

Spring 2021

控制系統
Control Systems

Unit 50 & 60

Root Locus (s-Domain) and Bode Plot (w-Domain)

Feng-Li Lian

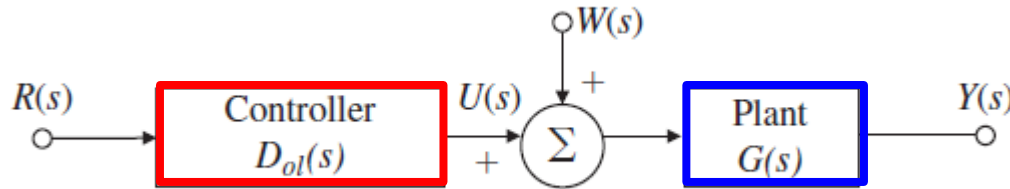
NTU-EE

Feb – Jun, 2021

- Open-loop system showing reference, R, control, U, disturbance, W, and output Y

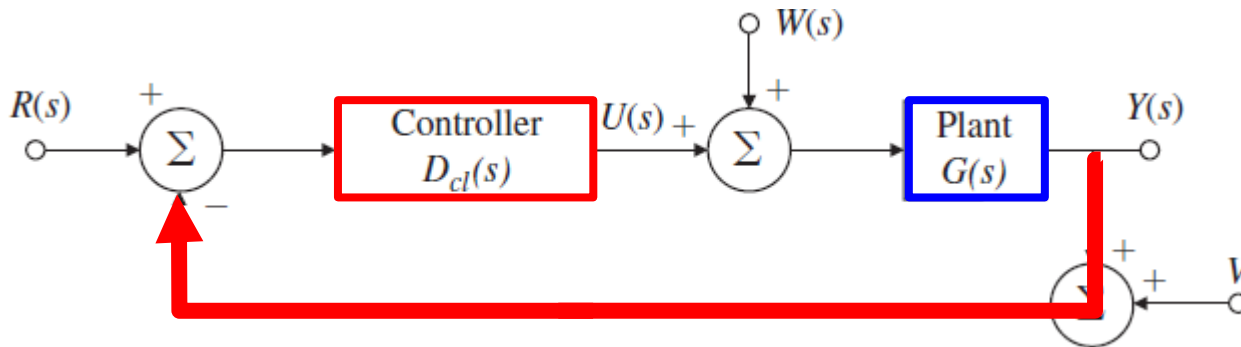
$$G = \frac{b(s)}{a(s)}$$

$$D = \frac{c(s)}{d(s)}$$



$$T_{ol} = \frac{Y_{ol}}{R} = G D_{ol} = \frac{b(s) c(s)}{a(s) d(s)}$$

$$T_{cl} = \frac{Y_{cl}}{R} = \frac{G D_{cl}}{1 + G D_{cl}} = \frac{b(s) c(s)}{1 + a(s) d(s)}$$

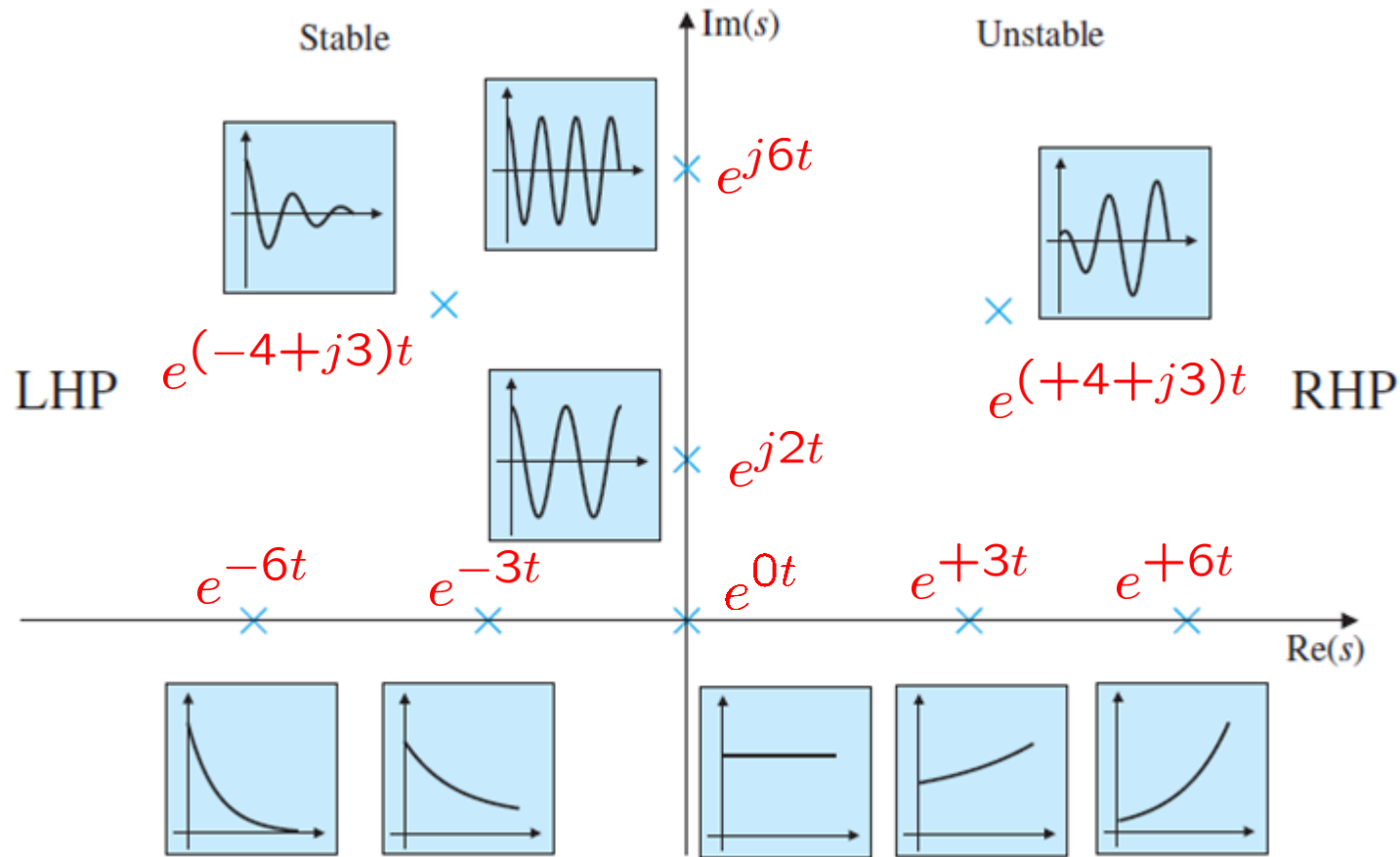


- Poles of Transfer Function
- Roots of Characteristic Equation

- Closed-loop system showing reference, R, control, U, disturbance, W, output, Y, and sensor noise, V

Effect of Pole Locations

- Time functions associated with points in the s-plane
 (LHP, left half-plane; RHP, right half-plane)



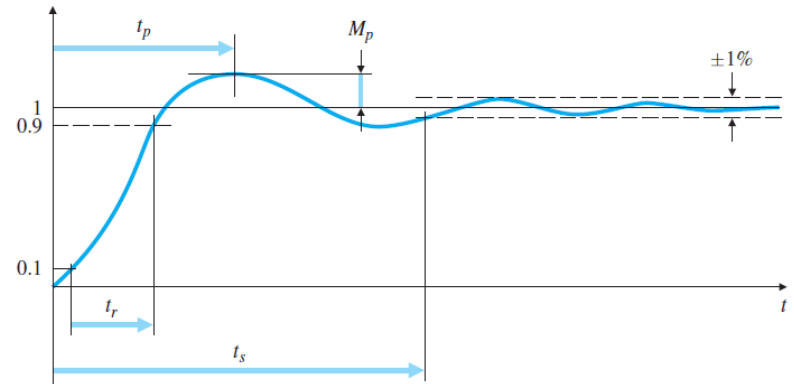
Overhoot M_p and Peak time t_p

$$Y(s) = H(s) \frac{1}{s}$$

$$H(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$\sigma = \omega_n \zeta$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$



$$A \sin(\alpha) + B \cos(\beta) = C \cos(\alpha - \beta)$$

$$C = \sqrt{A^2 + B^2} = \frac{1}{\sqrt{1 - \zeta^2}}$$

$$\beta = \tan^{-1}\left(\frac{A}{B}\right)$$

$$= \tan^{-1}\left(\frac{\zeta}{\sqrt{1 - \zeta^2}}\right) = \sin^{-1}(\zeta)$$

$$\Rightarrow y(t) = 1 - e^{-\sigma t} \left(\cos \omega_d t + \frac{\sigma}{\omega_d} \sin \omega_d t \right)$$

$$\Rightarrow M_p = e^{\frac{-\pi\zeta}{\sqrt{1-\zeta^2}}}$$

$$\Rightarrow y(t) = 1 - \frac{e^{-\sigma t}}{\sqrt{1 - \zeta^2}} \cos(\omega_d t - \beta)$$

$$\Rightarrow t_r \approx \frac{1.8}{\omega_n}$$

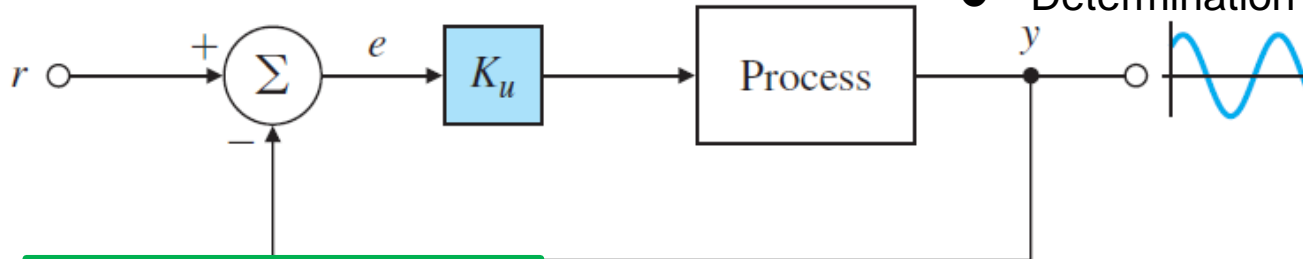
$$\Rightarrow t_s = \frac{4.6}{\zeta\omega_n} = \frac{4.6}{\sigma}$$

- **Method 2: Ultimate Sensitivity Method:**

Based on evaluating the **amplitude and frequency**

of the **oscillations** of the system at the **limit of stability** rather than on taking a step response.

- Determination of ultimate gain and period

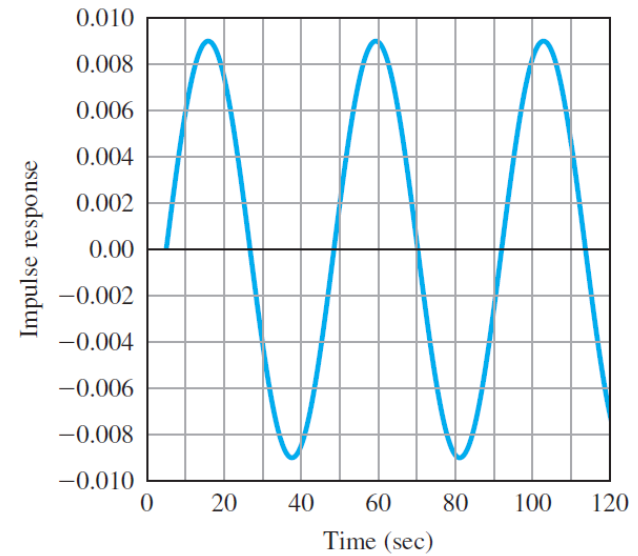


- K_u : Ultimate Gain
- P_u : Ultimate Period

Ziegler-Nichols Tuning for the Regulator

$D_c(s) = k_p(1 + 1/T_I s + T_D s)$, Based on the Ultimate Sensitivity Method

Type of Controller	Optimum Gain
P	$k_p = 0.5K_u$
PI	$\begin{cases} k_p = 0.45K_u \\ T_I = \frac{P_u}{1.2} \end{cases}$
PID	$\begin{cases} k_p = 0.6K_u \\ T_I = 0.5P_u \\ T_D = 0.125P_u \end{cases}$



$$H(s) = \frac{\omega_n^2}{(s + \zeta\omega_n)^2 + \omega_n^2(1 - \zeta^2)}$$

$$\sigma = \omega_n \zeta$$

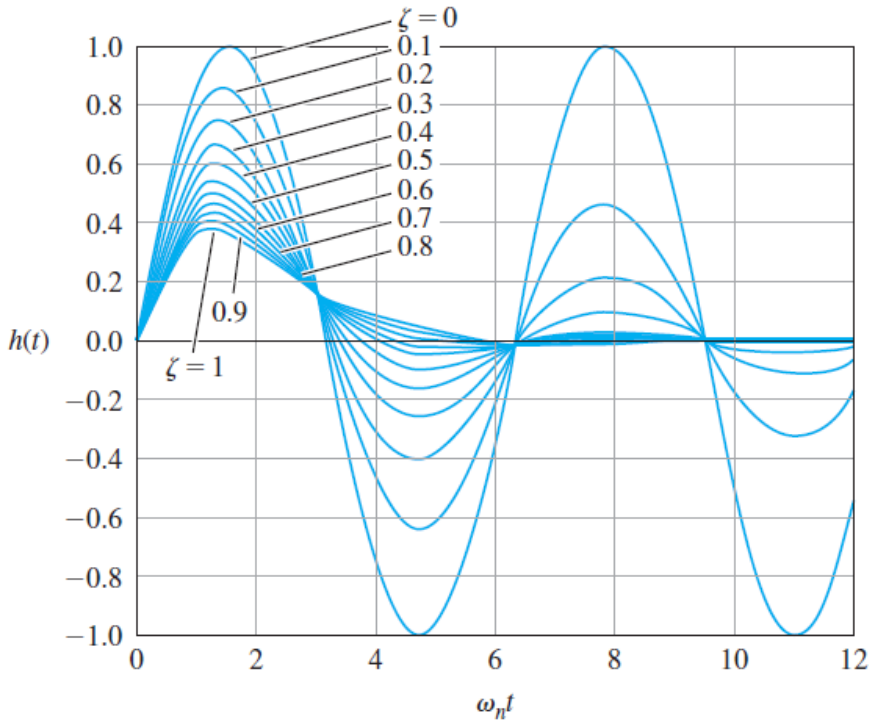
$$h(t) = \frac{\omega_n}{\sqrt{1 - \zeta^2}} e^{-\sigma t} (\sin \omega_d t) 1(t)$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

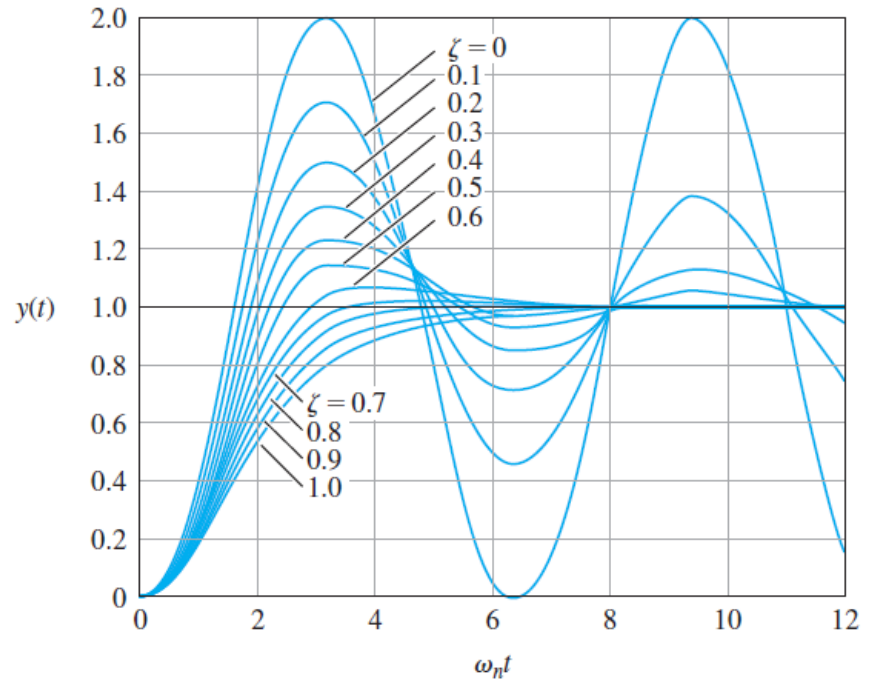
- Responses of second-order systems versus ζ :

(a) Impulse Responses

(b) Step Responses



(a)



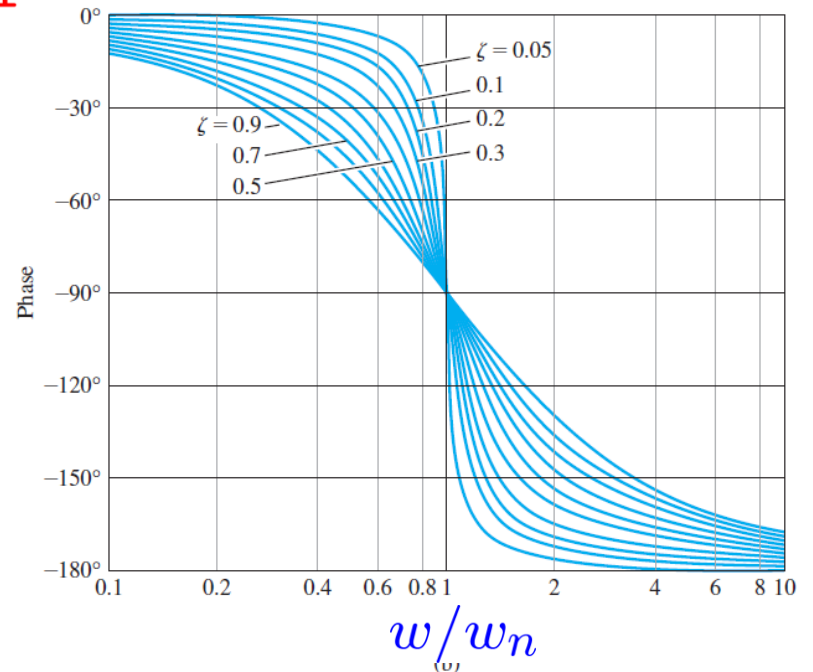
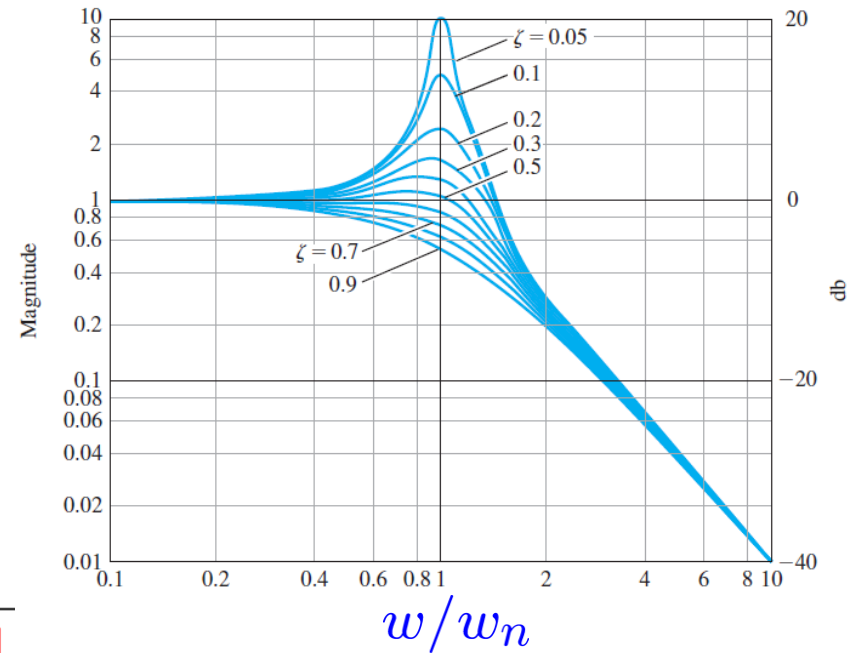
(b)

