Spring 2020

控制系統 Control Systems

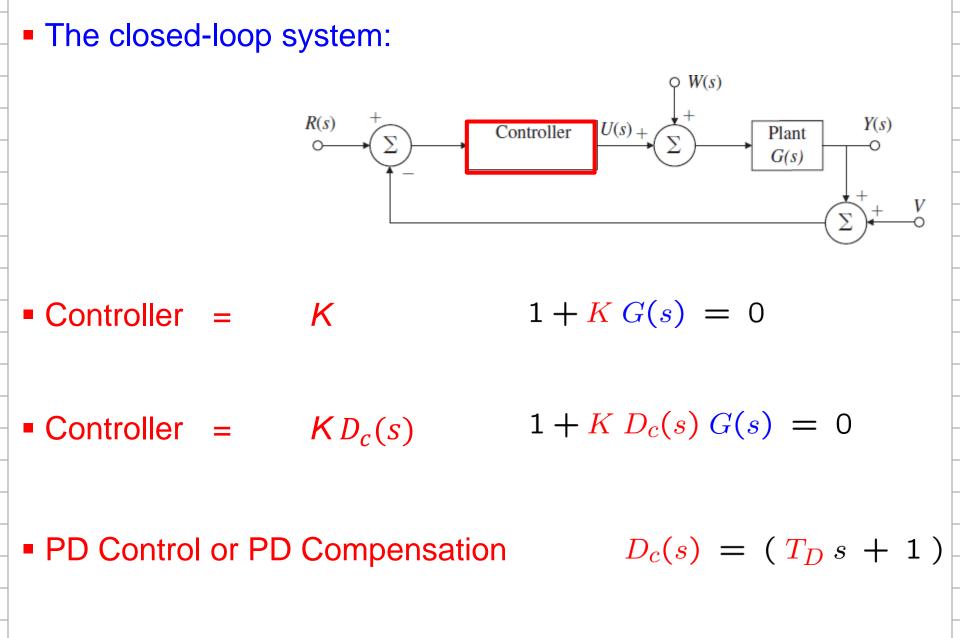
Unit 6I PD Compensation and Lead Compensation

Feng-Li Lian & Ming-Li Chiang NTU-EE Mar 2020 – Jul 2020

Dynamic elements (or compensation)

are typically added to feedback controllers

- To improve the system's stability and error characteristics
- Because the process itself cannot be made to have acceptable characteristics with proportional feedback alone.
- Unit 43:
 - 3 basic types of feedback: P, I, D.
- Unit 54:
 - 3 kinds of dynamic compensation: Lead-PD, Lag-PI, Notch.



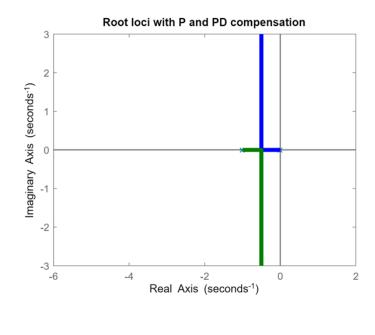
PD Compensation

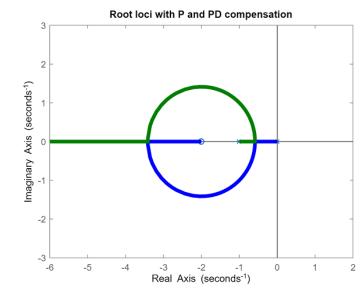
CS6I-PDLead - 4 Feng-Li Lian © 2020



 $D_c(s) = K$ $G(s) = \frac{1}{s(s+1)}$ $\Rightarrow 1 + K \frac{1}{s(s+1)} = 0$ $\Rightarrow s(s+1) + K = 0$

