) = \frac{A}{s(s+a)}.$$

- (a) Compute the sensitivity of the closed-loop transfer function to changes in the parameter A.
- (b) Compute the sensitivity of the closed-loop transfer function to changes in the parameter a.
- (c) If the unity gain in the feedback changes to a value of $\beta \neq 1$, compute the sensitivity of the closed-loop transfer function with respect to β .

Solution:

(a)

$$T(s) = \frac{G(s)}{1 + G(s)} = \frac{\frac{A}{s(s+a)}}{1 + \frac{A}{s(s+a)}} = \frac{A}{s^2 + as + A},$$

$$\frac{dT}{dA} = \frac{(s^2 + as + A) - A}{(s^2 + as + A)^2},$$

$$\mathcal{S}_{A}^{T} = \frac{A}{T} \frac{dT}{dA} = \frac{A(s^{2} + as + A)}{A} \frac{s^{2} + as}{(s^{2} + as + A)^{2}} = \frac{s(s + a)}{s(s + a) + A}.$$

(b)

$$\frac{dT}{da} = \frac{-sA}{(s^2 + as + A)^2}.$$

$$\frac{a}{T}\frac{dT}{da} = \frac{a(s^2 + as + A)}{A}\frac{-sA}{(s^2 + as + A)^2}.$$

$$\mathcal{S}_a^T = \frac{-as}{s(s+a) + A}.$$

(c) In this case,

$$T(s) = \frac{G(s)}{1 + \beta G(s)},$$

$$\frac{dT}{d\beta} = \frac{-G(s)^2}{(1 + \beta G(s))^2},$$

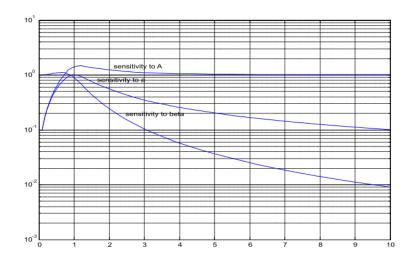
$$\frac{\beta}{T} \frac{dT}{d\beta} = \frac{\beta(1 + \beta G)}{G} \frac{-G^2}{(1 + \beta G)^2} = \frac{-\beta G}{1 + \beta G},$$

$$S_{\beta}^T = \frac{\frac{-\beta A}{s(s+A)}}{1 + \frac{\beta A}{s(s+a)}} = \frac{-\beta A}{s(s+a) + \beta A}.$$

- If a=A=1, the transfer function is most sensitive to variations in a and A near $\omega=1$ rad/sec .
- The steady-state response is not affected by variations in A and a ($\mathcal{S}_A^T(0)$ and $\mathcal{S}_a^T(0)$ are both zero).
- The steady-state response is heavily dependent on β since $|S_{\beta}^{T}(0)| = 1.0$

See attached plots of sensitivities versus radian frequency for a=A=1.0.

Sensitivity function frequency response follows.

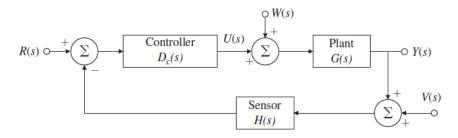


2. (U4B: Steady-State Error and System Type)

8. A standard feedback control block diagram is shown in Fig. 4.5 with

$$G(s) = \frac{1.5}{s}; \ D_c(s) = \frac{(s+9)}{(s+3)}; H(s) = \frac{70}{(s+70)}; V(s) = 0.$$

- (a) Let W = 0 and compute the transfer function from R to Y.
- (b) Let R = 0 and compute the transfer function from W to Y.
- (c) What is the tracking error if R a unit-step input and W = 0?
- (d) What is the tracking error if R is a unit-ramp input and W = 0?
- (e) What is the system type with respect to the reference inputs and the corresponding error coefficient?