Frequency Response and Bode Plot

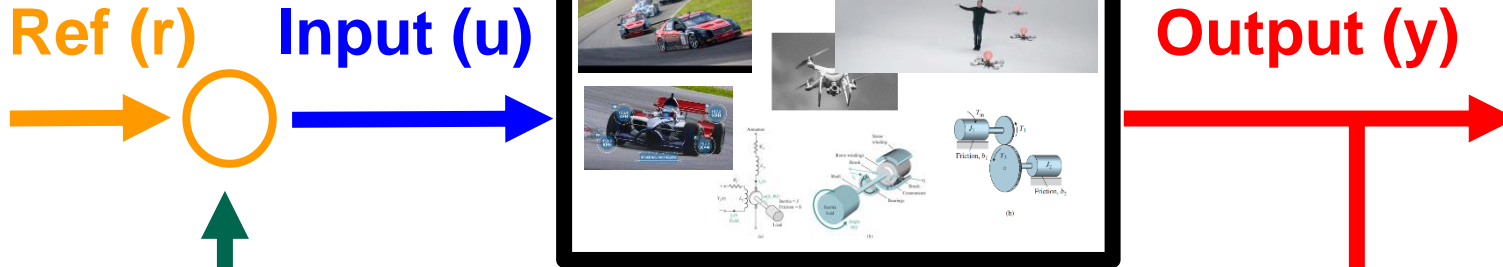
$$|G(s = jw)|$$

$$G(s) = \frac{1}{(s/w_n)^2 + 2\zeta(s/w_n) + 1}$$

$$\angle G(s = jw)$$

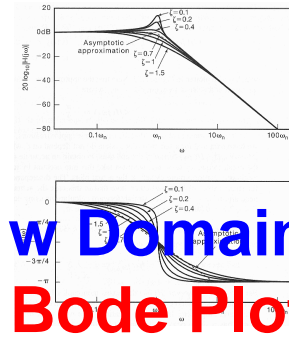
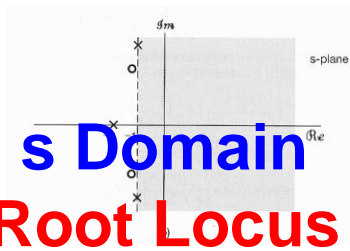


Plant (P)



$$\frac{d^2y(t)}{dt^2} + 2 \frac{dy(t)}{dt} - 3y(t) = 5u(t)$$

$$P(s) = \frac{Y(s)}{U(s)} = \frac{5}{s^2 + 2s - 3}$$



$$\frac{d^2y(t)}{dt^2} + 4 \frac{dy(t)}{dt} + 3y(t) = 3r(t)$$

$$G(s) = \frac{Y(s)}{R(s)} = \frac{3}{s^2 + 4s + 3}$$

1. Model
2. Response
3. Analysis
4. Feedback
5. Control

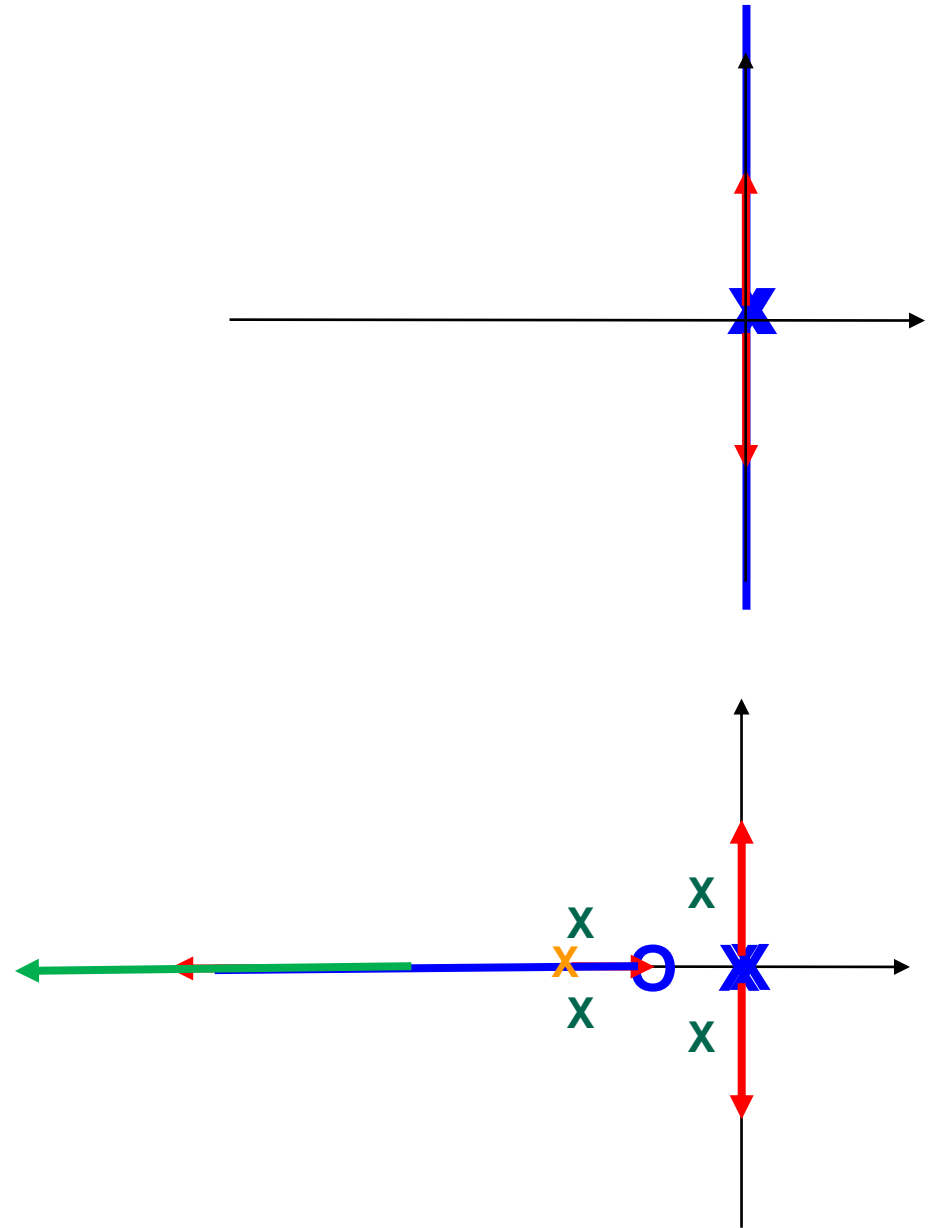
Unit 5

Root Locus

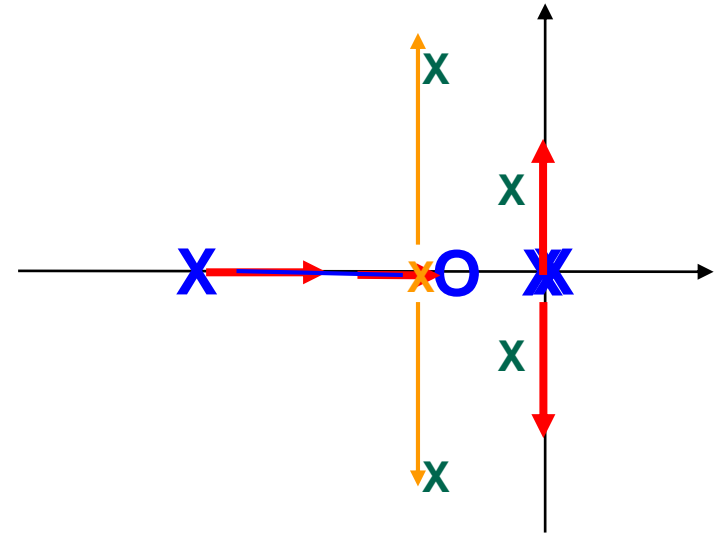
- **By Hand:**
 - Hand Writing in Exam (40%)
 - Use the **5 rules** of Root Locus Method
 - to **roughly sketch** the root locus of any transfer function
 - by **identifying** these **critical** root locations
 - **Properly choose** some roots
 - between these **critical** root locations
- **By Computer:**
 - Multiple Choice in Exam (60%)
 - Use Matlab codes
 - to draw the **exact root locus** of any transfer function
 - **Design proper** transfer function and
 - select associated and reasonable** gain value

$$\Rightarrow 1 + K_P \frac{1}{s^2} = 0$$

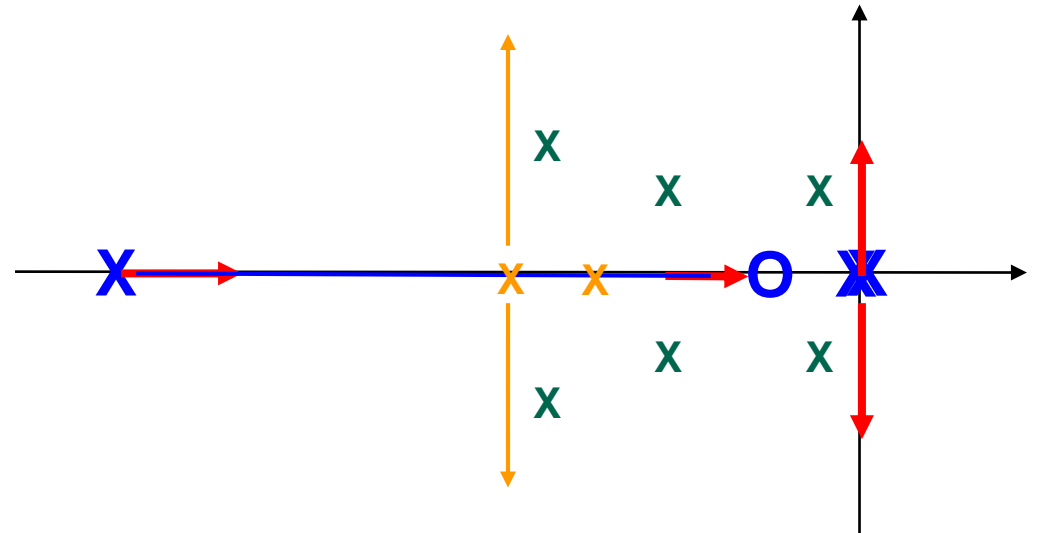
$$\Rightarrow 1 + K \frac{s+1}{s^2} = 0$$



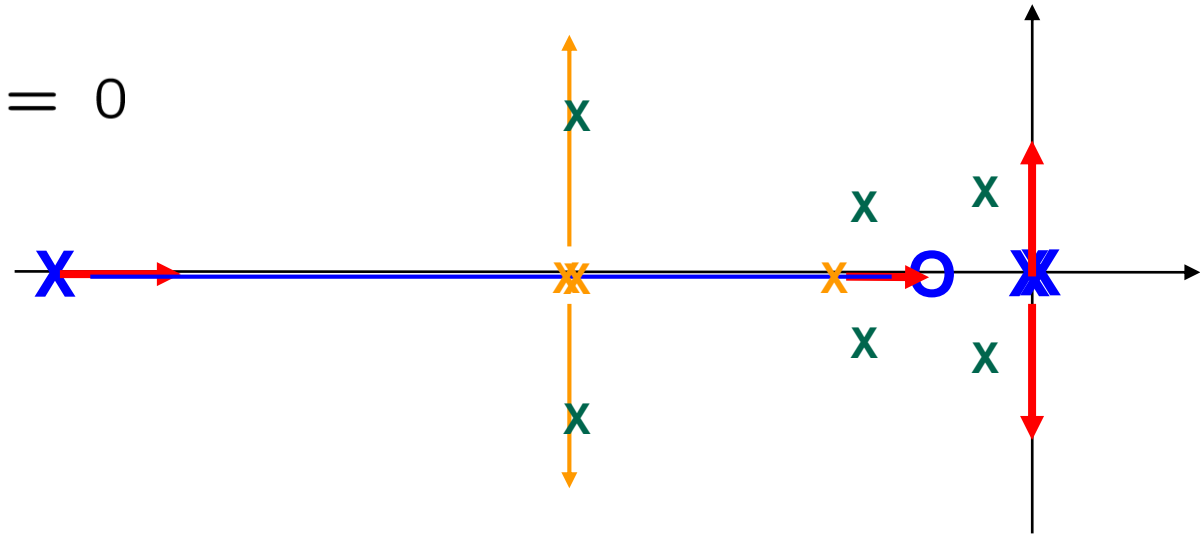
$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 4)} = 0$$



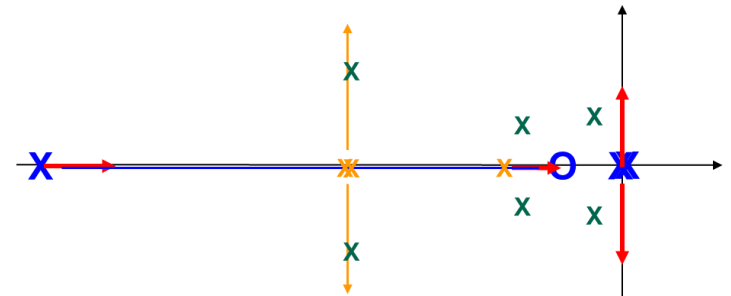
$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 9)} = 0$$



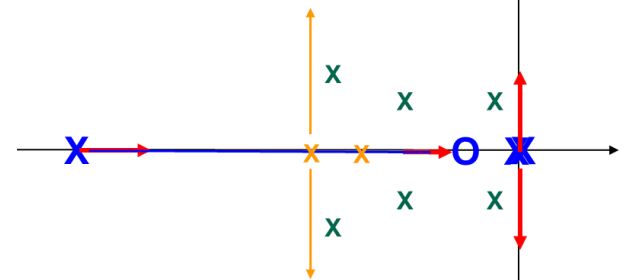
$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 12)} = 0$$



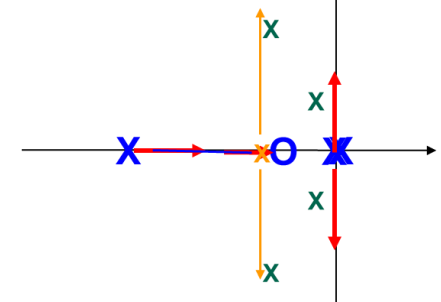
$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 12)} = 0$$



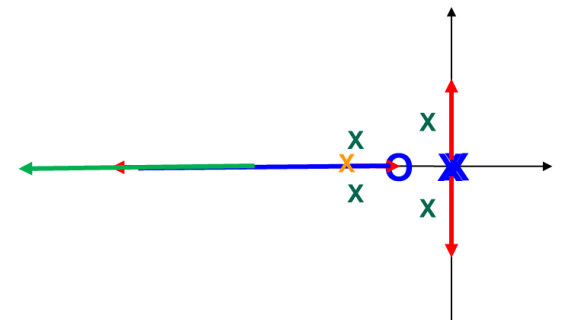
$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 9)} = 0$$



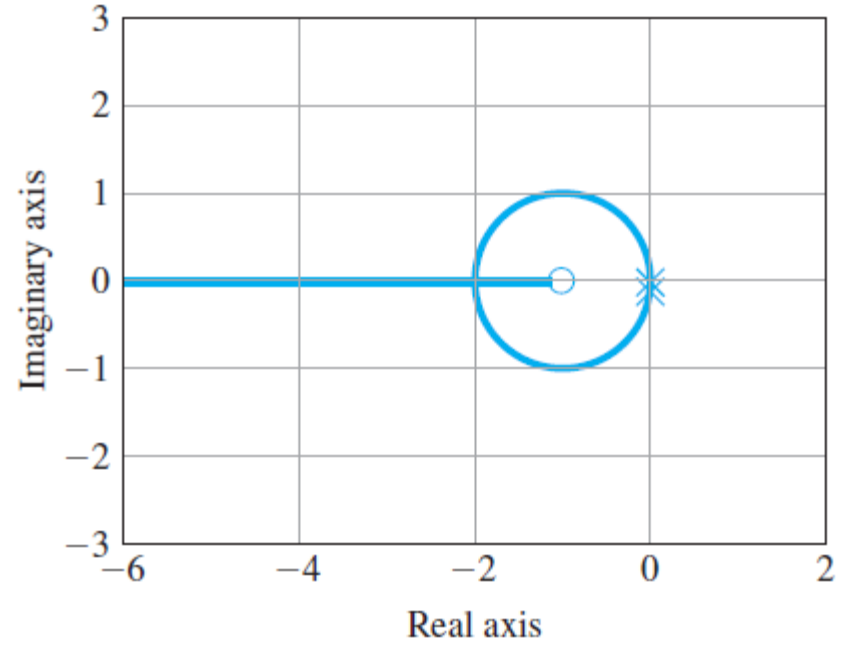
$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 4)} = 0$$



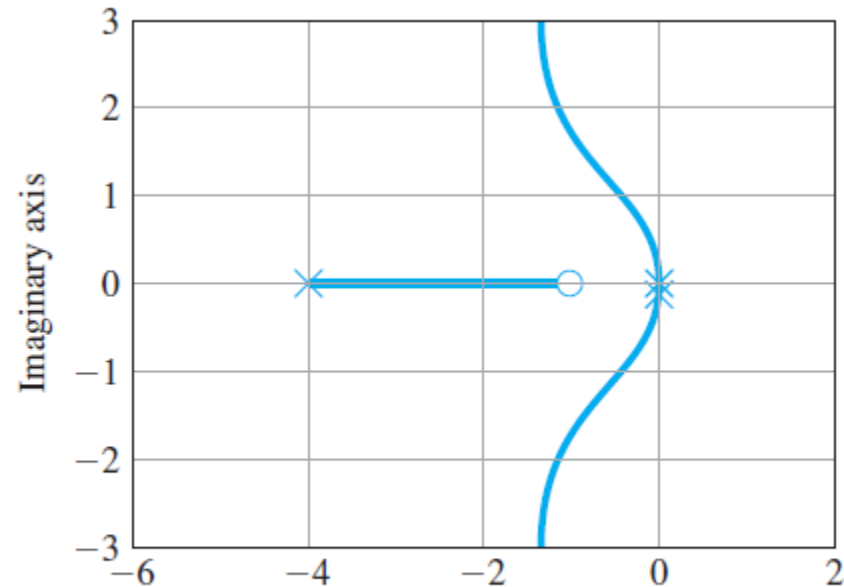
$$\Rightarrow 1 + K \frac{s + 1}{s^2} = 0$$



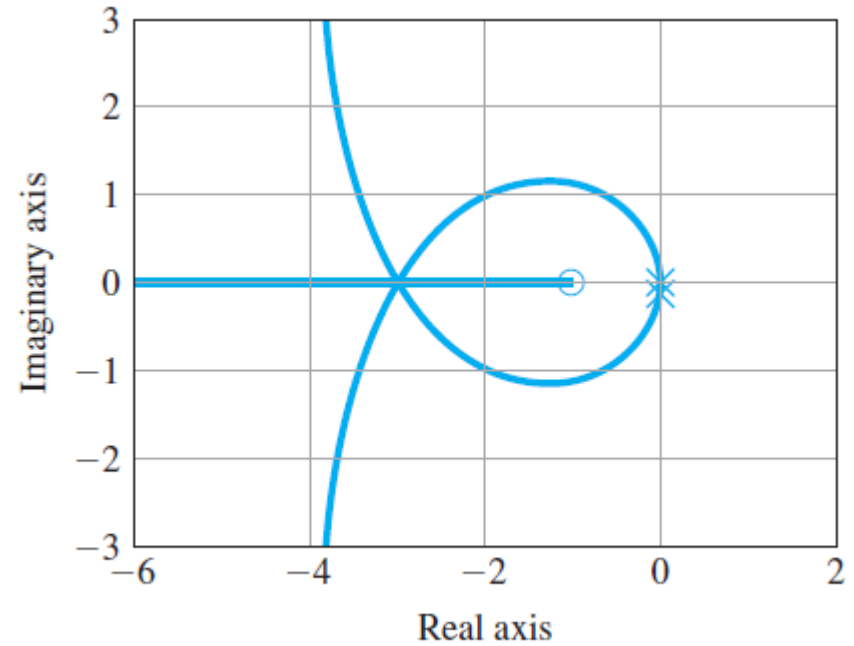
$$\Rightarrow 1 + K \frac{s+1}{s^2} = 0$$