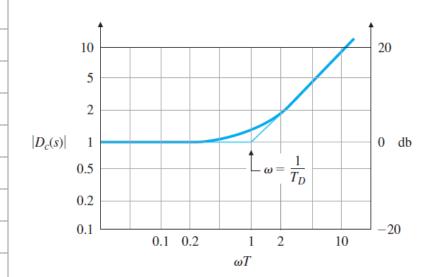
 $D_c(s) = K (s+2)$ $G(s) = \frac{1}{s(s+1)}$ $\Rightarrow 1 + K (s+2) \frac{1}{s(s+1)} = 0$ $\Rightarrow s(s+1) + K(s+2) = 0$

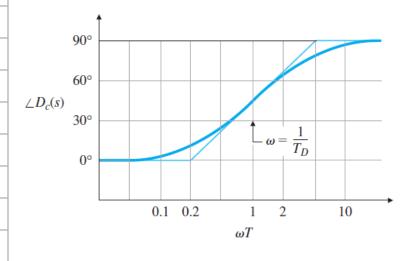




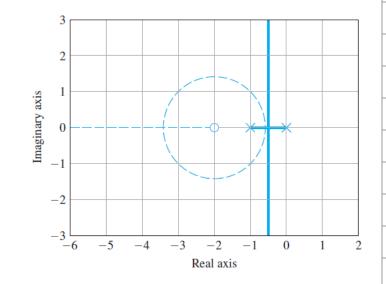
PD Compensation

PD Control or PD Compensation





$$D_c(s) = (T_D s + 1)$$



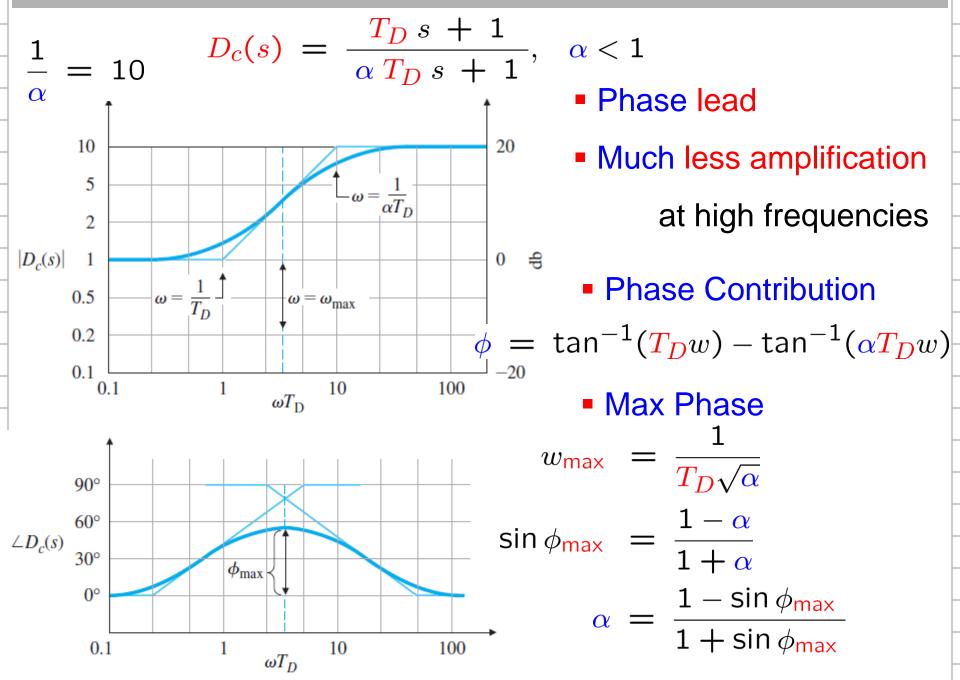
- Increase in phase
- +1 slope above $\omega = 1/T_D$
- Gain increased with frequency
 - Undesirable
 - Amplify high-frequency noise