Solution:

(a)

$$\frac{Y(s)}{R(s)} = T(s) = F(s) \frac{D_c(s)G(s)}{1 + D_c(s)G(s)H(s)} = \frac{1.5(s+9)(s+70)}{s^3 + 73s^2 + 315s + 945}$$

(b)

$$\frac{Y(s)}{W(s)} = T(s) = \frac{G(s)}{1 + D_c(s)G(s)H(s)} = \frac{1.5(s+3)(s+70)}{s^3 + 73s^2 + 315s + 945}$$

(c)

$$\frac{E(s)}{R(s)} = 1 - T(s) = S(s) = \frac{s^3 + 71.5s^2 + 196.5s}{s^3 + 73s^2 + 315s + 945}.$$
With $R(s) = \frac{1}{s}$,
$$e_{step}(\infty) = \lim_{s \to 0} sE(s) = 0.$$

(d)

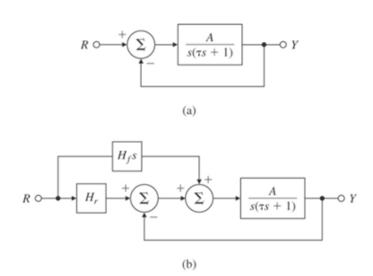
With
$$R(s) = \frac{1}{s^2}$$
,
 $e_{ramp}(\infty) = \lim_{s \to 0} sE(s) = 0.2$.

(e) System is Type 1 and K_v is

$$K_v = \frac{1}{\left| e_{ramp(\infty)} \right|} = 5 \, sec^{-1}.$$

3. (U4B: System Type and Tracking)

- 14. Consider the system shown in Fig. 4.36(a).
 - (a) What is the system type? Compute the steady-state tracking error due to a ramp input $r(t) = r_o t 1(t)$.
 - (b) For the modified system with a feed forward path shown in Fig.4.36(b), give the value of H_f so the system is Type 2 for reference inputs and compute the K_a in this case.
 - (c) Is the resulting Type 2 property of this system robust with respect to changes in H_f i.e., will the system remain Type 2 if H_f changes slightly?



Solution:

(a) System is Type 1 since it is unity feedback and has a pole at s=0in the forward path. Also,

$$E(s) = [1 - \mathcal{T}(s)]R(s),$$

$$= \left[\frac{1}{1 + G(s)}\right]R(s),$$

$$= \frac{s(\tau s + 1)}{s(\tau s + 1) + A}\frac{r_o}{s^2}.$$

The steady-state tracking error using the FVT (assuming stability) is,

$$e_{ss} = \lim_{s \to 0} sE(s) = \frac{r_o}{A}.$$

(b)
$$Y(s) = \frac{A}{s(\tau s + 1)} U(s),$$

$$U(s) = H_f s R(s) + H_r R(s) - Y(s),$$

$$Y(s) = \frac{A(H_f s + H_r)}{s(\tau s + 1) + A} R(s).$$

The tracking error is,

$$E(s) = R(s) - Y(s),$$

$$= \frac{s(\tau s + 1) + A - A(H_f s + H_r)}{s(\tau s + 1) + K} R(s),$$

$$= \frac{\tau s^2 + (1 - AH_f)s + A(1 - H_r)}{s(\tau s + 1) + A}.$$

To get zero steady-state error with respect to a ramp, the numerator in the above equation must have a factor s^2 . For this to happen, let

$$\begin{array}{rcl} H_r & = & 1, \\ AH_f & = & 1. \end{array}$$

Then

$$E(s) = \frac{\tau s^2}{s(\tau s + 1) + A} R(s)$$

and, with $R(s) = \frac{r_o}{s^2}$, apply the FVT (assuming stability) to obtain

$$e_{ss}=0.$$

Thus the system will be Type 2 with $K_a = \frac{\tau}{A}$. $K_a = \frac{A}{\tau}$

$$K_a = \frac{A}{\tau}$$

(c) No, the system is not robust Type 2 because the property is lost if either H_r or H_f changes slightly.