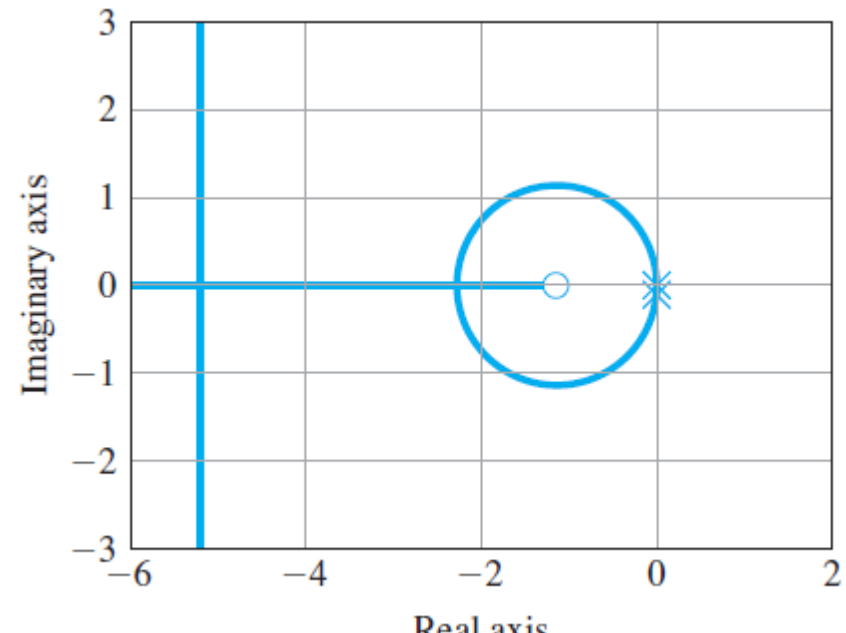
$$\Rightarrow 1 + K \frac{(s+1)}{s^2 (s+4)} = 0$$

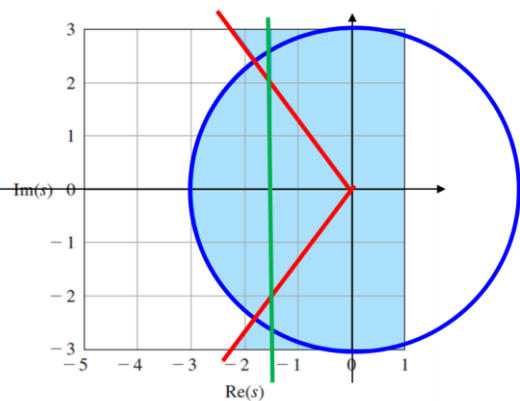
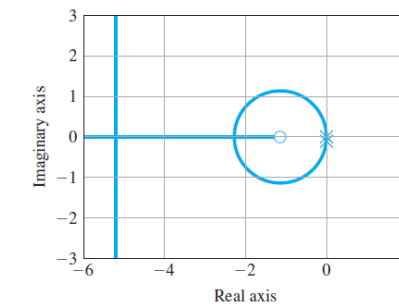
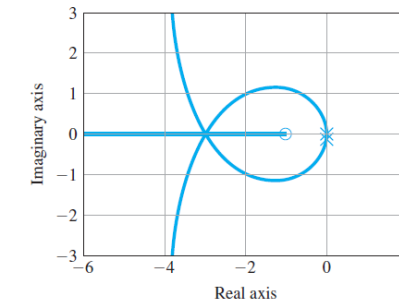
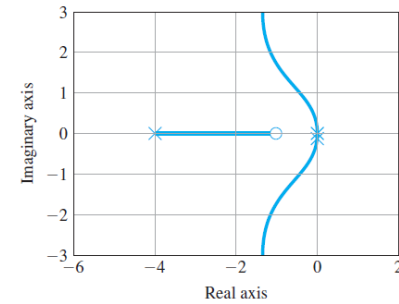
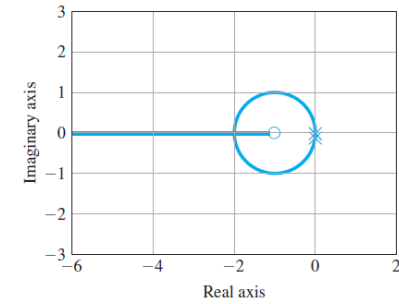
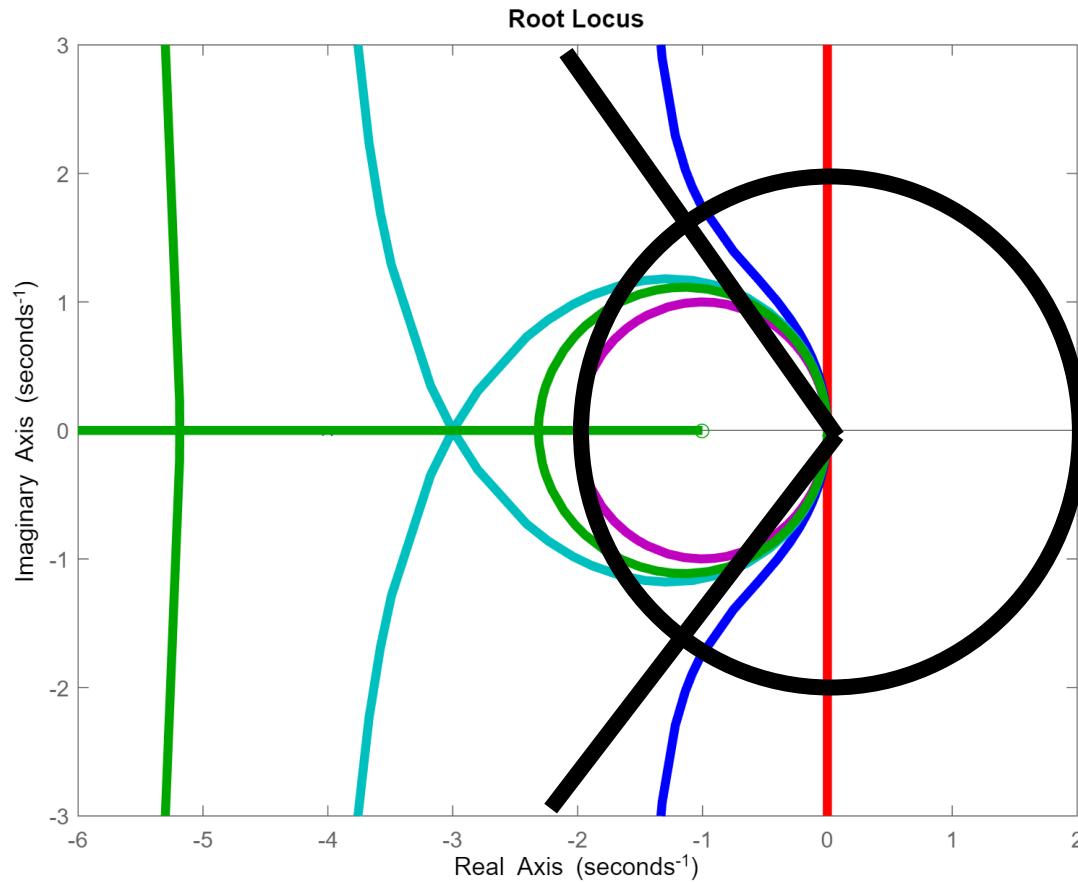


$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 9)} = 0$$



$$\Rightarrow 1 + K \frac{(s + 1)}{s^2 (s + 12)} = 0$$

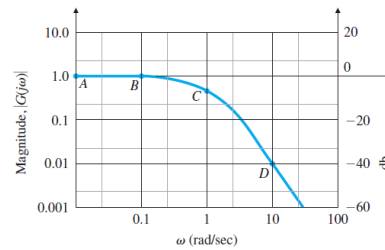
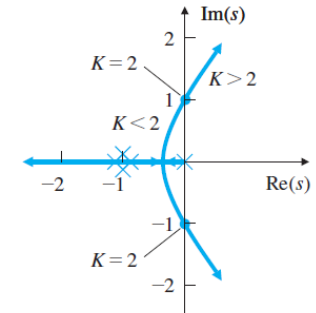
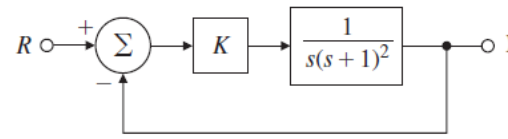
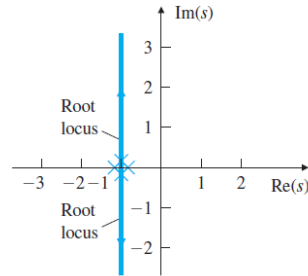
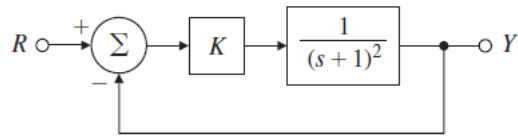




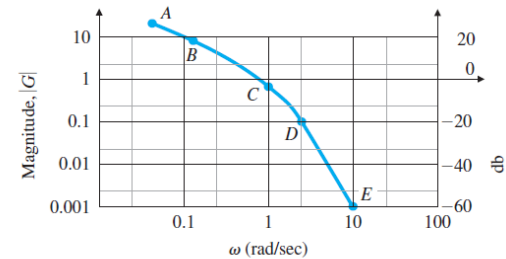
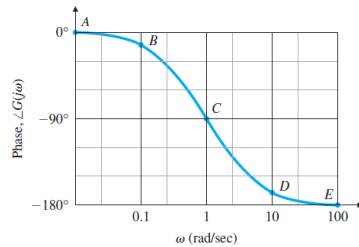
- Proper transfer function
- Reasonable gain values

Unit 6

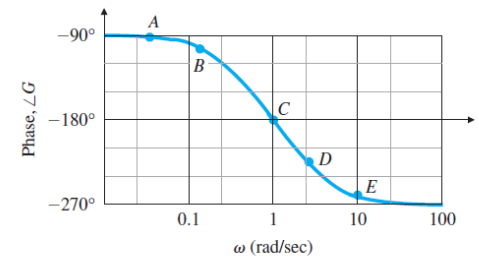
Bode Plot



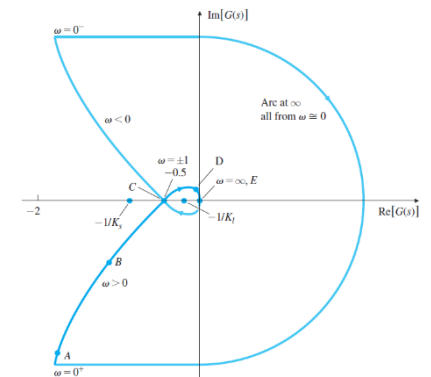
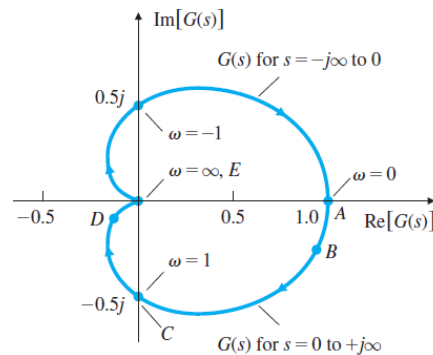
(a)



(b)



- Root Locus:
- Bode Plot:
- Nyquist Plot:



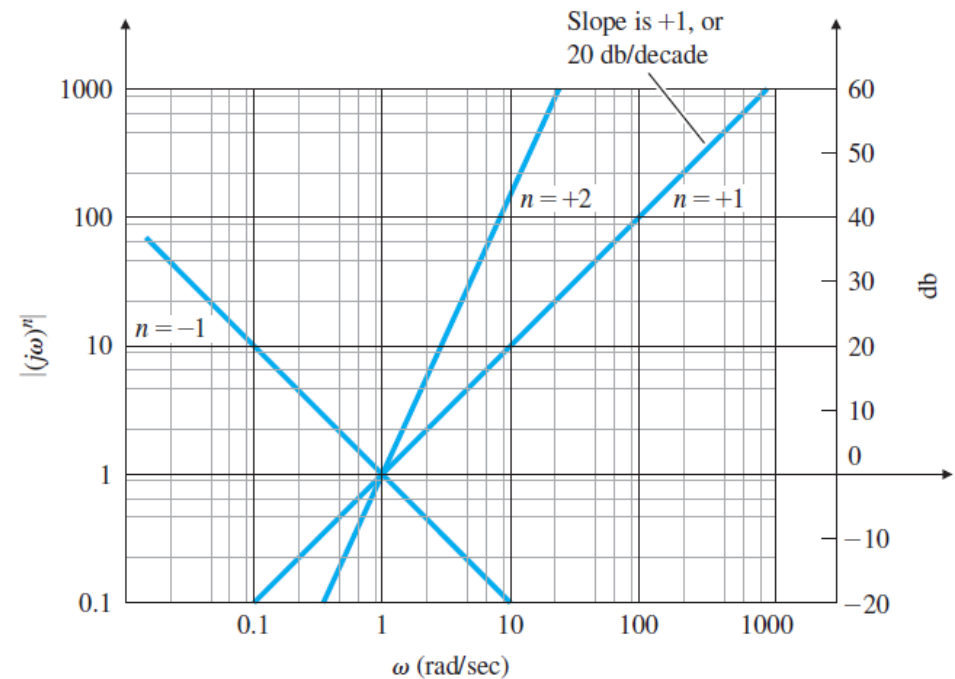
- **By Hand:**
 - Hand Writing in Exam (40%)
- Use the Bode Plot Techniques
 - to roughly sketch the magnitude and phase plots
 - of any transfer function
 - by identifying these critical frequency locations
- **By Computer:**
 - Multiple Choice in Exam (60%)
- Use Matlab codes
 - to draw the exact Bode Plot, Nyquist Plot
 - of any transfer function
- Design proper transfer function and
 - select associated and reasonable gain value

- Class 1: Singularities at the origin

$$K_0 (j\omega)^n$$

$$\begin{aligned} \log K_0 |(j\omega)^n| \\ = \log K_0 + n \log |j\omega| \end{aligned}$$

$$\angle K_0 (j\omega)^n = n \times 90^\circ$$

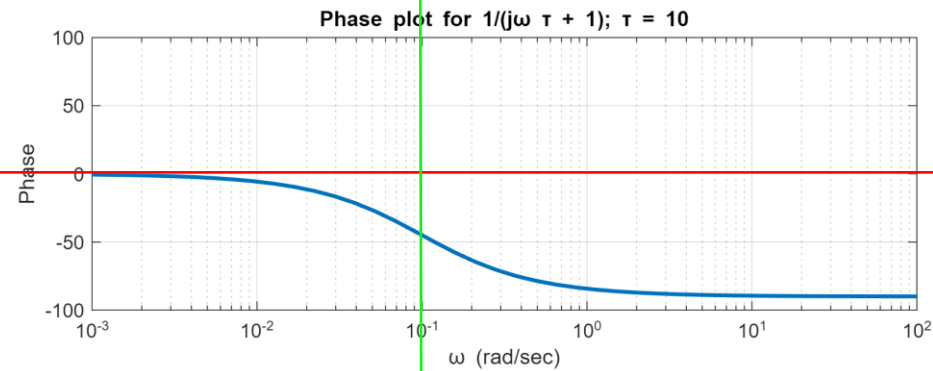
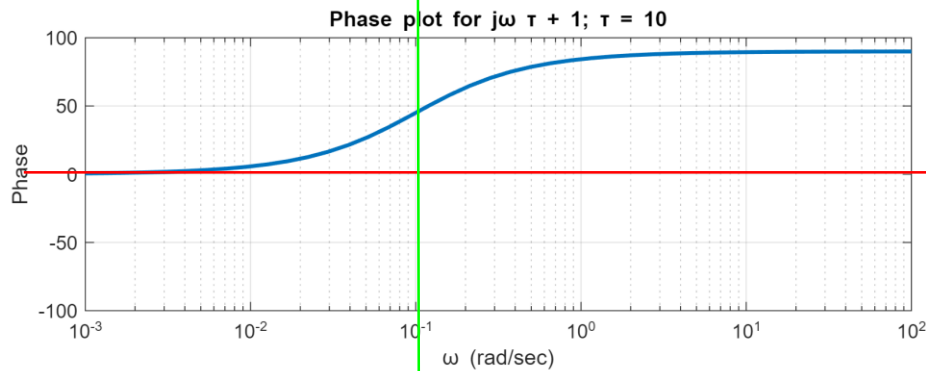
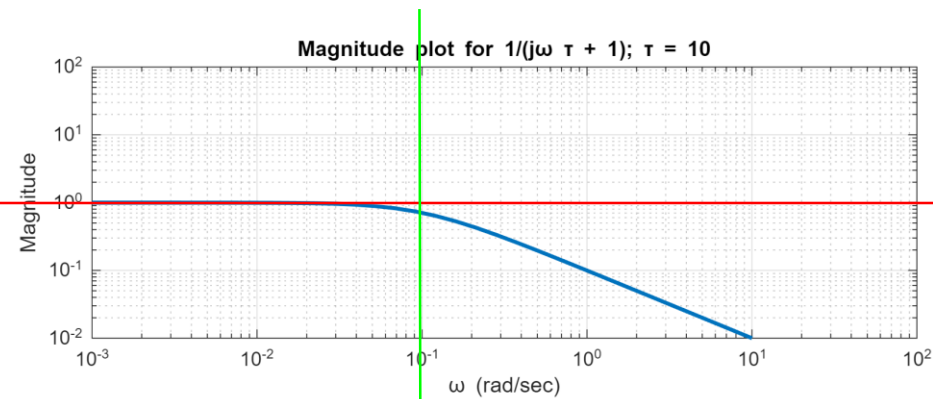
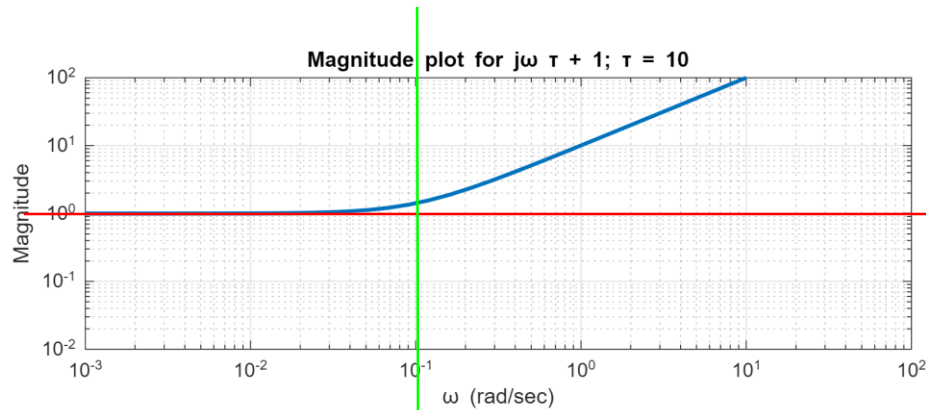


Class 2: First-order term

$$G(s) = 10s + 1$$

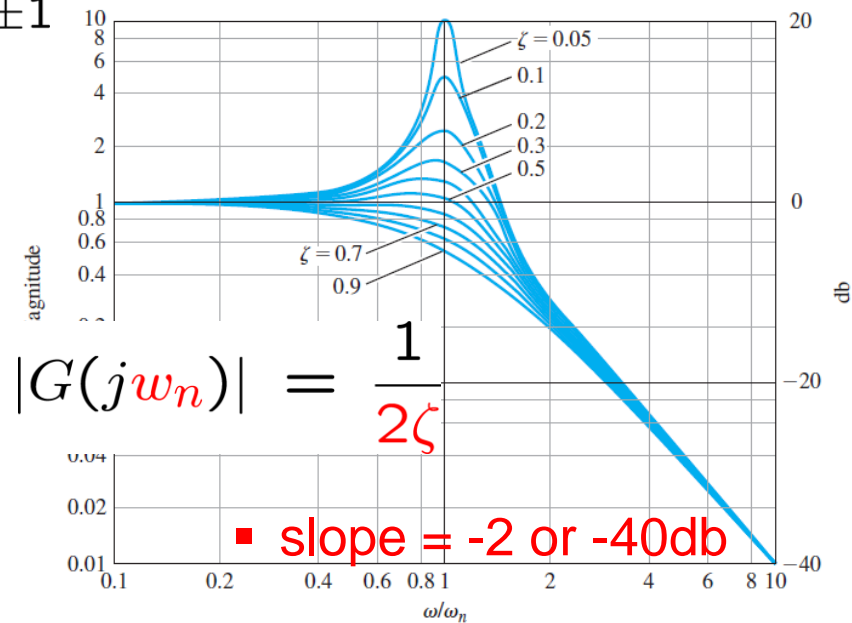
$$(j\omega\tau + 1)^{\pm 1}$$

$$G(s) = \frac{1}{10s + 1}$$



■ **Class 3:** $\left[\left(\frac{jw}{w_n} \right)^2 + 2\zeta \frac{jw}{w_n} + 1 \right]^{\pm 1}$