- To alleviate the high-frequency amplification of the PD compensation,
- A first-order pole is added in the denominator at frequencies substantially higher than the break point of PD compensator.
- Thus, the phase increase (or lead) still occurs,
- But, the amplification at high frequencies is limited.
- The resulting lead compensation has a transfer function of:

$$D_c(s) = \frac{T_D s + 1}{\alpha T_D s + 1}, \quad \alpha < 1$$

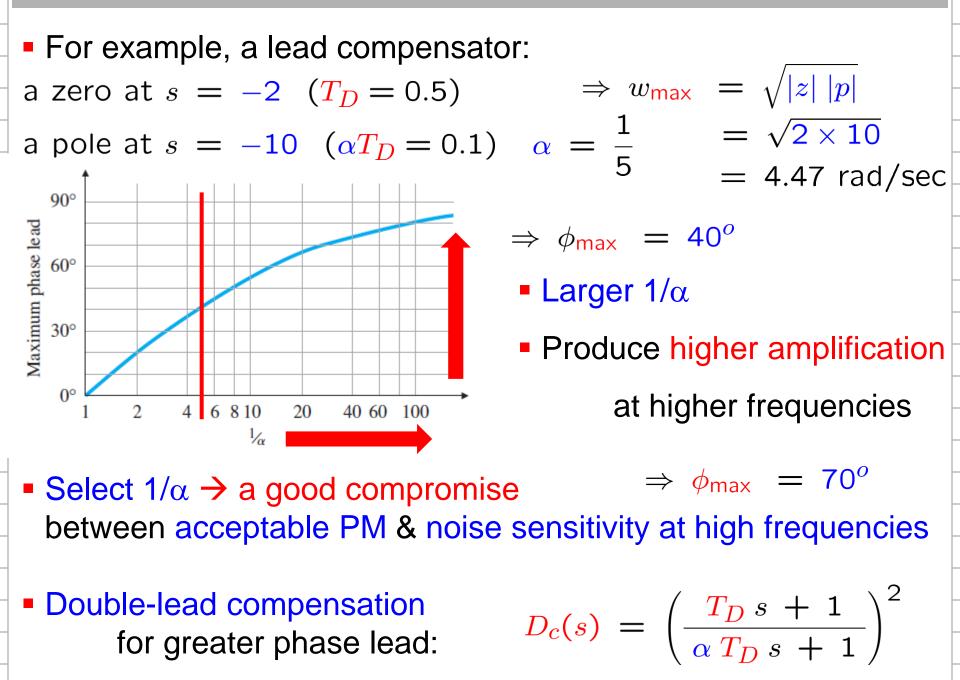
Lead Compensation

CS6I-PDLead - 7 Feng-Li Lian © 2020



The maximum phase occurs at a frequency that lie midway between the two break-point frequencies (sometimes called corner frequencies) on a logarithmic scale $\log w_{\max} = \log \frac{\sqrt{T_D}}{\sqrt{T_D}}$ $= \log \frac{1}{\sqrt{T_D}} + \log \frac{1}{\sqrt{\alpha T_D}}$ $= \frac{1}{2} \left| \log \left(\frac{1}{T_D} \right) + \log \left(\frac{1}{\alpha T_D} \right) \right|$ $D_c(s) = \frac{s+z}{s+n}$ $\Rightarrow w_{\max} = \sqrt{|z| |p|}$ $z = \frac{-1}{T_D}$ $\log w_{\max} = \frac{1}{2} \left(\log |z| + \log |p| \right)$ $=\frac{-1}{2}$

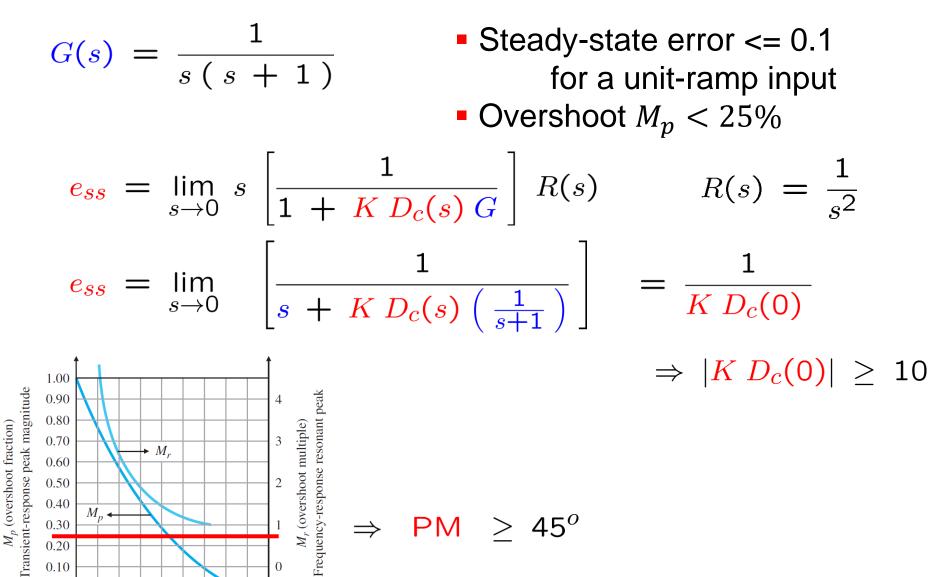
Lead Compensation



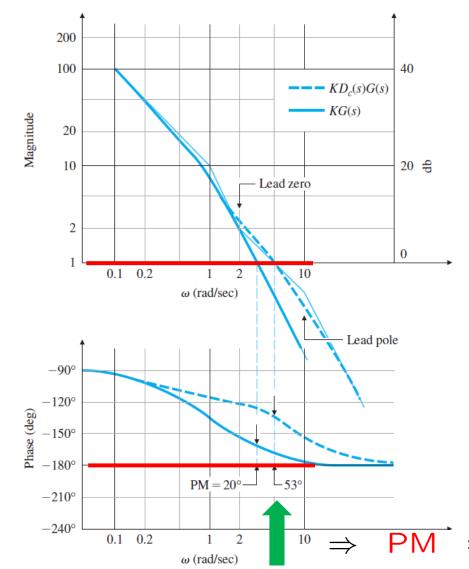
0.10 0

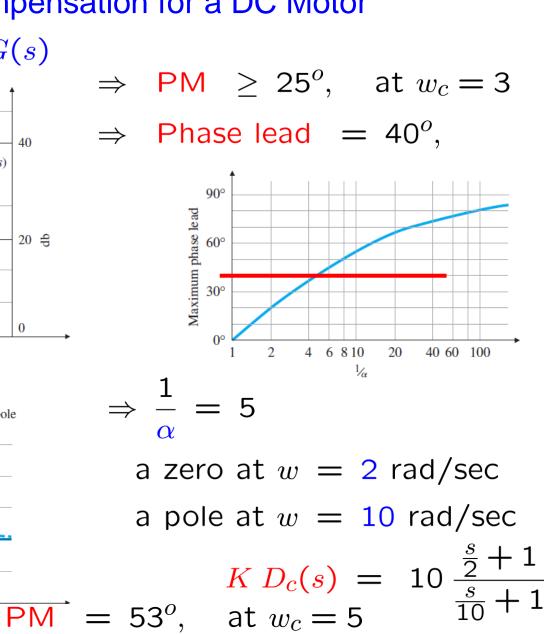
> 0° 10° 20° 30° 40° 50° 60° 70° 80° 90° Phase margin