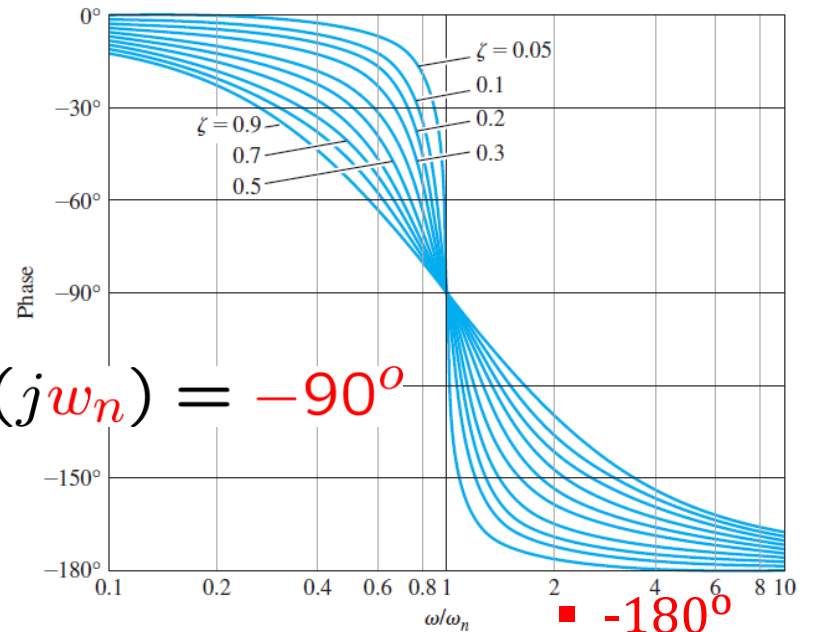
$$|G(s = jw)|$$



$$G(s) = \frac{1}{(s/w_n)^2 + 2\zeta(s/w_n) + 1}$$

$$\angle G(s = jw)$$

$$\angle G(jw_n) = -90^\circ$$

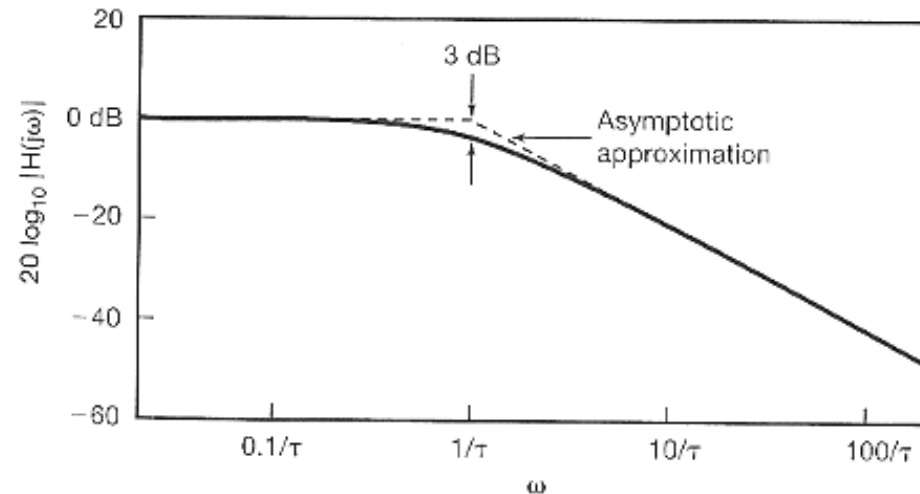


First-Order CT Systems:

$$H(j\omega) = \frac{1}{j\omega\tau + 1}$$

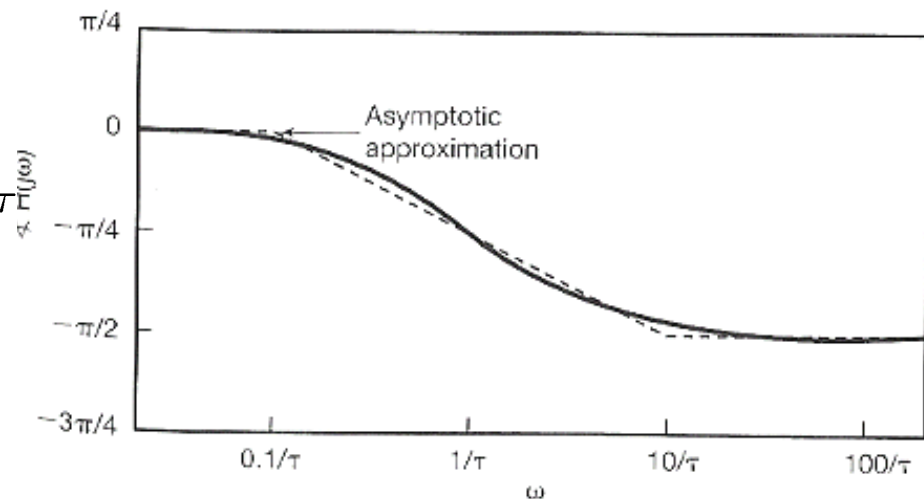
$$20 \log_{10} |H(j\omega)| =$$

$$\begin{cases} 0 & \omega \ll 1/\tau \\ -10 \log_{10}(2) \approx -3dB & \omega = 1/\tau \\ -20 \log_{10}(\omega\tau) & \omega \gg 1/\tau \\ = -20 \log_{10}(\omega) - 20 \log_{10}(\tau) \end{cases}$$



$$\angle H(j\omega) =$$

$$\begin{cases} 0 & \omega \leq 0.1/\tau \\ -(\pi/4)[\log_{10}(\omega\tau) + 1] & 0.1/\tau \leq \omega \leq 10/\tau \\ = -(\pi/4) [\log_{10}(\omega) + \log_{10}(\tau) + 1] \\ -\pi/4 & \omega = 1/\tau \\ -\pi/2 & \omega \geq 10/\tau \end{cases}$$

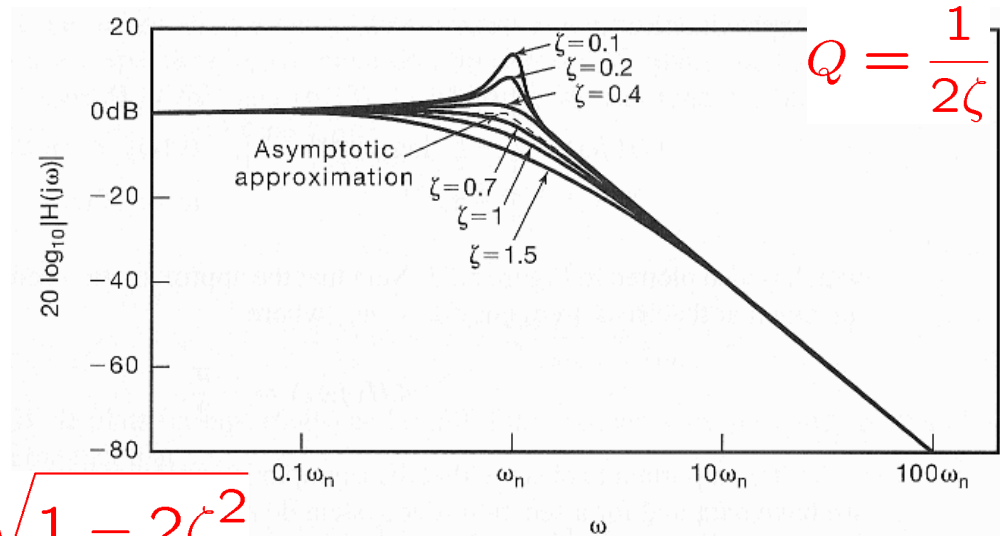


Second-Order CT Systems:

$$H(j\omega) = \frac{1}{(j\frac{\omega}{\omega_n})^2 + 2\zeta(j\frac{\omega}{\omega_n}) + 1}$$

$$20 \log_{10} |H(j\omega)| =$$

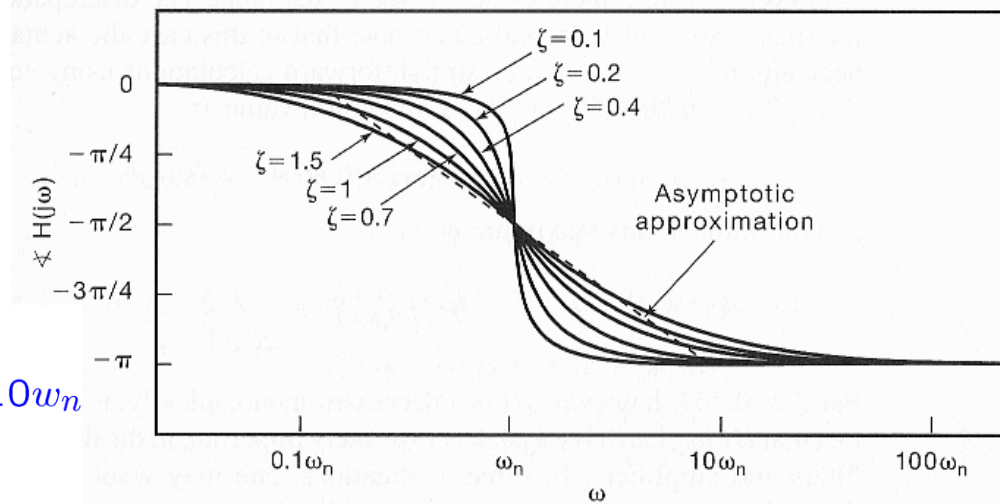
$$\begin{cases} 0 & \omega \ll \omega_n \\ -20 \log_{10}(2\zeta) & \omega = \omega_n \\ -40 \log_{10}(\omega) + 40 \log_{10}(\omega_n) & \omega \gg \omega_n \end{cases}$$



• For $\zeta < \frac{\sqrt{2}}{2}$ $\omega_{\max} = \omega_n \sqrt{1 - 2\zeta^2}$

$$\angle H(j\omega) =$$

$$\begin{cases} 0 & \omega \leq 0.1\omega_n \\ -(\pi/2)[\log_{10}(\omega/\omega_n) + 1] & 0.1\omega_n \leq \omega \leq 10\omega_n \\ -\pi/2 & \omega = \omega_n \\ -\pi & \omega \geq 10\omega_n \end{cases}$$



Examples

- Example 6.3: Bode Plot for Real Poles and Zeros

$$K G(s) = \frac{2000 (s + 0.5)}{s (s + 10) (s + 50)}$$

- (1) Break points

$$K G(jw) = \frac{2 \left[\frac{jw}{0.5} + 1 \right]}{(jw) \left[\frac{jw}{10} + 1 \right] \left[\frac{jw}{50} + 1 \right]}$$

- Break points: 0.5, 10, 50

- (2) Asymptotes

- Low-Frequency Asymptote: $K G(jw) = \frac{2}{(jw)}$ for $w < 0.1$

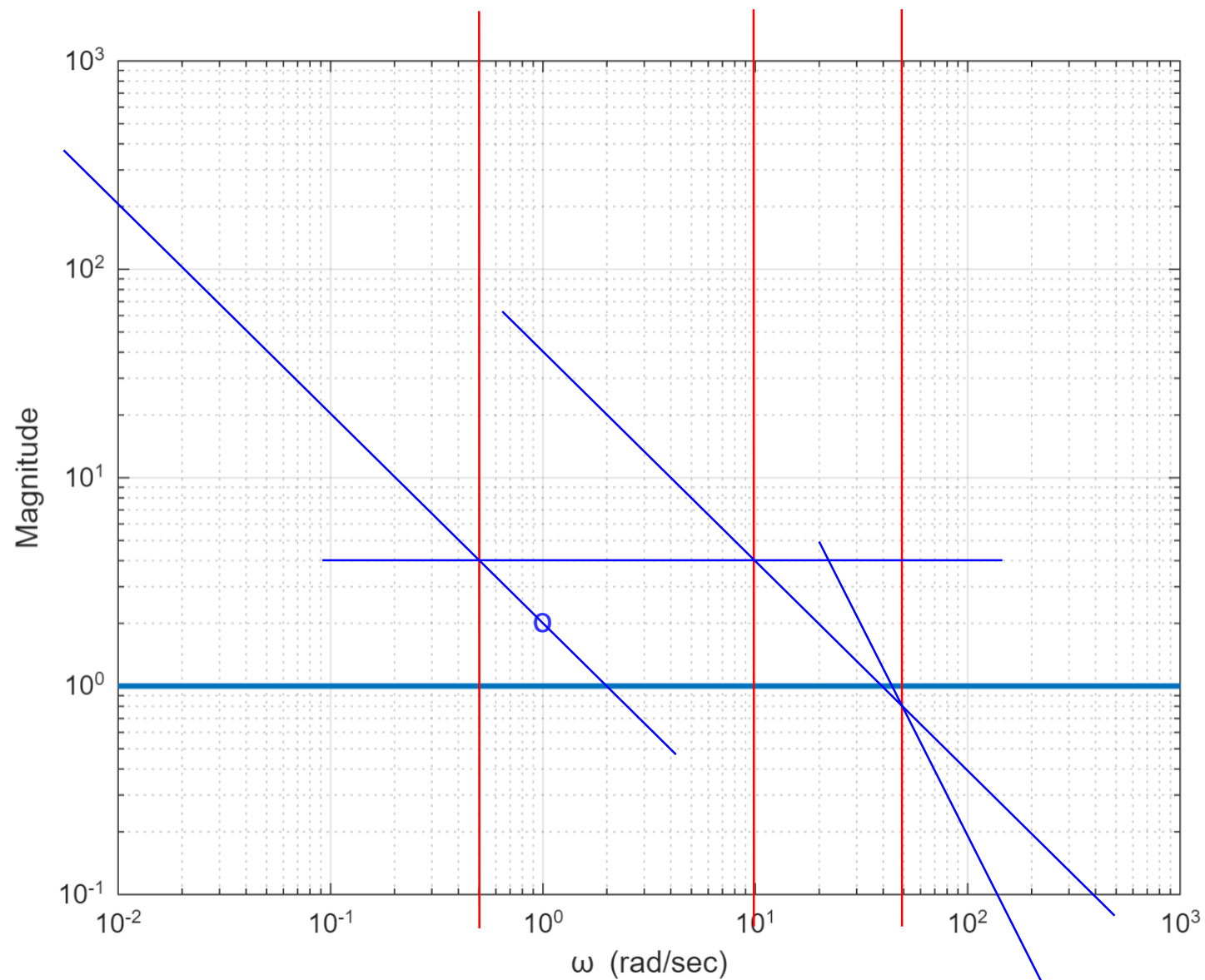
- $\omega \ll 0.5$: slope = -1 (or -20 db/decade)

- $0.5 < \omega < 10$: slope = 0 (or 0 db/decade)

- $10 < \omega < 50$: slope = -1 (or -20 db/decade)

- $50 < \omega$: slope = -2 (or -40 db/decade)

- Example 6.3: Bode Plot for Real Poles and Zeros

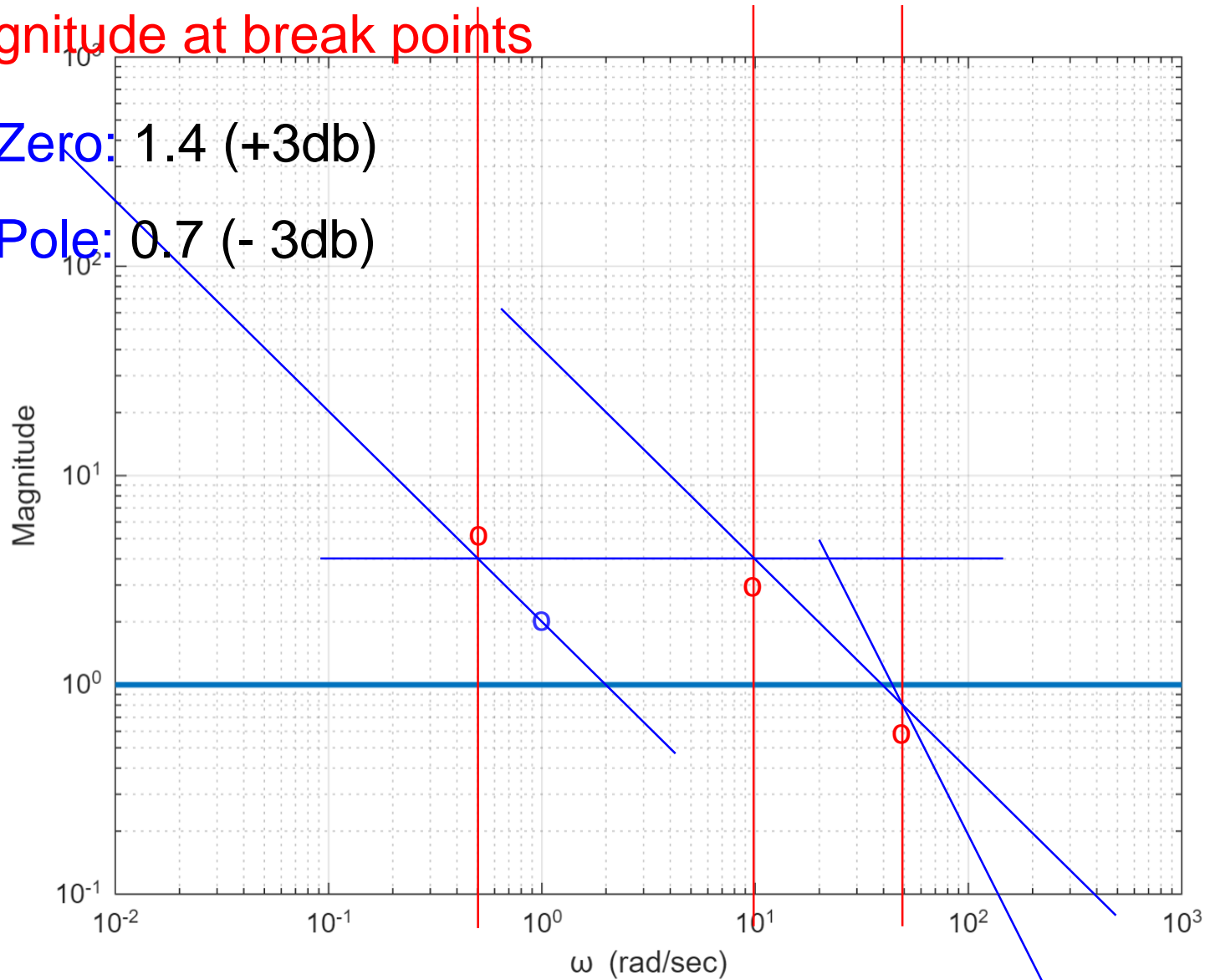


Examples

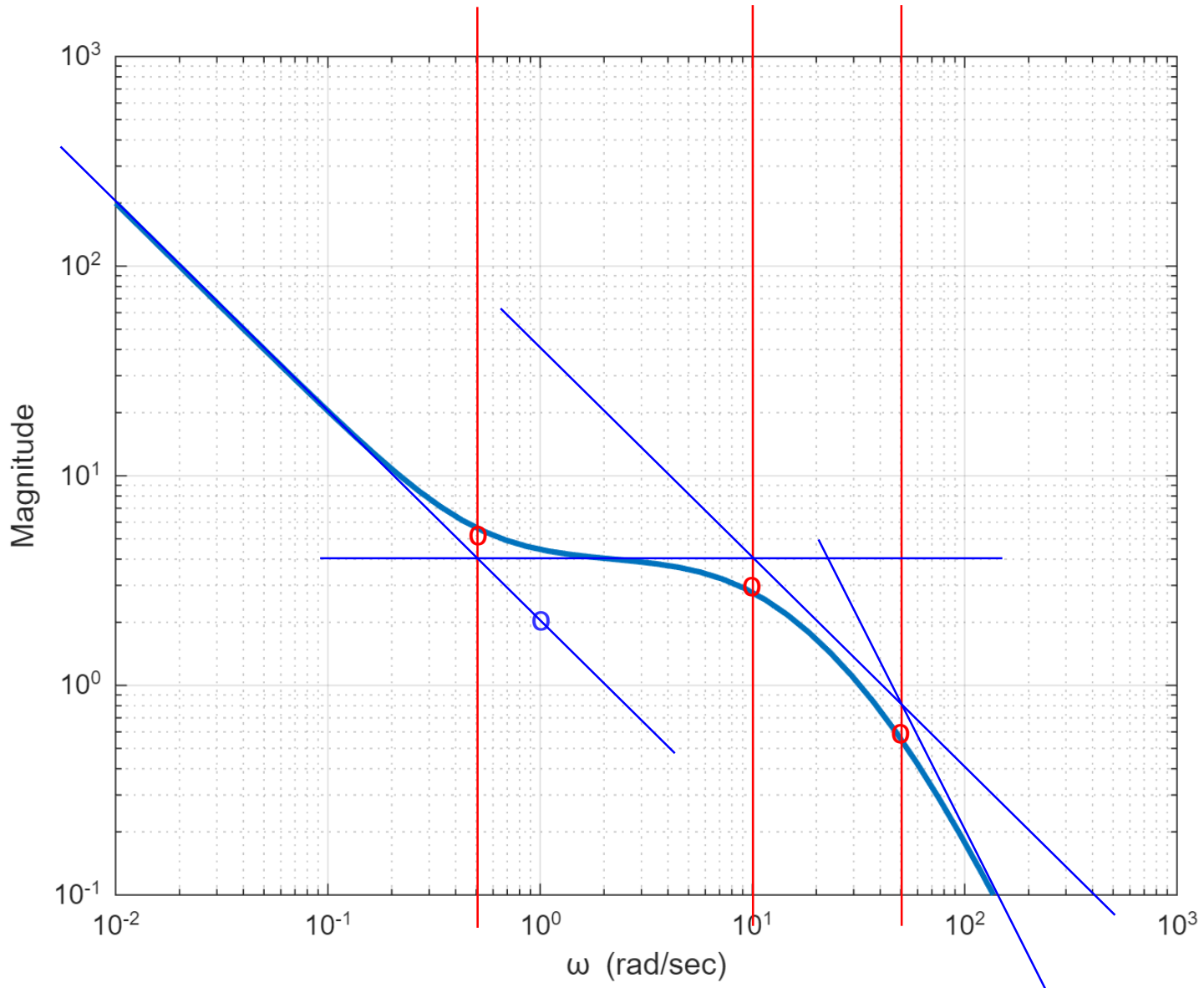
Example 6.3: Bode Plot for Real Poles and Zeros

(3) Magnitude at break points

- By Zero: 1.4 (+3db)
- By Pole: 0.7 (- 3db)



■ Example 6.3: Bode Plot for Real Poles and Zeros



Examples

- Example 6.3: Bode Plot for Real Poles and Zeros

$$K G(s) = \frac{2000 (s + 0.5)}{s (s + 10) (s + 50)}$$

- (1) Break points

$$K G(j\omega) = \frac{2 \left[\frac{j\omega}{0.5} + 1 \right]}{(j\omega) \left[\frac{j\omega}{10} + 1 \right] \left[\frac{j\omega}{50} + 1 \right]}$$

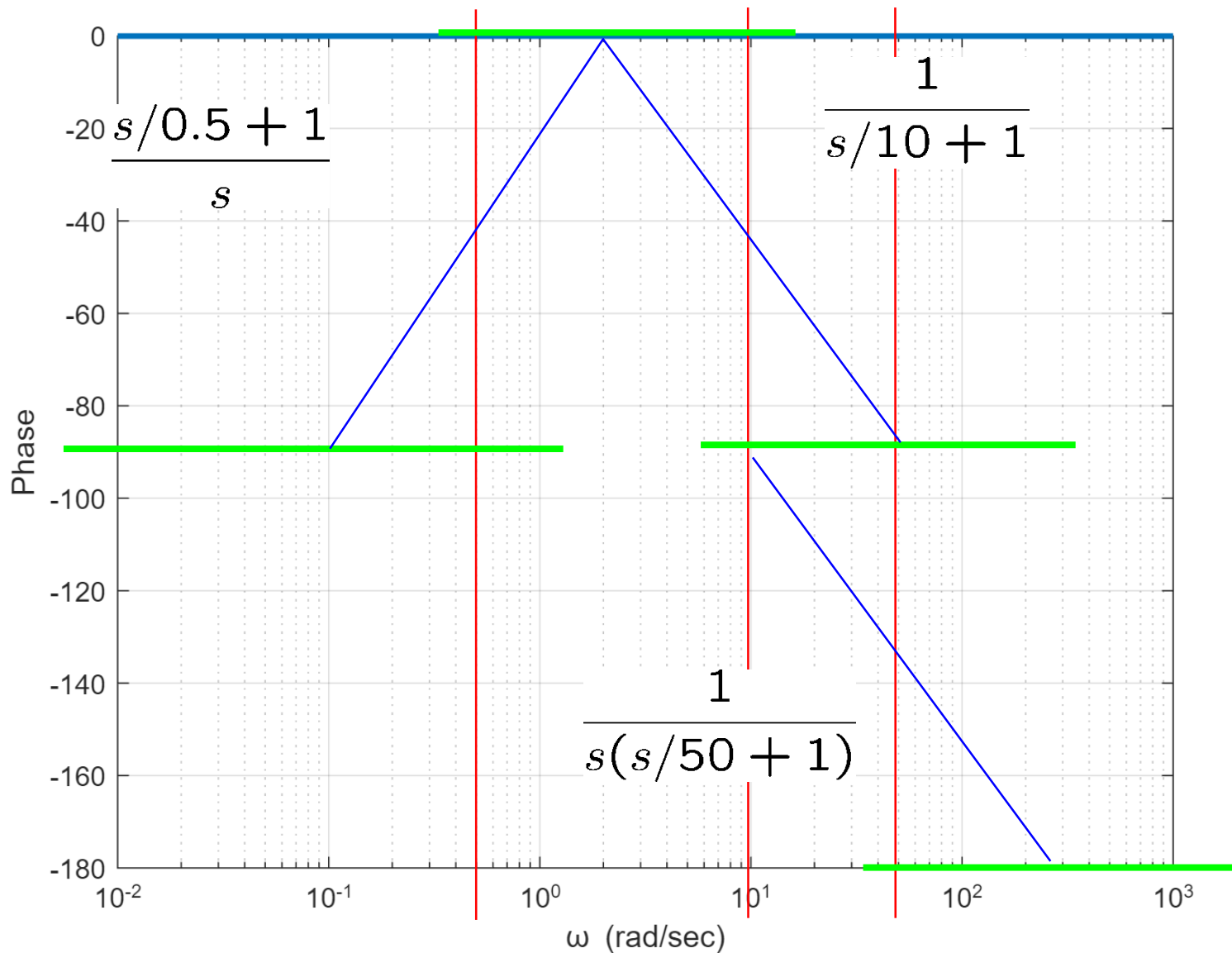
- Break points: 0.5, 10, 50

- (4) Phase

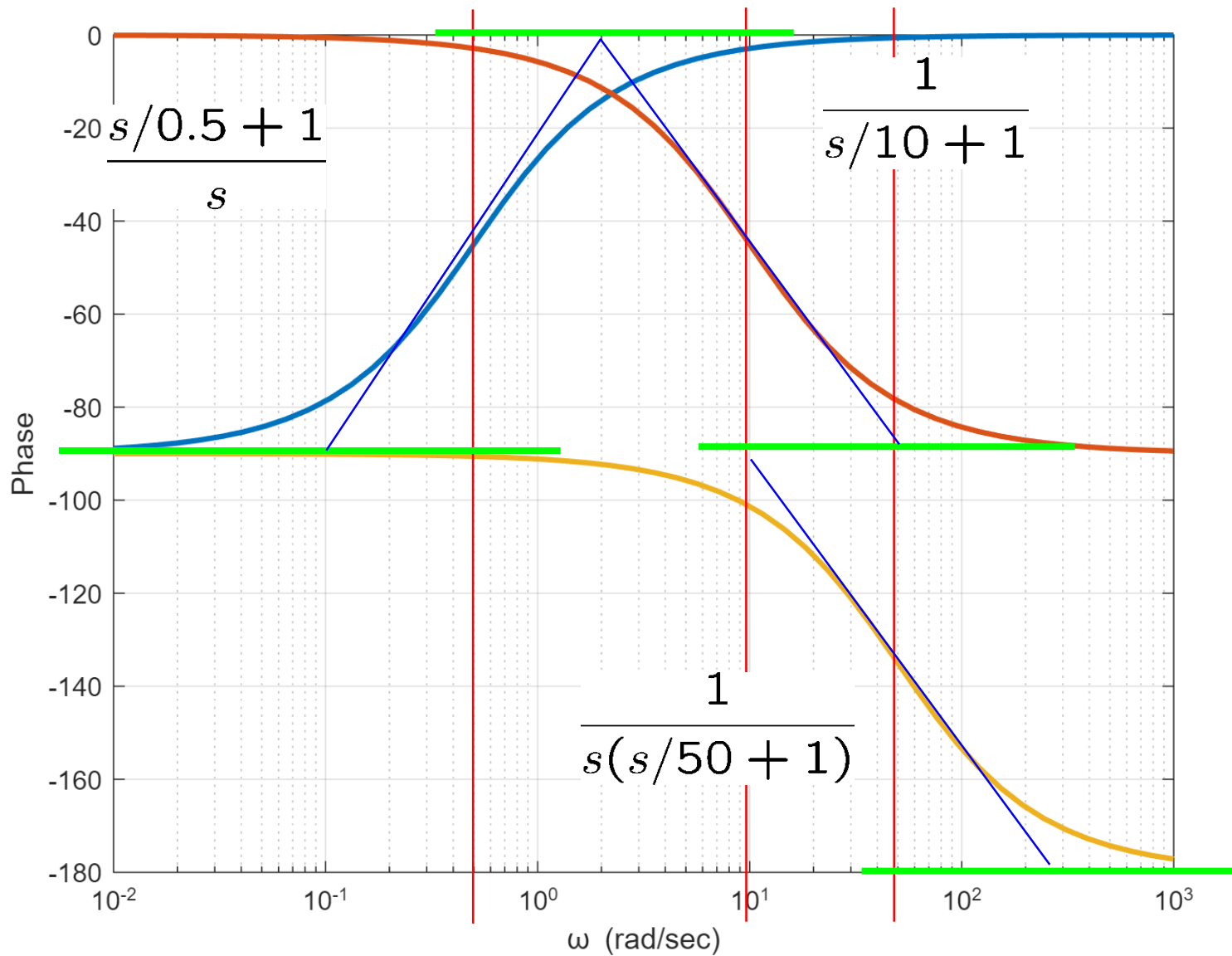
- Low-Frequency Asymptote: $K G(j\omega) = \frac{2}{(j\omega)}$ for $\omega < 0.1$

- $\omega \ll 0.5$: phase = -90°
- $0.5 < \omega < 10$: phase = 0°
- $10 < \omega < 50$: phase = -90°
- $50 < \omega$: phase = -180°

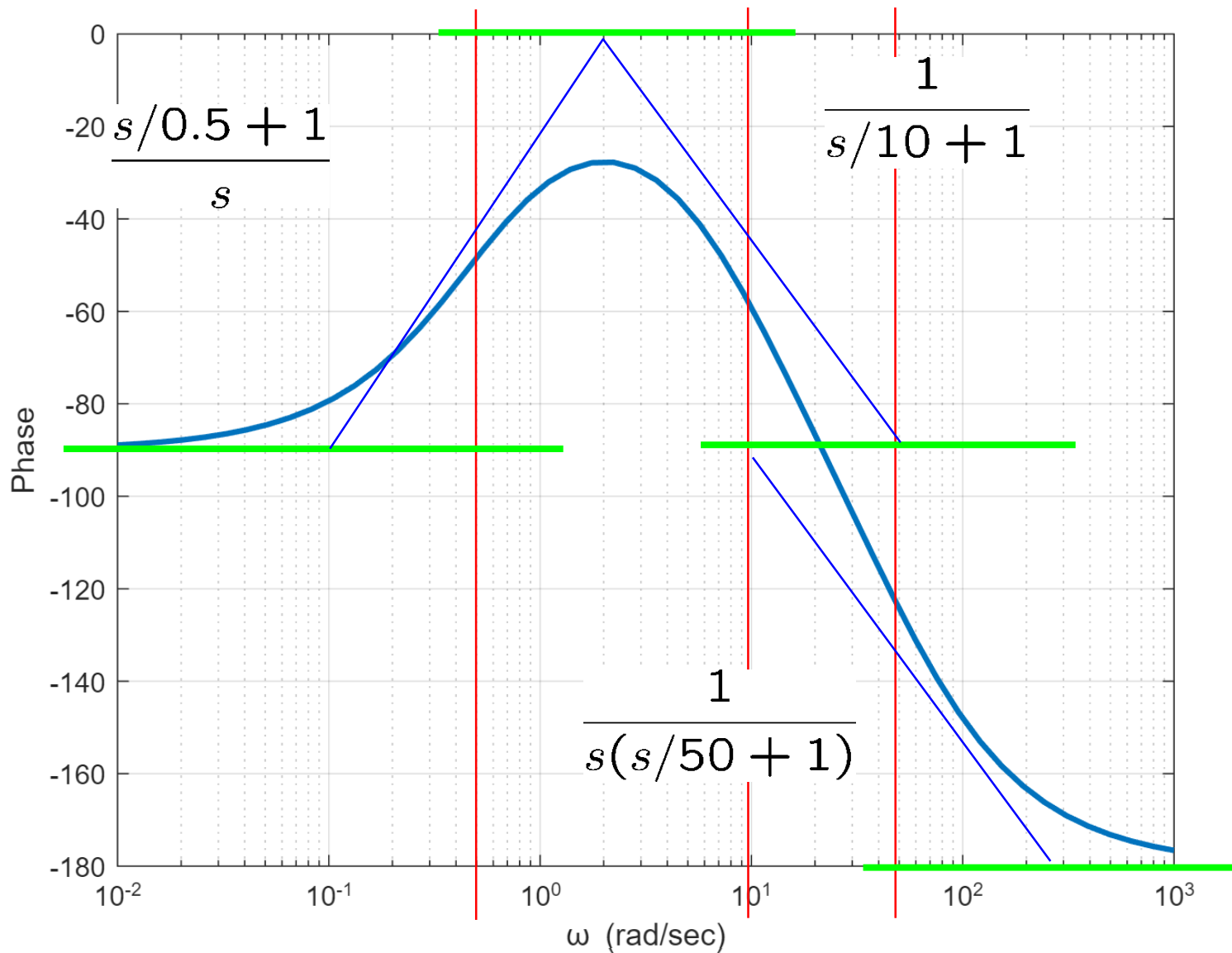
Example 6.3: Bode Plot for Real Poles and Zeros



■ Example 6.3: Bode Plot for Real Poles and Zeros



■ Example 6.3: Bode Plot for Real Poles and Zeros



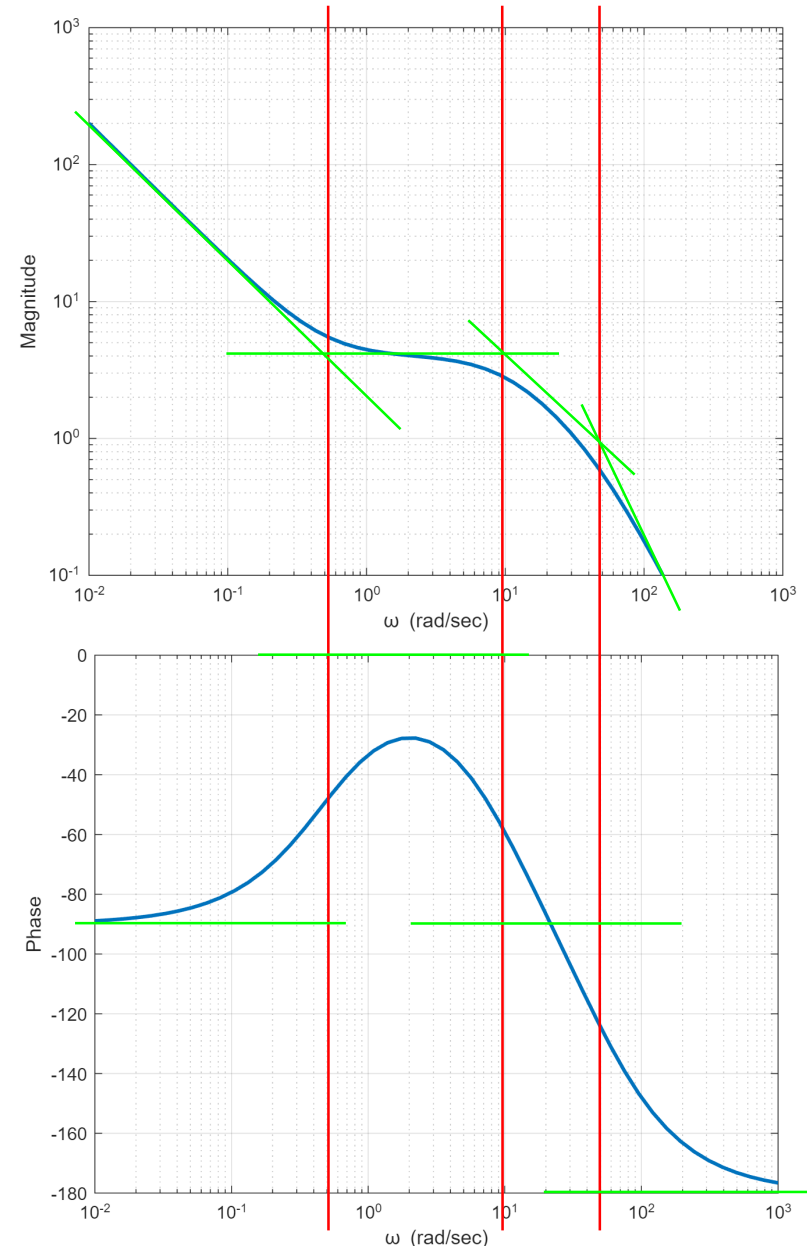
Examples

Example 6.3: Bode Plot for Real Poles and Zeros

$$\frac{2000 (s + 0.5)}{s (s + 10) (s + 50)}$$

$$\frac{2 \left[\frac{j\omega}{0.5} + 1 \right]}{(j\omega) \left[\frac{j\omega}{10} + 1 \right] \left[\frac{j\omega}{50} + 1 \right]}$$

- Break points: 0.5, 10, 50



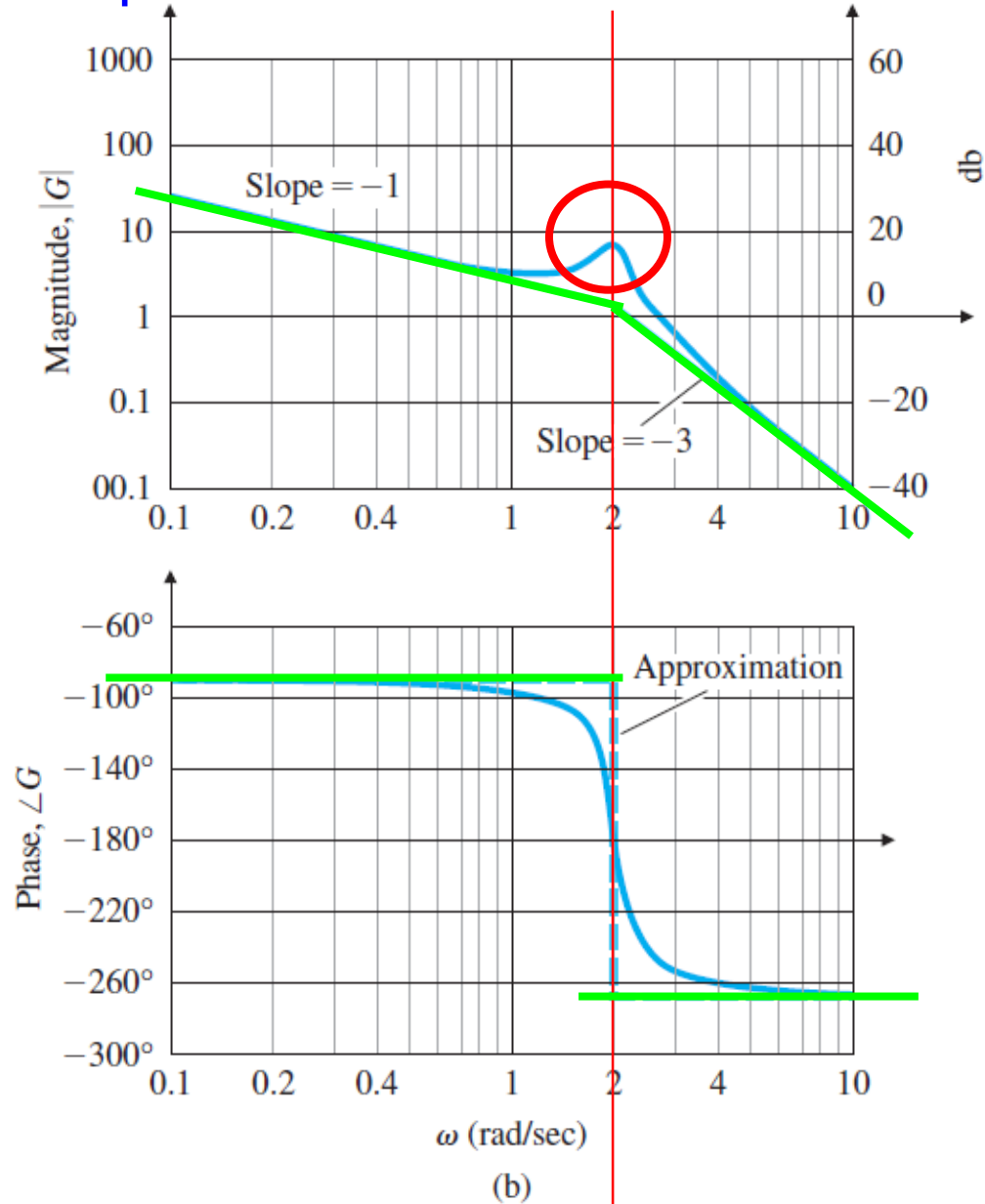
Examples

Example 6.4: Bode Plot for Complex Poles

$$K G(s) = \frac{10}{s [s^2 + 0.4s + 4]}$$

$$= \frac{10}{4} \frac{1}{s \left[\frac{s^2}{4} + 2(0.1)\frac{s}{2} + 1 \right]}$$