Example 6.15: Lead Compensation for a DC Motor

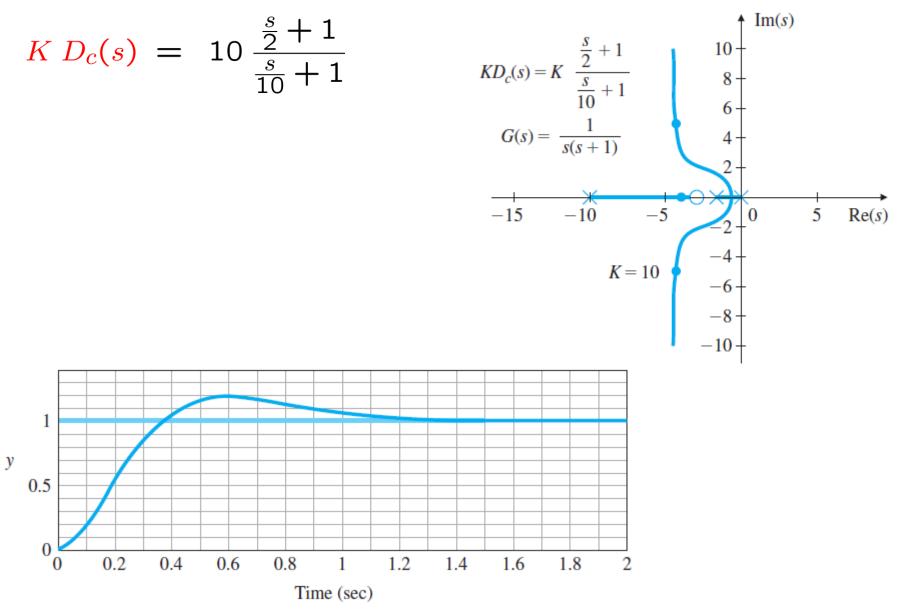


• Example 6.15: Lead Compensation for a DC Motor K G(s) $K D_c(s) G(s)$





Example 6.15: Lead Compensation for a DC Motor



1. Crossover frequency ω_c ,

which determines

bandwidth ω_{BW} ,

rise time t_r , settling time t_s

2. Phase Margin (PM),

which determines

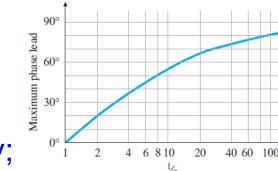
damping coefficient ζ , overshoot M_p

- 3. The low-frequency gain,
 - which determines

the steady-state error characteristics

- 1. Determine gain K to satisfy error or bandwidth requirements:
 - a) To meet error requirements, pick K to satisfy error constants (K_P, K_v, K_a), so that e_{ss} is met.
 - b) To meet bandwidth requirements, pick K so that the OL crossover frequency is a factor of two below the desired CL bandwidth.

- Evaluate the PM of the uncompensated system using the value of K obtained from Step 1
- 3. Allow for extra margin (about 10°) and determine the needed phase lead ϕ_{max}



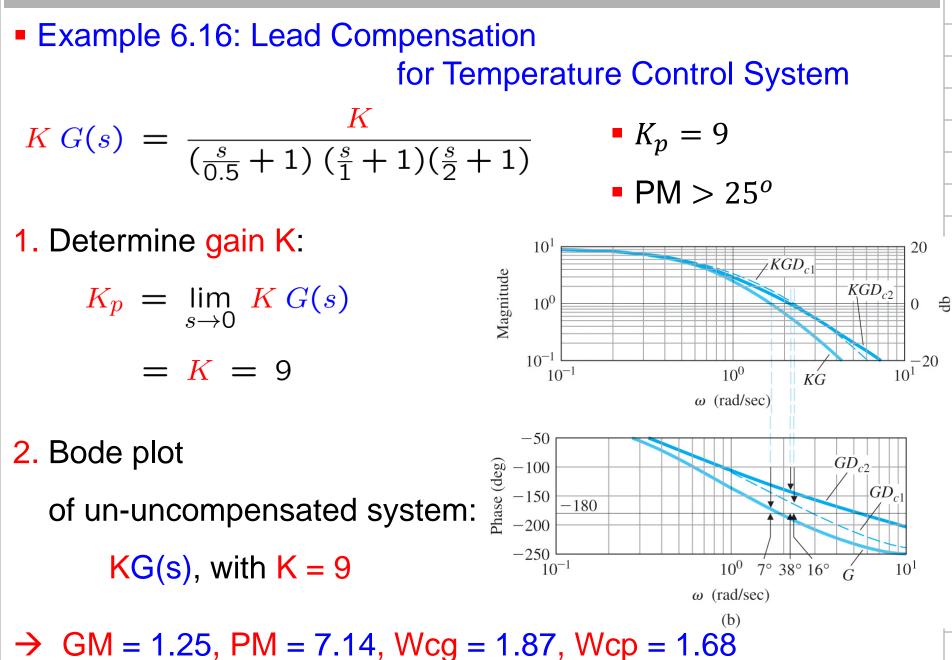
- 5. Pick ω_{max} to be the crossover frequency;
 - a zero at $1/T_D = w_{max}\sqrt{\alpha}$
 - a pole at $1/(\alpha T_D) = w_{max}/\sqrt{\alpha}$
- 6. Draw the compensated frequency response and check PM

7. Iterate on the design.

Determine α

4.

- Adjust compensator parameters (poles, zeros, gain) until all specification are met.
- Add an additional lead compensator if necessary.



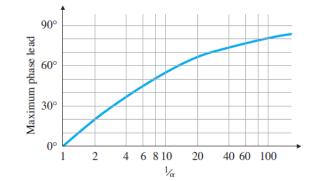
Example 6.16: Lead Compensation for Temperature Control System

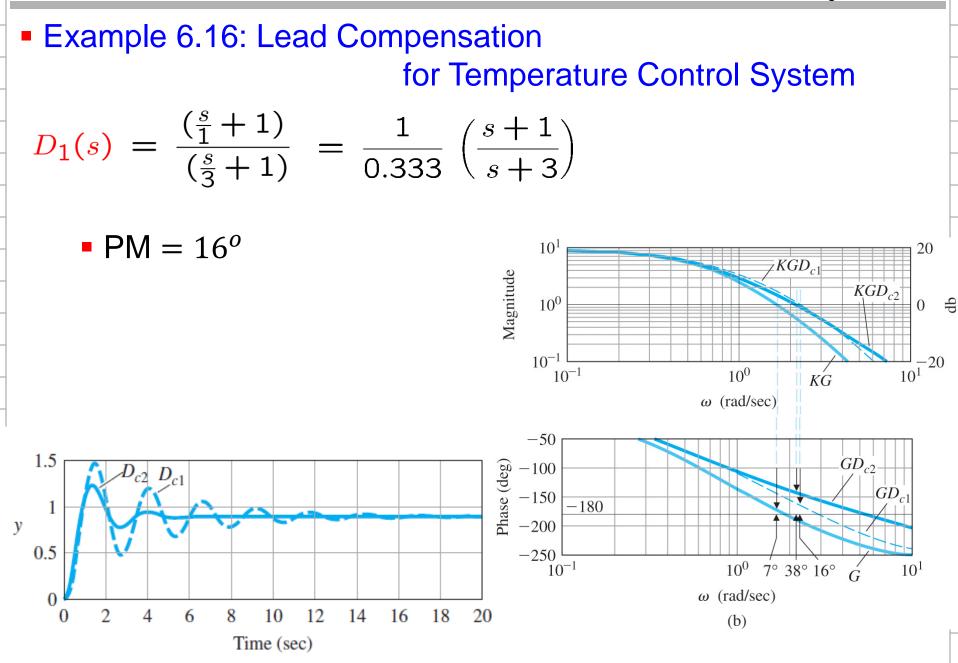
3. Allow for 10° of extra margin $\rightarrow 25^{\circ} + 10^{\circ} - 7^{\circ} = 28^{\circ}$

- 4. Pick $\alpha \rightarrow 1/\alpha = 3$
- 5. Zero & Pole

a zero at 1 $T_D = 1$ a pole at 3 $\alpha T_D = 1/3$

 $D_1(s) = \frac{(\frac{s}{1}+1)}{(\frac{s}{3}+1)} = \frac{1}{0.333} \left(\frac{s+1}{s+3}\right)$





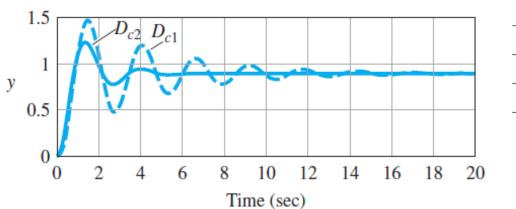
- Example 6.16: Lead Compensation
 - for Temperature Control System

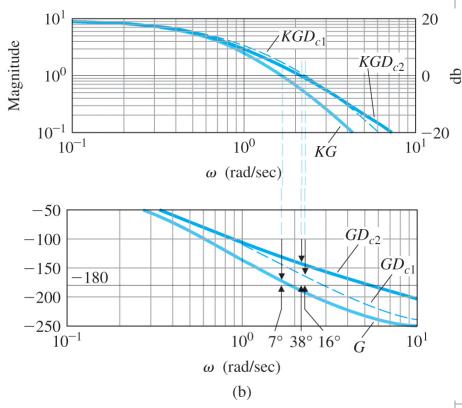
7. Move zero:

a zero at s = -1.5 $\alpha = 1/10$

$$D_2(s) = \frac{\left(\frac{s}{1.5} + 1\right)}{\left(\frac{s}{15} + 1\right)}$$
$$= \frac{1}{0.1} \left(\frac{s+1.5}{s+15}\right)$$

• PM = 38^o





Example 6.17: Lead-Compensation Design for Type 1 Servomechanism System

$$\frac{KG(s)}{s\left(\frac{s}{2.5}+1\right)\left(\frac{s}{6}+1\right)}$$

•
$$K_v = 10$$

1 0

T 7

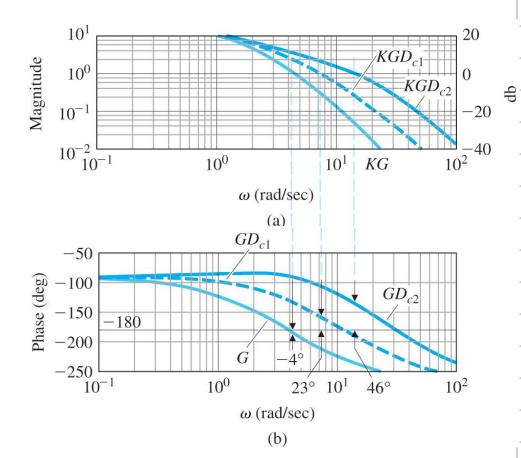
•
$$PM = 45^{\circ}$$

1. Determine gain K:

$$K_v = \lim_{s \to 0} s K G(s)$$
$$= K \times 10 = 10$$
$$\Rightarrow K = 1$$

2. Bode plot of KG(s), K = 1

→ PM ~= - 4, Wcp ~= 4



Example 6.17: Lead-Compensation Design for Type 1 Servomechanism System 90° 3. Allow for 5° of extra margin phase lead 09 → $45^{\circ} + 5^{\circ} - (-4^{\circ}) = 54^{\circ}$ Maximum 30° **0**° 4. Pick $\alpha \rightarrow 1/\alpha = 10$ 2 4 6 8 10 20 40 60 100 1/0 10^{1} 20 KGD_{c1} Magnitude 10^{0} 5. Zero & Pole 0 KGD_{c2} qp 10^{-1} -20 a zero at 2 -40 10^{-2} 10^{-1} 10^{0} 10^{2} 10^{1} KG a pole at 20 ω (rad/sec) $D_1(s) = \frac{(\frac{s}{2}+1)}{(\frac{s}{20}+1)}$ GD_{c1} -50(deg) −100 −150 −200 GD_{c2} -180 $=\frac{1}{0.1}\left(\frac{s+2}{s+20}\right)$ $-250 \ \ 10^{-1}$ G 10 10^{0} $23^{\circ} 10^{1}$ 10^{2} 46° ω (rad/sec) → PM ~= 23, Wcp ~= 7 (b)

CS6I-PDLead - 22 Feng-Li Lian © 2020

 Example 6.17: Lead-Compensation Design for Type 1 Servomechanism System
7. A double-lead compensator:

 $D_2(s) = \frac{\left(\frac{s}{2}+1\right)\left(\frac{s}{4}+1\right)}{\left(\frac{s}{20}+1\right)\left(\frac{s}{40}+1\right)} = \frac{1}{(0.1)^2} \frac{(s+2)(s+4)}{(s+20)(s+40)}$