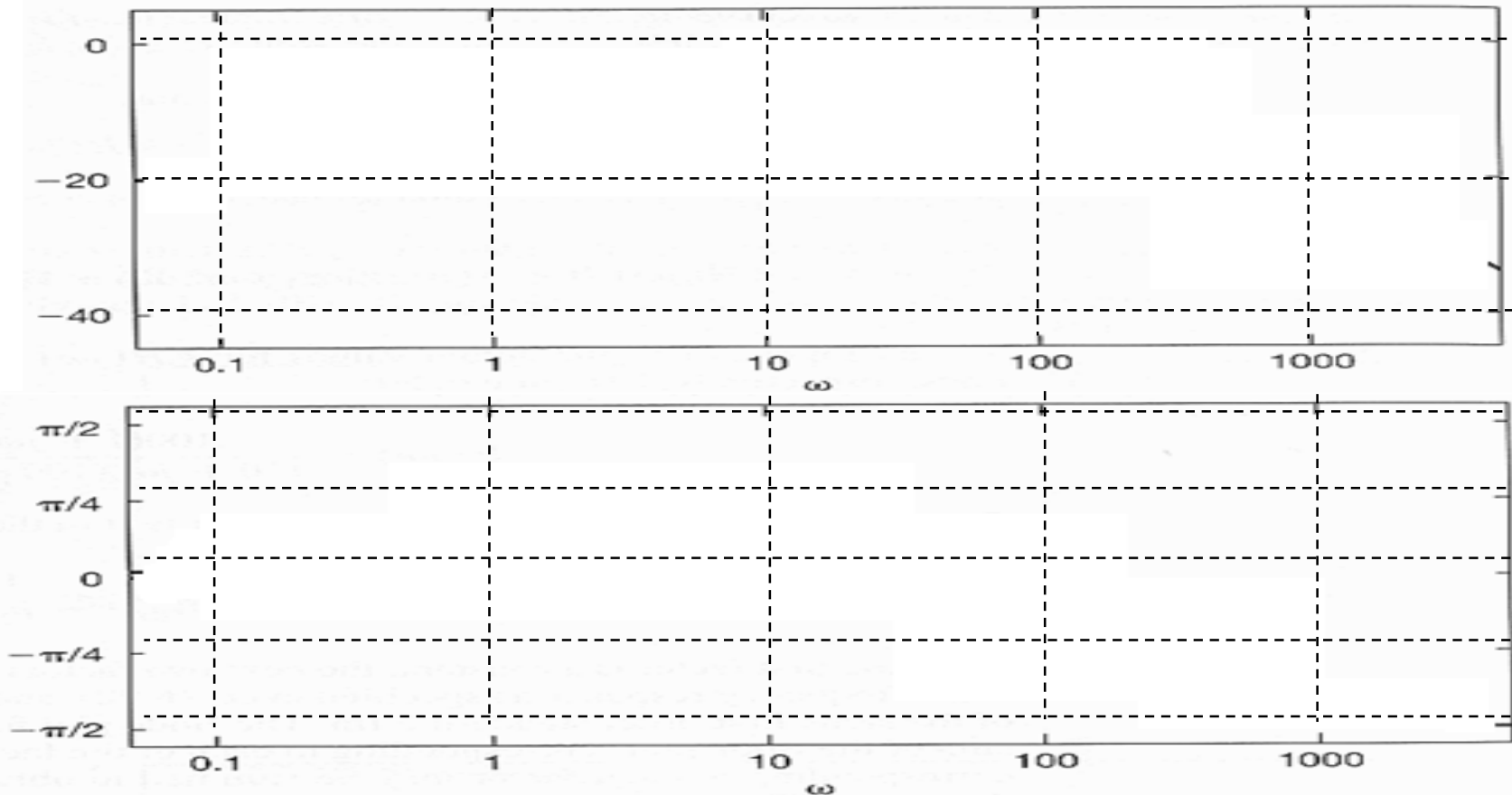
- Break points: 2



Examples

■ Example 6.5:

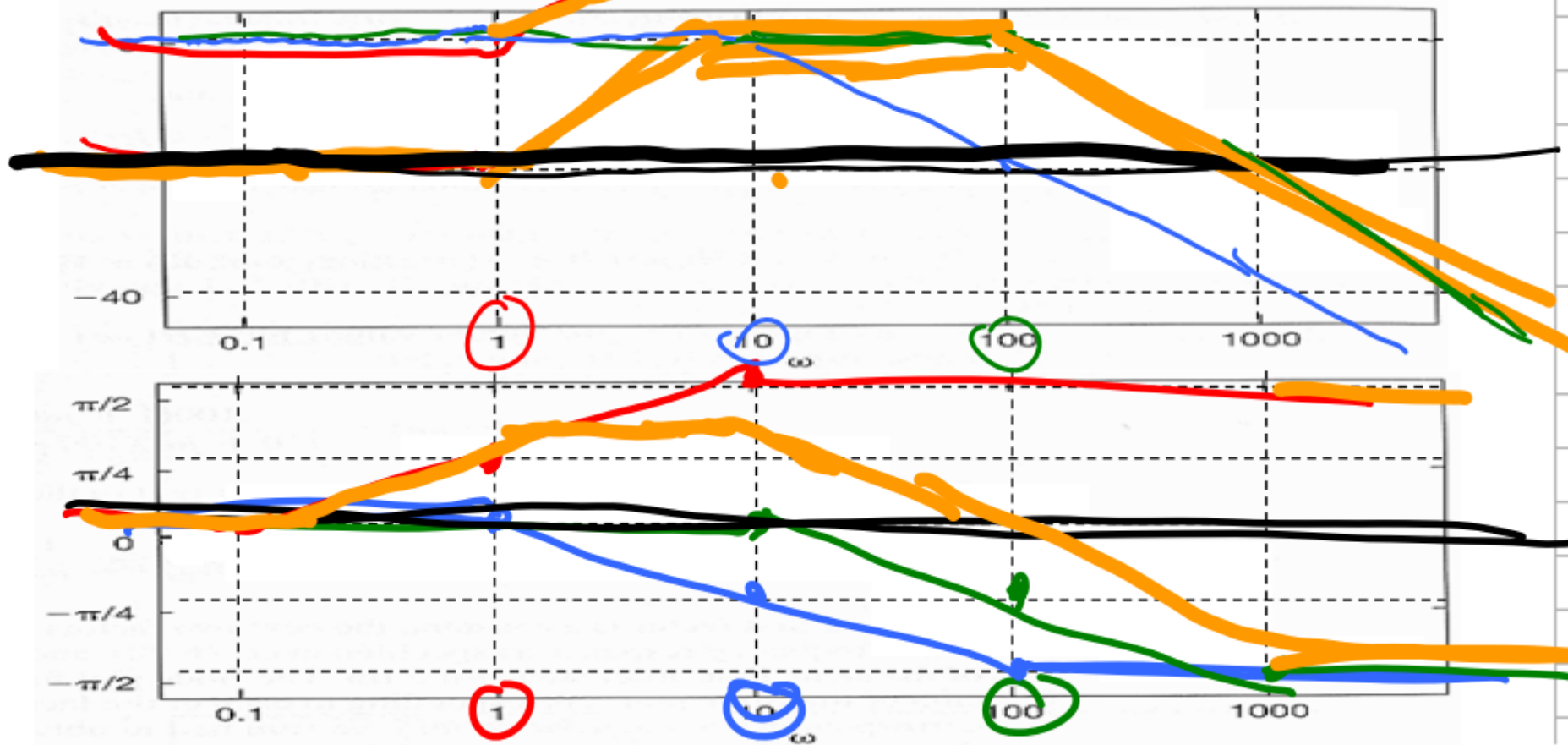
$$H(j\omega) = \frac{100(1 + j\omega)}{(10 + j\omega)(100 + j\omega)}$$
$$= \left(\frac{1}{10}\right) (1 + j\omega) \left(\frac{1}{1 + j\frac{\omega}{10}}\right) \left(\frac{1}{1 + j\frac{\omega}{100}}\right)$$

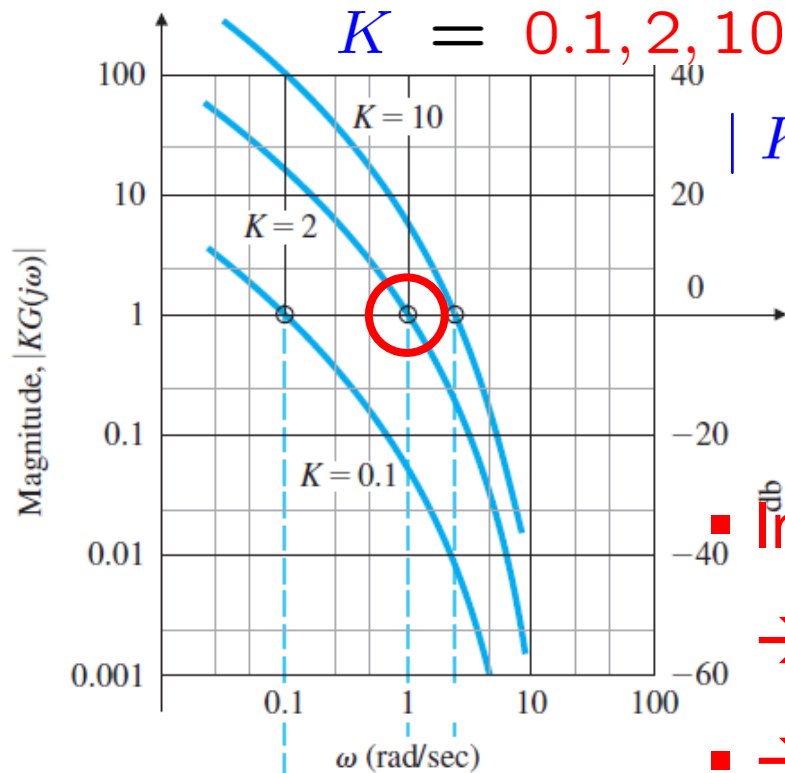


■ Example 6.5:

$$H(j\omega) = \frac{100(1 + j\omega)}{(10 + j\omega)(100 + j\omega)}$$

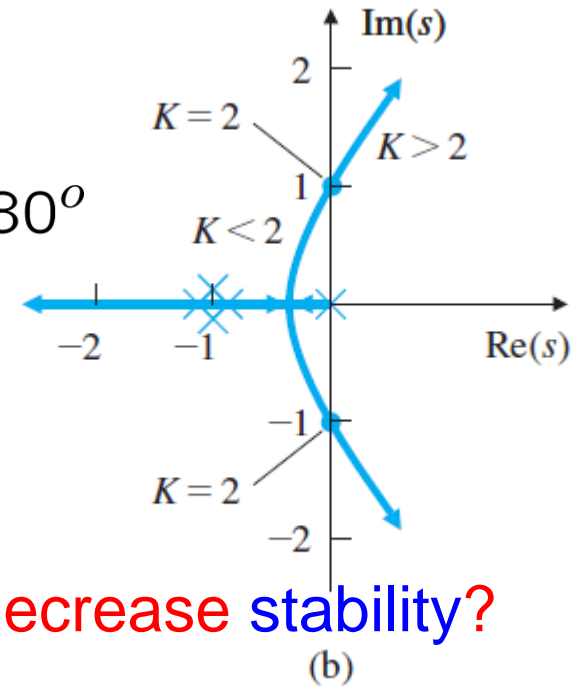
$$= \left(\frac{1}{10}\right) (1 + j\omega) \left(\frac{1}{1 + j\frac{\omega}{10}}\right) \left(\frac{1}{1 + j\frac{\omega}{100}}\right)$$





$$|KG(j\omega)| = 1$$

$$\angle G(j\omega) = 180^\circ$$



- Increasing gain
- increase OR decrease stability?

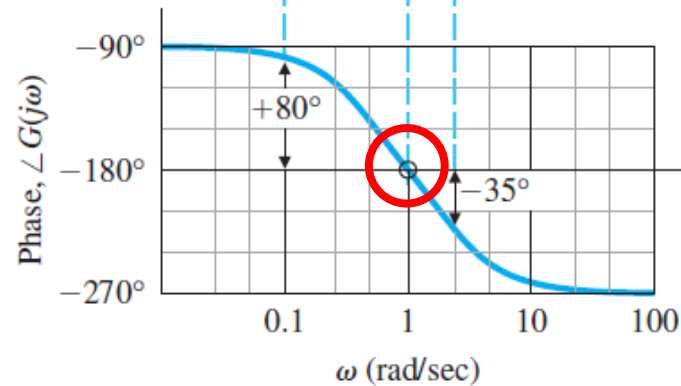
- $|KG(j\omega)| < 1$!!!

- However, for some systems,

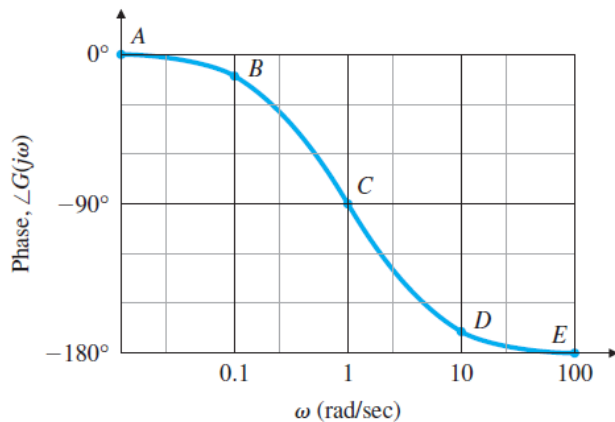
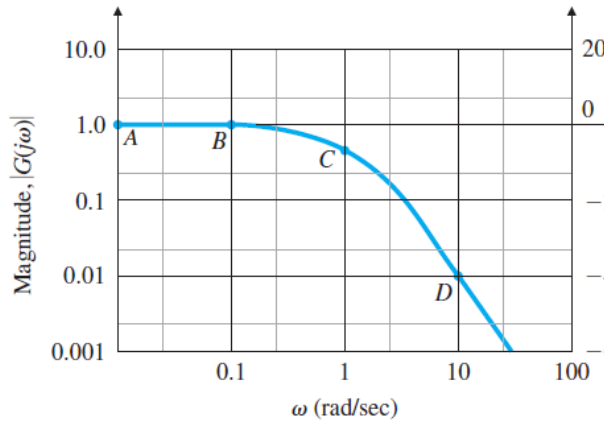
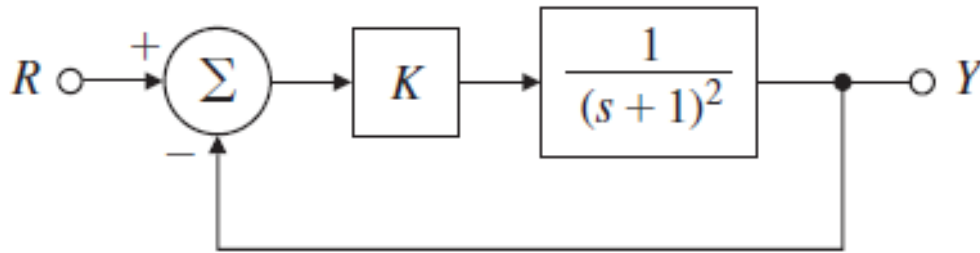
- $|KG(j\omega)| > 1$!!!

- Need to check their root locus

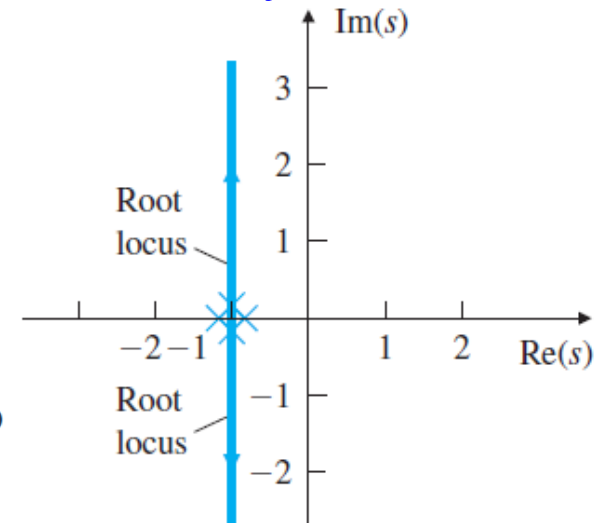
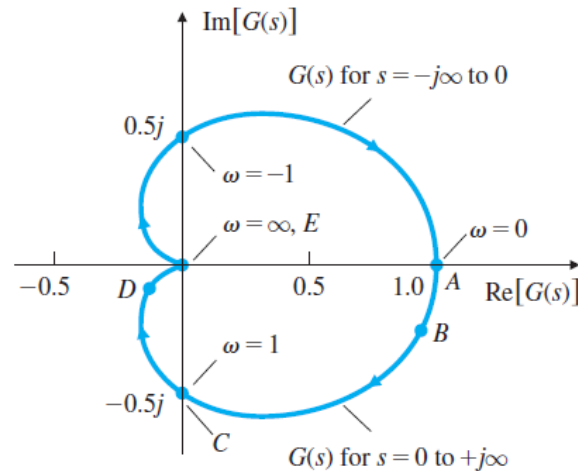
- OR, by Nyquist Stability Criterion



Example 6.8: Nyquist Plot for a Second-Order System

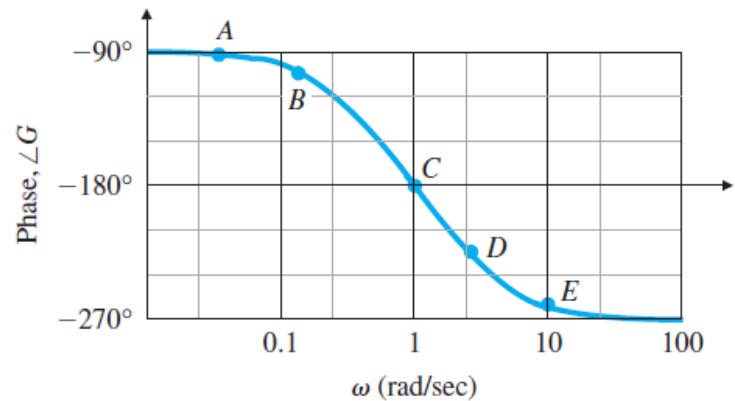
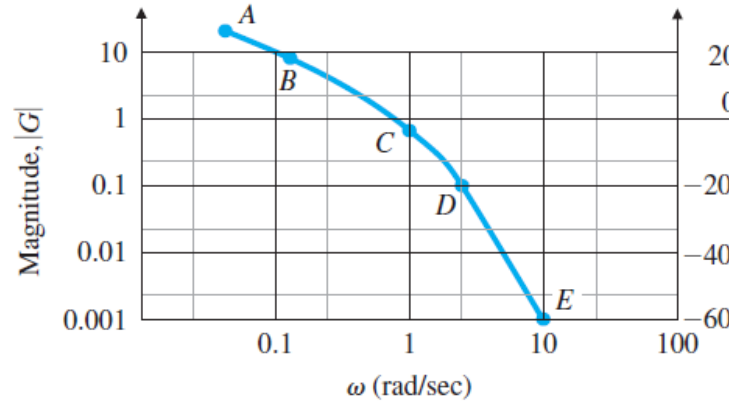
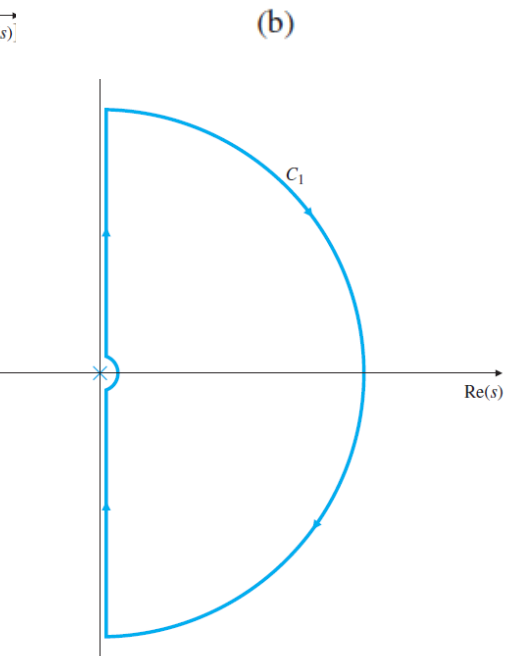
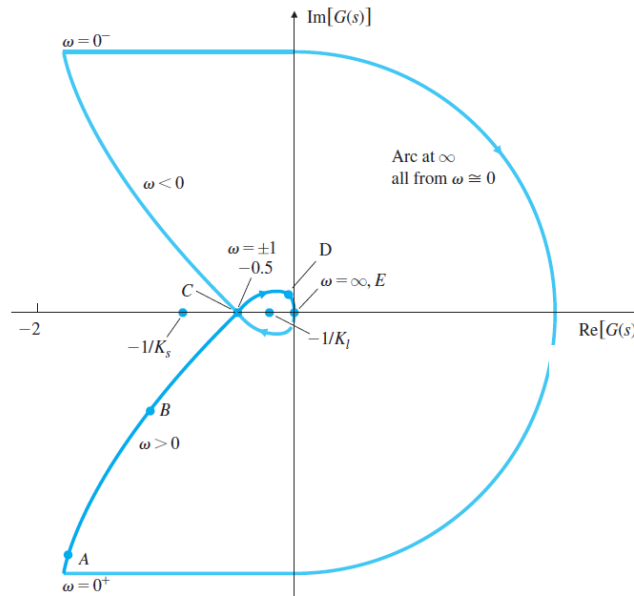
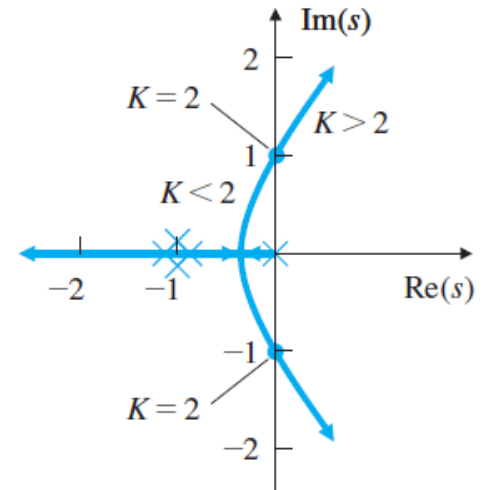
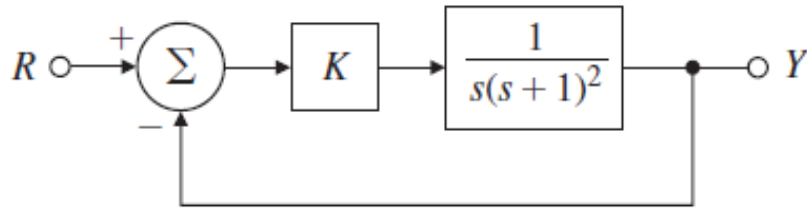


(b)



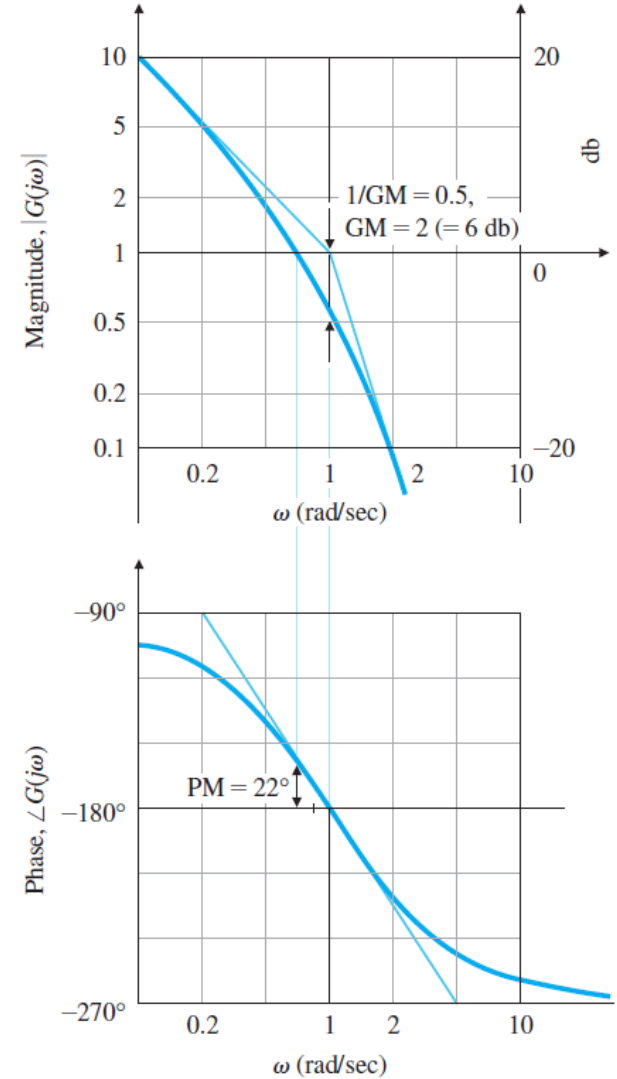
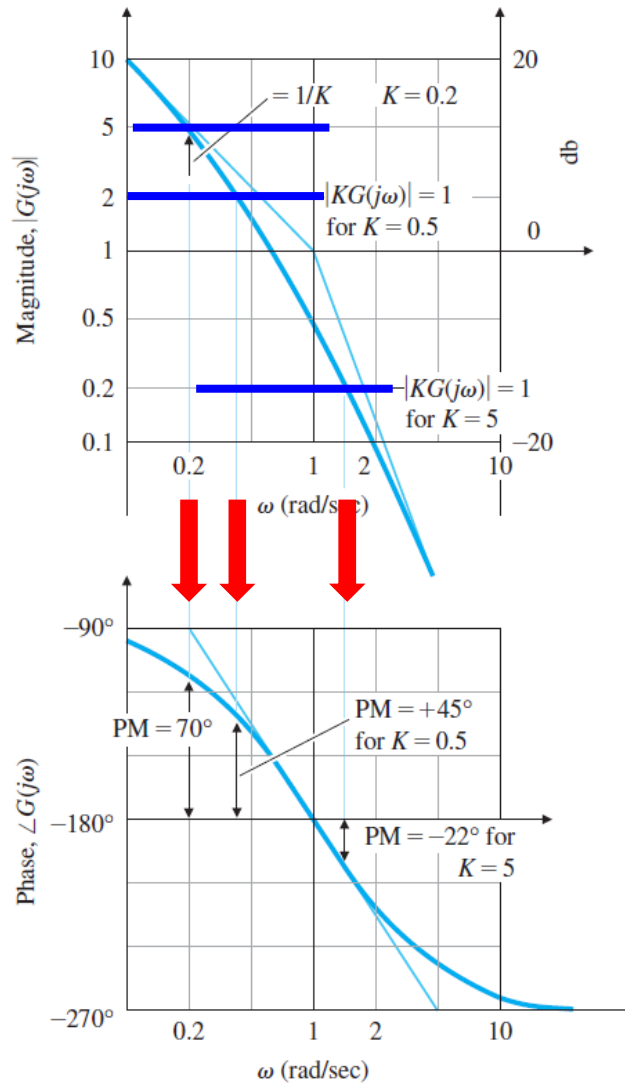
- $N = 0$: not encircle -1
- $P = 0$: no poles of $G(s)$ in RHP
- $Z = N + P \rightarrow Z = 0$,
no unstable roots for $K = 1$
- $K > 0$ also holds

Example 6.9: Nyquist Plot for a Third-Order System



(b)

- PM vs K
- $K = 5$
- $|KG(j\omega)| = 1$
- $PM = -22^\circ$
- $K = 0.5$
- $|KG(j\omega)| = 1$
- $PM = +45^\circ$
- $K = 0.2$
- $|KG(j\omega)| = 1$
- $PM = +70^\circ$

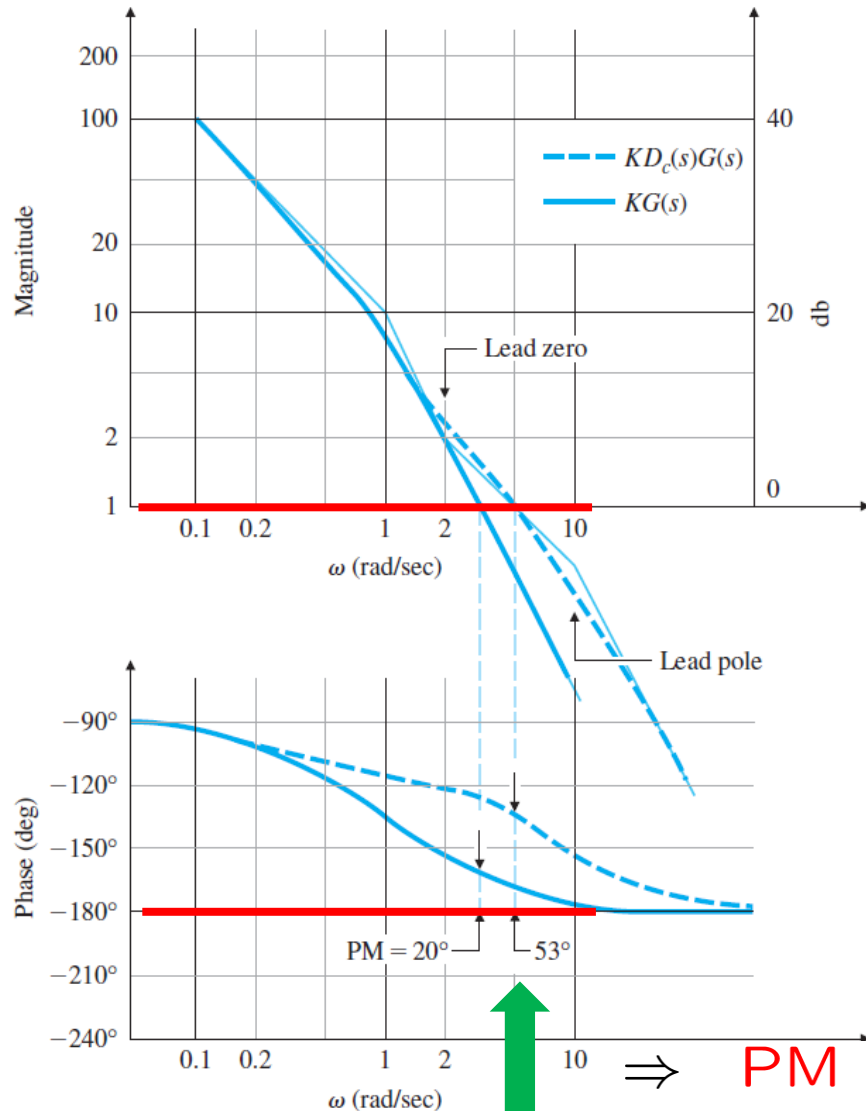


Examples

Example 6.15: Lead Compensation for a DC Motor

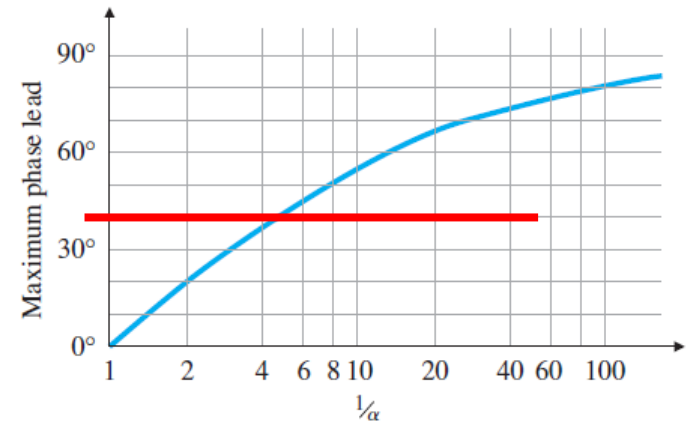
$$K G(s)$$

$$K D_c(s) G(s)$$



\Rightarrow PM $\geq 25^\circ$, at $w_c = 3$

\Rightarrow Phase lead = 40° ,



$$\Rightarrow \frac{1}{\alpha} = 5$$

a zero at $w = 2$ rad/sec

a pole at $w = 10$ rad/sec

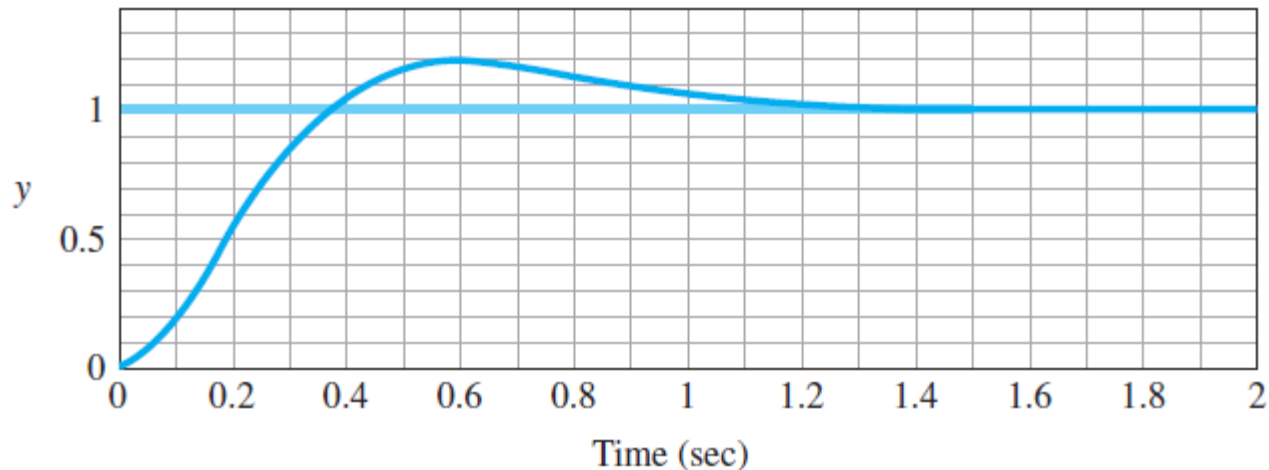
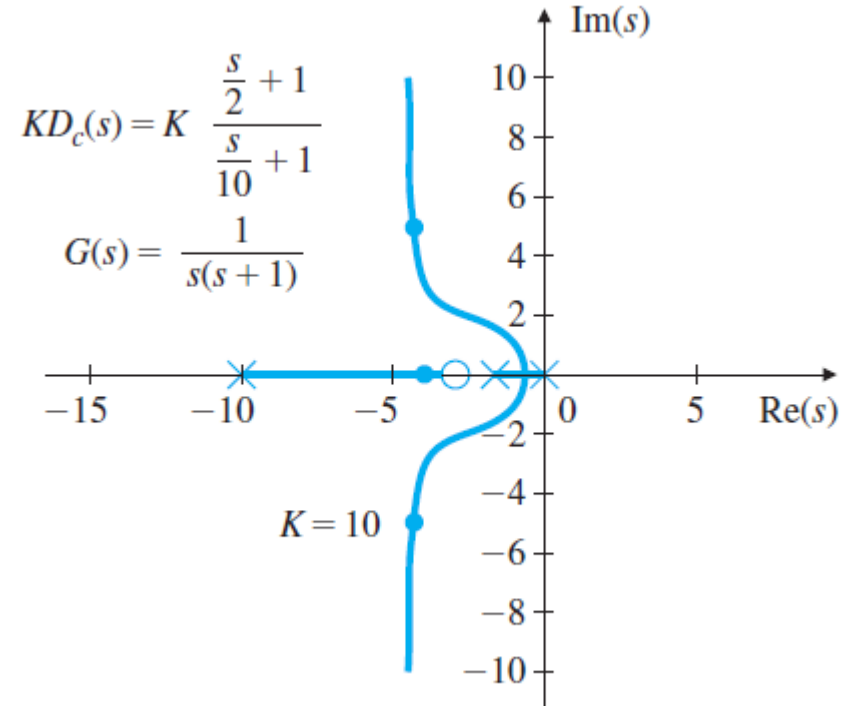
$$K D_c(s) = 10 \frac{\frac{s}{2} + 1}{\frac{s}{10} + 1}$$

\Rightarrow PM = 53° , at $w_c = 5$

Examples

Example 6.15: Lead Compensation for a DC Motor

$$K D_c(s) = 10 \frac{\frac{s}{2} + 1}{\frac{s}{10} + 1}$$



Examples

Example 6.17: Lead-Compensation Design for Type 1 Servomechanism System

$$K G(s) = K \frac{10}{s \left(\frac{s}{2.5} + 1\right) \left(\frac{s}{6} + 1\right)}$$

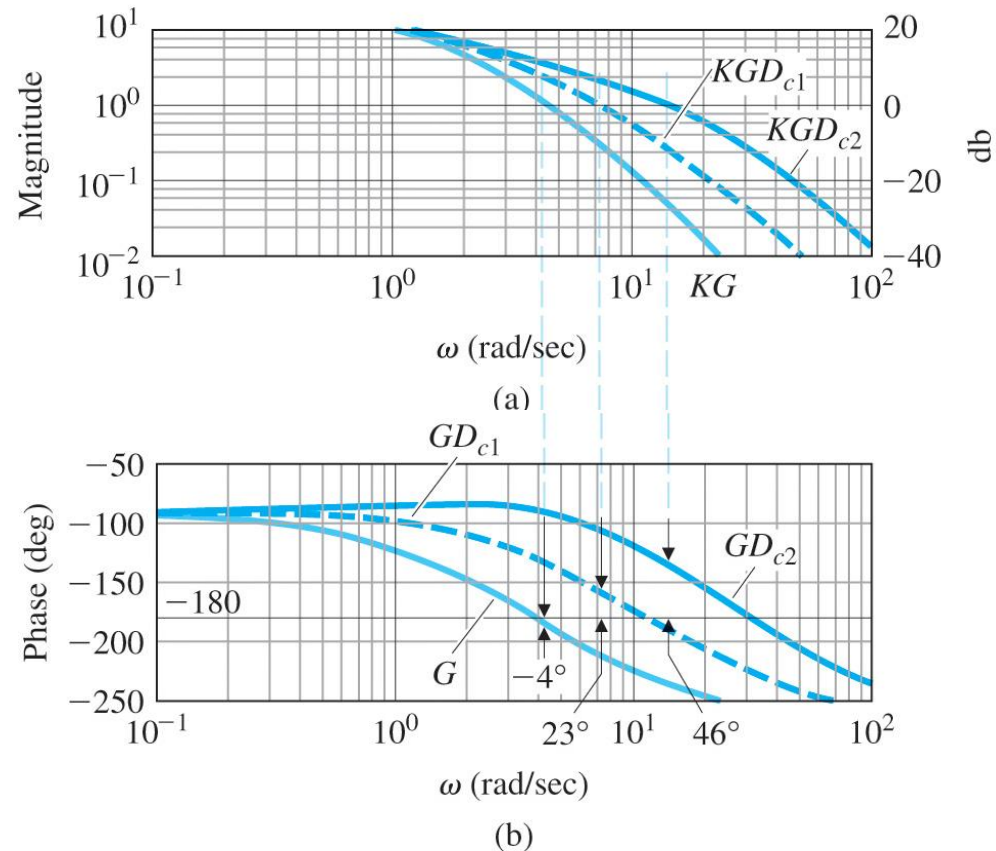
- $K_v = 10$
- $PM = 45^\circ$

1. Determine gain K:

$$\begin{aligned} K_v &= \lim_{s \rightarrow 0} s K G(s) \\ &= K \times 10 = 10 \\ \Rightarrow K &= 1 \end{aligned}$$

2. Bode plot of $KG(s)$, $K = 1$

$$\rightarrow PM \approx -4, W_{cp} \approx 4$$



Examples

Example 6.17: Lead-Compensation Design for Type 1 Servomechanism System

3. Allow for 5° of extra margin

$$\rightarrow 45^\circ + 5^\circ - (-4^\circ) = 54^\circ$$

4. Pick $\alpha \rightarrow 1/\alpha = 10$

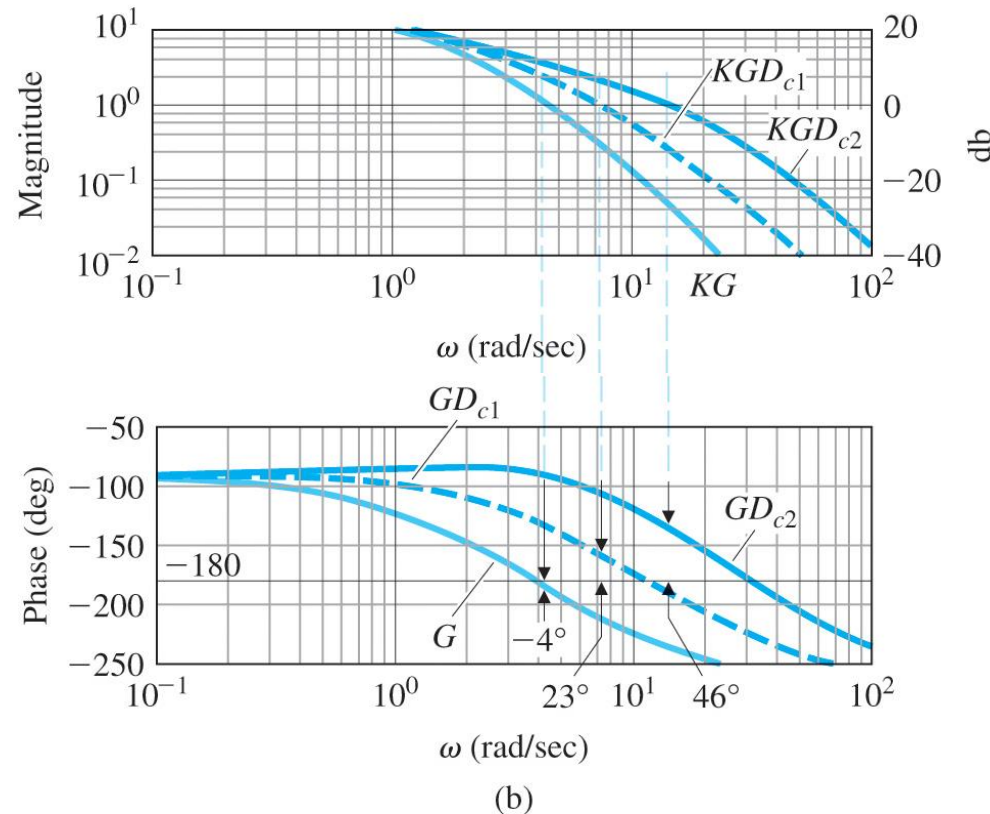
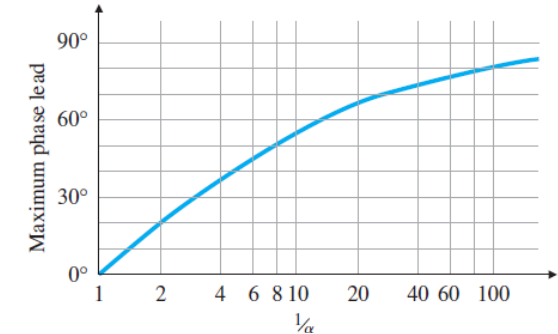
5. Zero & Pole

a zero at 2

a pole at 20

$$D_1(s) = \frac{\left(\frac{s}{2} + 1\right)}{\left(\frac{s}{20} + 1\right)}$$

$$= \frac{1}{0.1} \left(\frac{s + 2}{s + 20}\right)$$



\rightarrow PM ≈ 23 , $W_{cp} \approx 7$

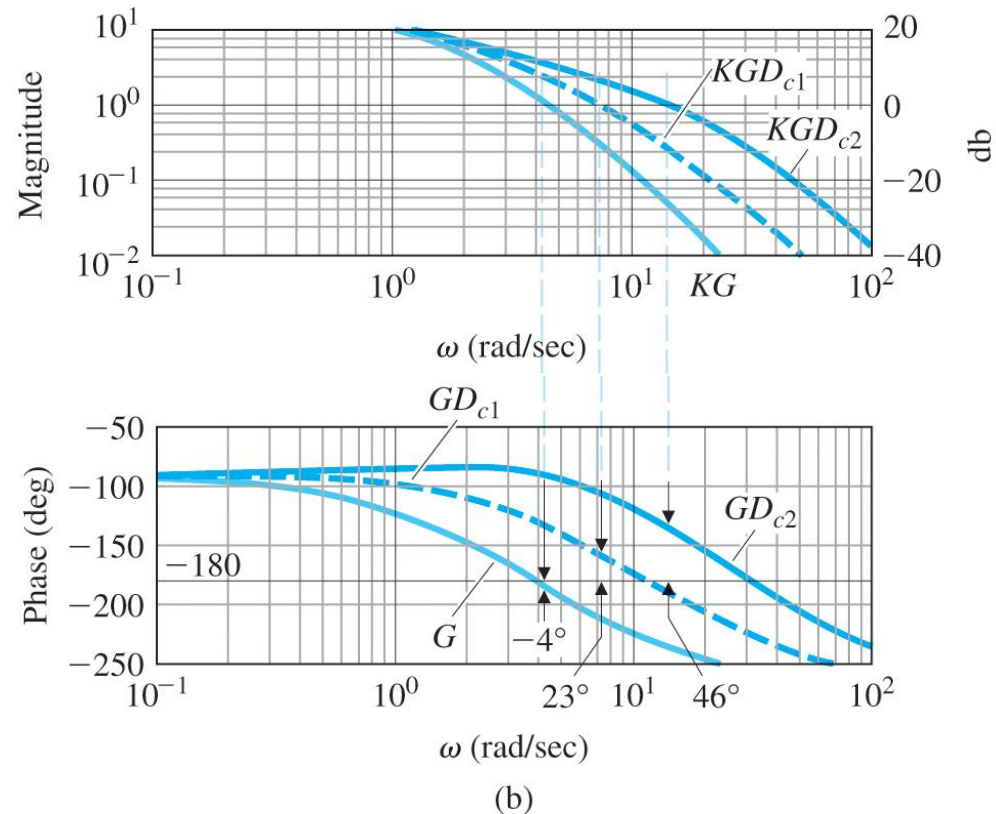
Examples

Example 6.17: Lead-Compensation Design for Type 1 Servomechanism System

7. A double-lead compensator:

$$D_2(s) = \frac{\left(\frac{s}{2} + 1\right) \left(\frac{s}{4} + 1\right)}{\left(\frac{s}{20} + 1\right) \left(\frac{s}{40} + 1\right)} = \frac{1}{(0.1)^2} \frac{(s + 2)(s + 4)}{(s + 20)(s + 40)}$$

■ PM = 46°



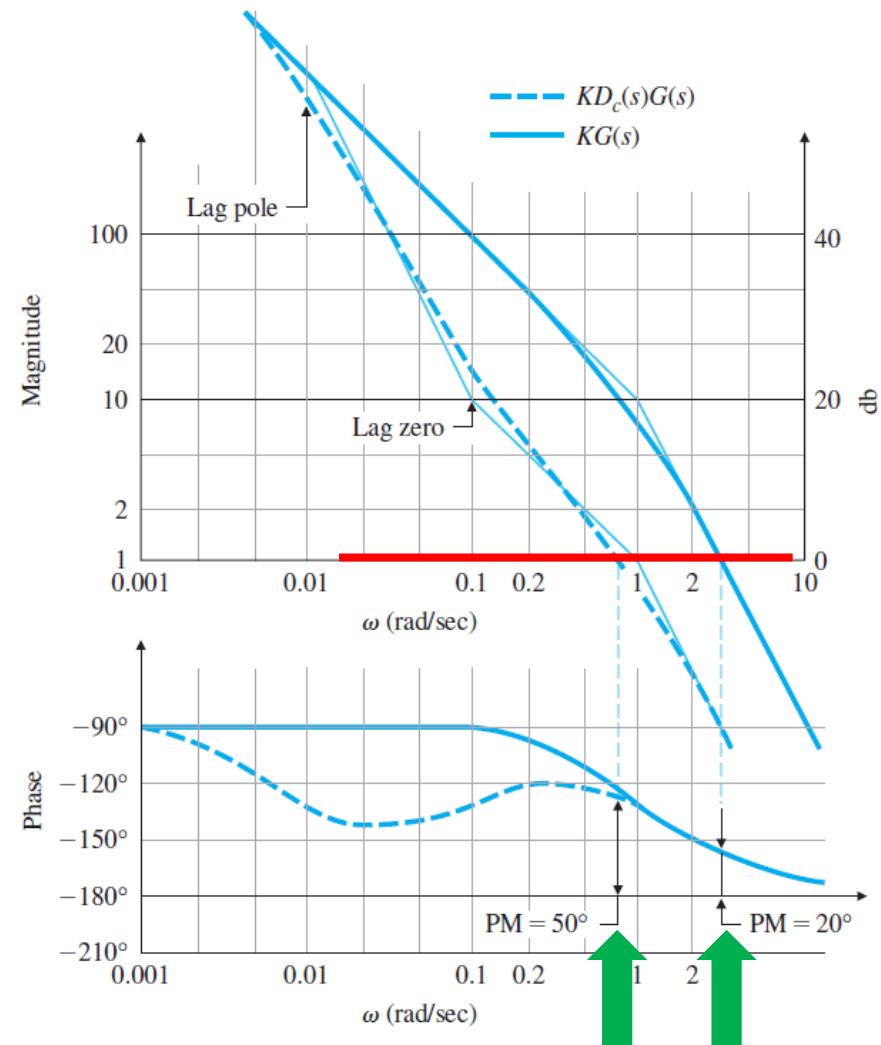
Examples

Example 6.19: Lag Compensation for the DC Motor

$$G(s) = \frac{1}{s(s+1)}$$

- $K = 10$
- $PM = 20^\circ$ at $\omega_c \approx 3$
- Select break points
 - ✓ ω_c is lowered
 - ✓ more favorable PM results
- Lag zero = 0.10
- Lag pole = 0.01
- ✓ $PM = 50^\circ$

- Error constant: $K_v = 10$
- $PM = 45^\circ$



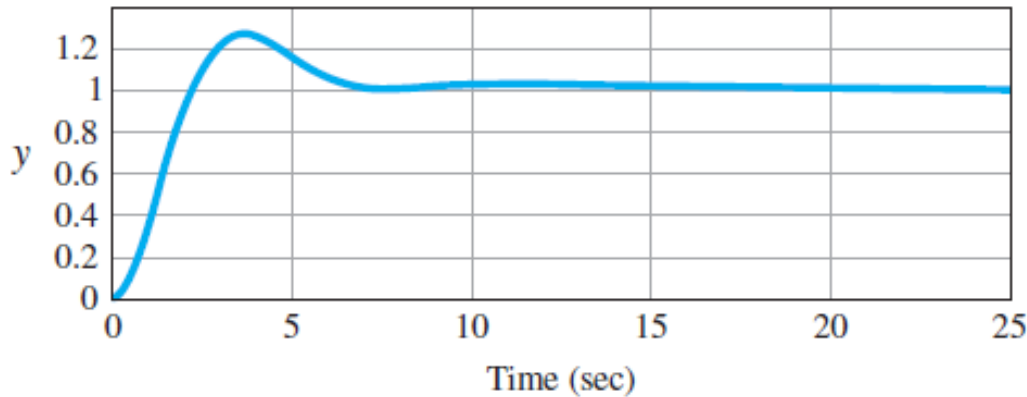
Examples

Example 6.19: Lag Compensation for the DC Motor

$$G(s) = \frac{1}{s(s+1)}$$

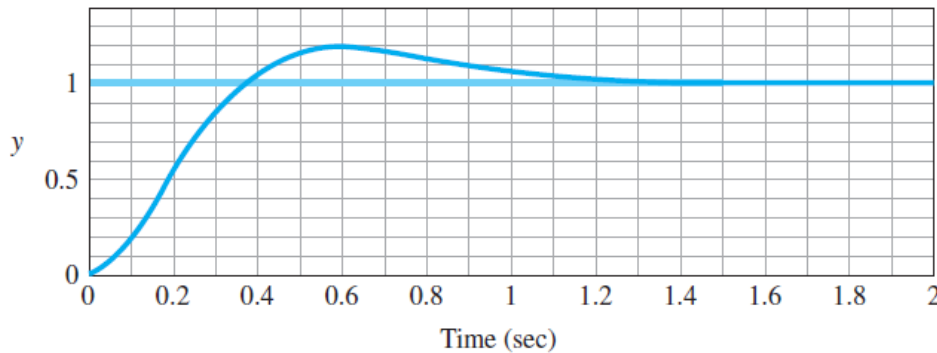
▪ Error constant: $K_v = 10$

▪ $PM = 45^\circ$



- No steady-state error
 - ✓ a Type 1 system
- Settling time ≈ 25 sec
- Rise time ≈ 2 sec

Example 6.15: Lead Compensation



Example

Example 6.20: PID Compensation

$$D_c(s) = \frac{K}{s} \left[(T_D s + 1) \left(s + \frac{1}{T_I} \right) \right] \quad \text{for Spacecraft Attitude Control}$$

$$G(s) = \frac{0.9}{s^2}$$

$$H(s) = \frac{2}{s + 2}$$

$\frac{1}{T_I} = 0.5$	$\frac{1}{T_D} = 10$
$\frac{1}{T_I} = 0.05$	$\frac{1}{T_D} = 1$
$\frac{1}{T_I} = 0.005$	$\frac{1}{T_D} = 0.1$

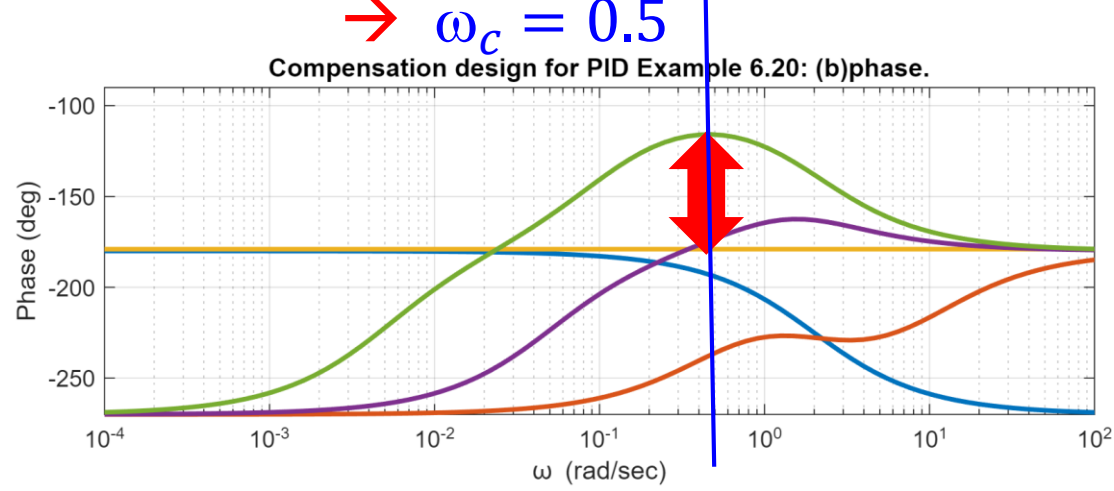
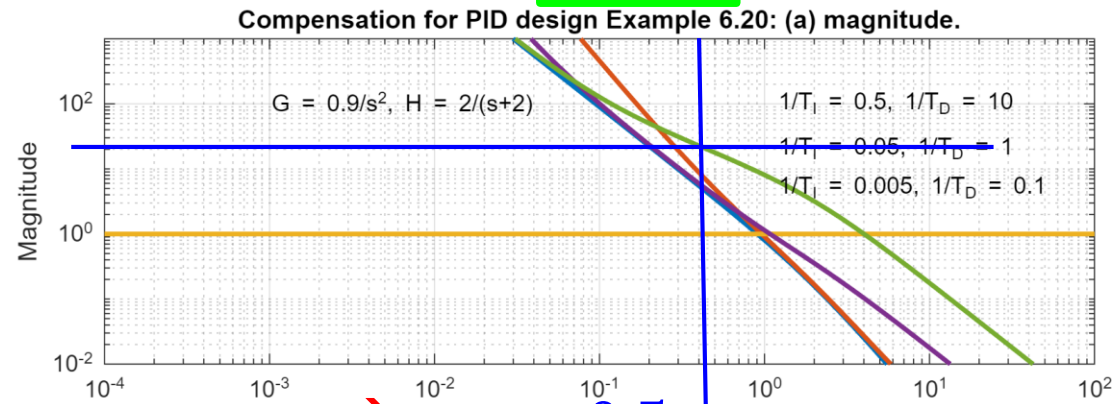
PM = 65°

→ Find K

$$| D_c(s) G(s) | = 20$$

$$1/K = 20$$

$$K = 0.05$$

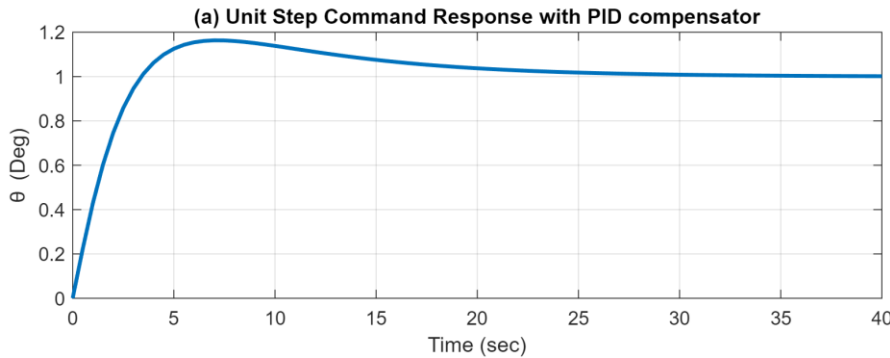


Example

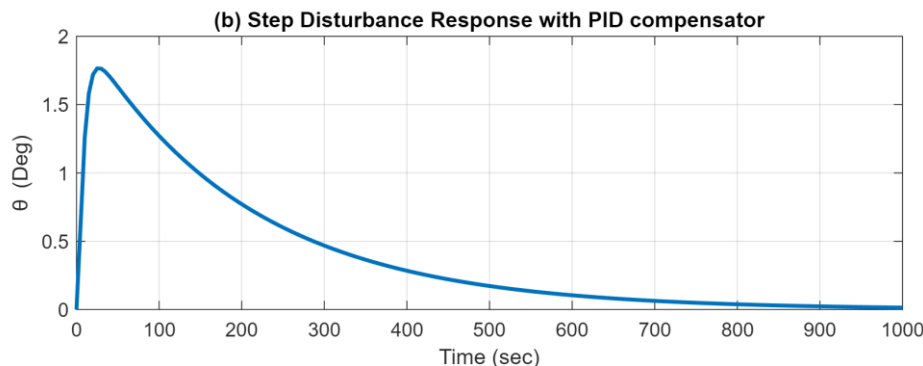
Example 6.20: PID Compensation

for Spacecraft Attitude Control

$$T(s) = \frac{\Theta}{\Theta_{com}} = \frac{D_c G}{1 + D_c G H}$$



$$\frac{\Theta}{T_d} = \frac{G}{1 + D_c G H}$$



Frequency Response of $T(s)$ and $S(s)$ are shown:

